



Computational Combustion and Energy Group



NUS

National University
of Singapore

Department of Mechanical Engineering
Faculty of Engineering

Clean Combustion Technology

Haze in Singapore



Agricultural residual burning



Domestic combustion device



Aviation



Ground transportation

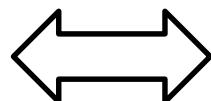


Power generation



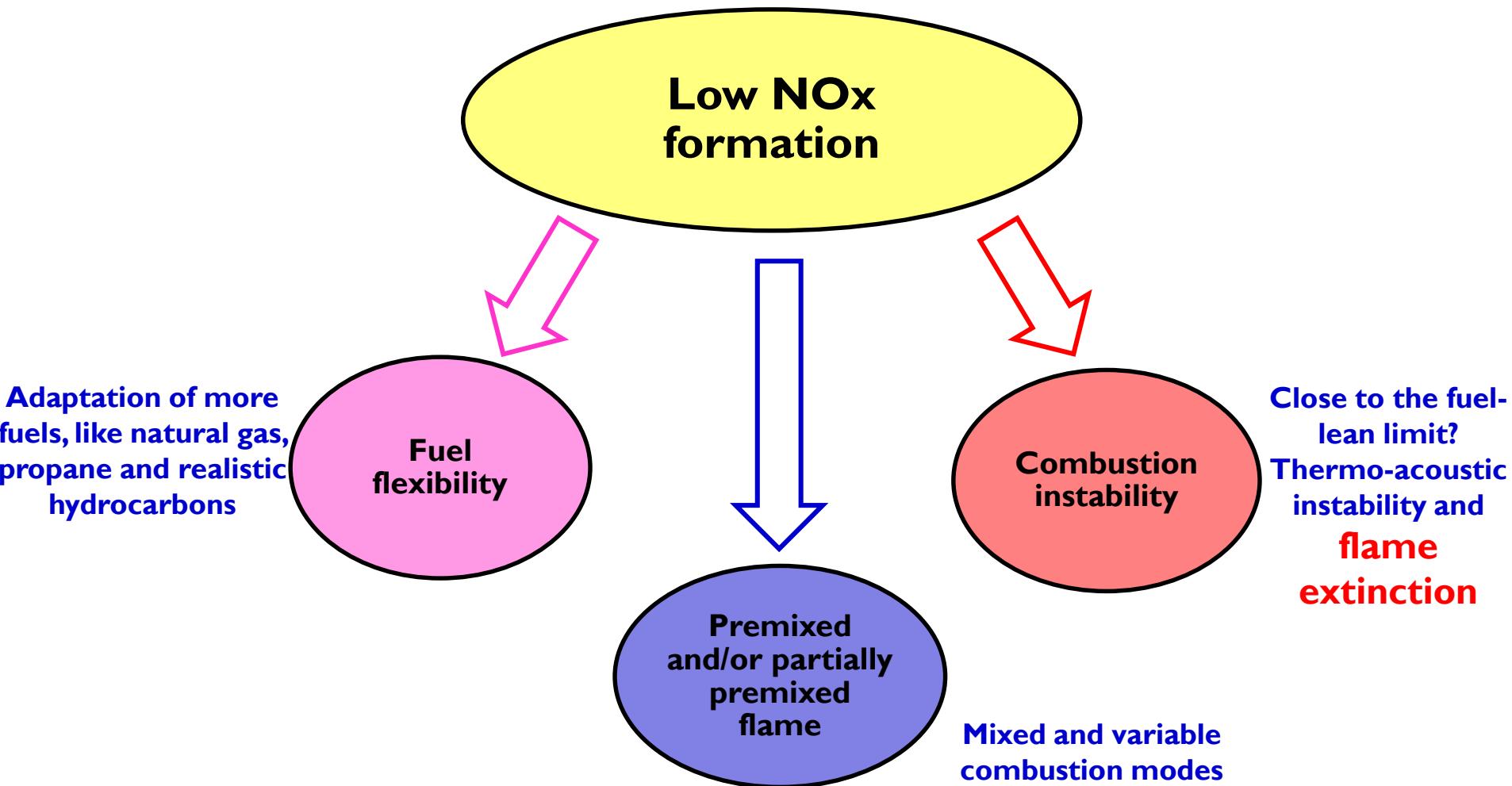
- Decrease pollutant emission (mainly nitrogen oxides NOx)
- Increase combustion efficiency

Innovative
combustion
Technologies

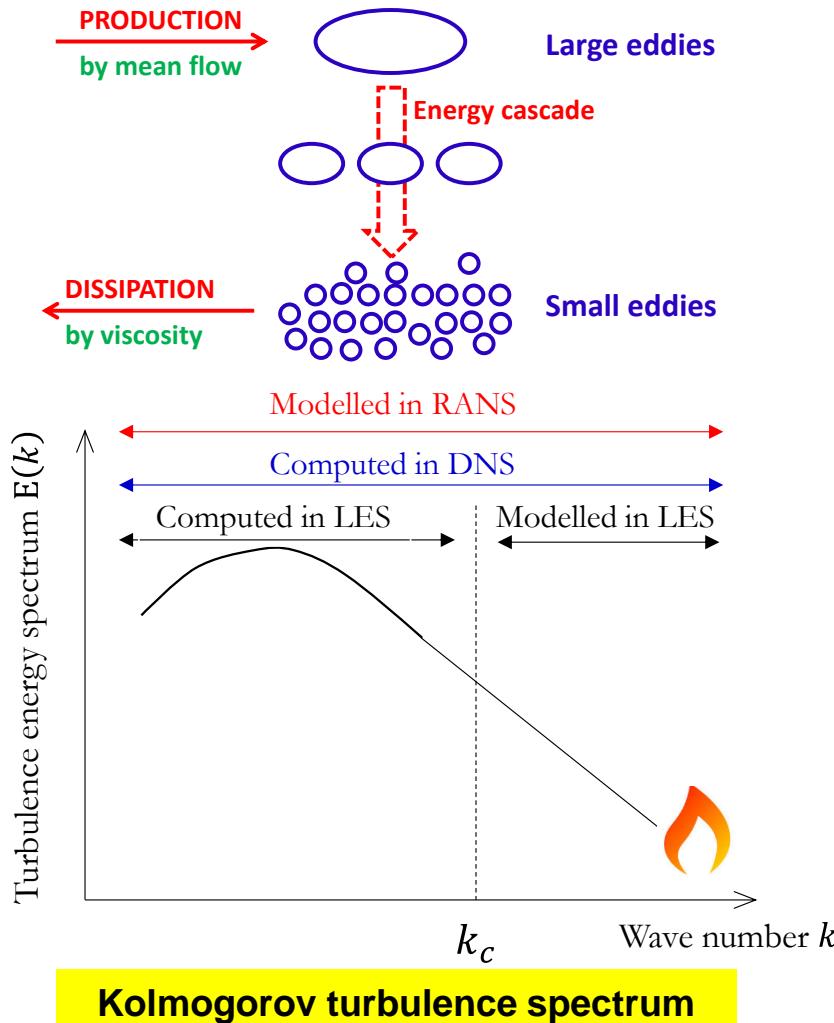


Alternative Fuels

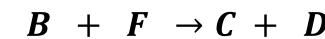
❖ low NOx (nitrogen oxide) emission \leftrightarrow Fuel lean combustion



Multi-scale and nonlinearity in turbulence & combustion



A Hypothetical Chemical Mechanism



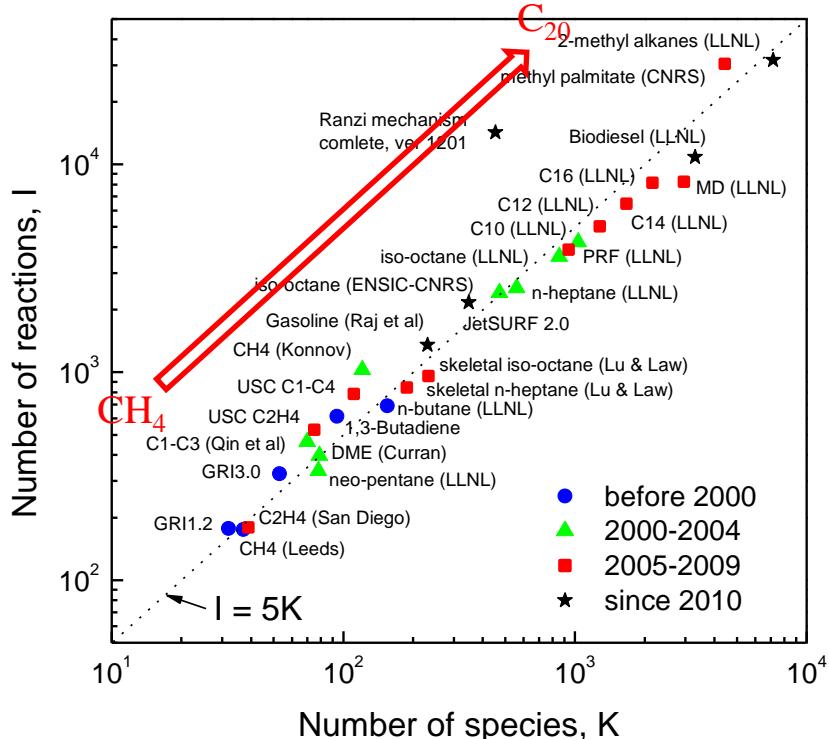
Nonlinearity in Arrhenius-type reaction rate:

$$\dot{\omega}_F = \dot{\omega}_F(Y_1 \dots Y_\alpha, T) = -AT^\beta Y_F Y_O \exp\left(-\frac{T_A}{T}\right)$$

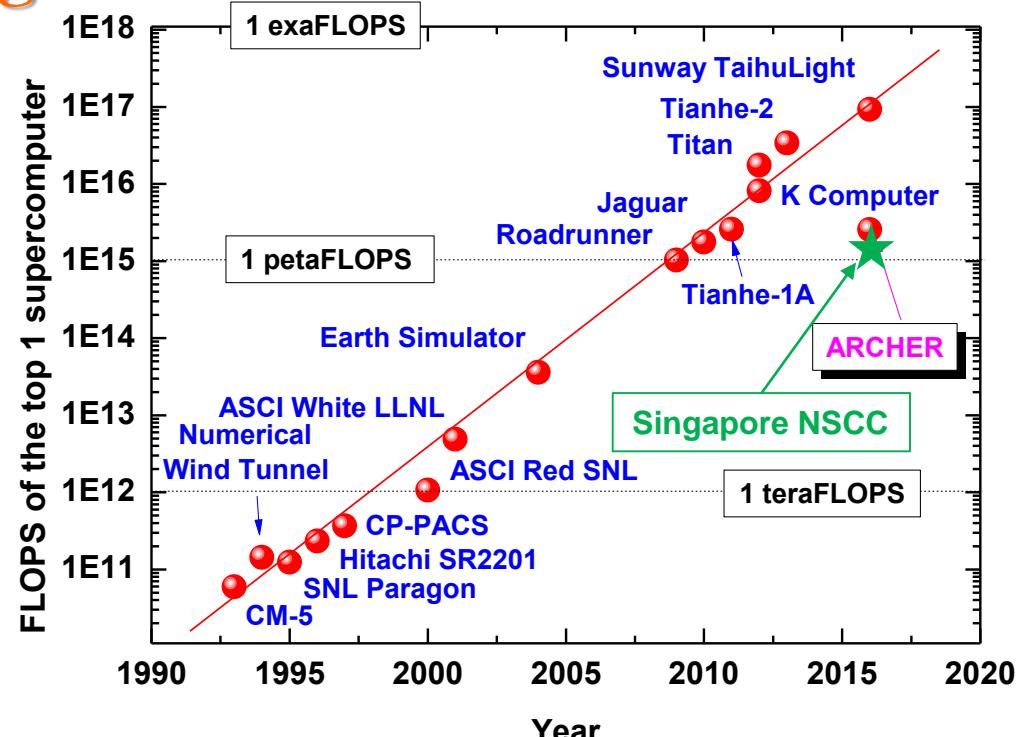
Is Direct Numerical Simulation of Turbulent Combustion Possible?



Opportunity and challenge in numerical combustion



(Lu & Law, Prog. Energy Combust. Sci. 2009)

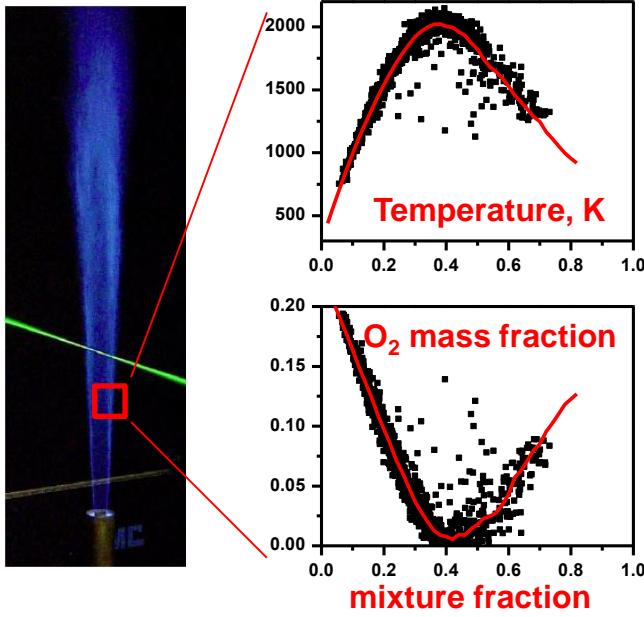


(Data source: Top 500 Supercomputer Website)

Is Direct Numerical Simulation of **Realistic** Turbulent Combustion with **Detailed** Chemistry Possible?



Conditional Moment Closure (CMC) Model



Sandia flame D

Barlow and Frank, Proc. Combust. Inst. 1998

$$\widetilde{\omega}_\alpha \neq \omega_\alpha(\tilde{Y}_1 \dots \tilde{Y}_n, \tilde{T})$$

$$\widetilde{\omega}_\alpha|\eta \approx \widetilde{\omega}_\alpha(\widetilde{Y_1}|\eta, \dots, \widetilde{Y_n}|\eta, \widetilde{T}|\eta)$$

- Five-dimensional Eqs. of species

mass fraction ($\widetilde{Y_\alpha}|\eta$) and energy ($\widetilde{h}|\eta$):

$$\begin{aligned} & \underbrace{\int_{\Omega^{CMC}} \frac{\partial Q_\alpha}{\partial t} d\Omega}_{T_0} + \underbrace{\int_{\Omega^{CMC}} \nabla \cdot (\widetilde{U}|\eta Q_\alpha) d\Omega}_{T_1} = \\ & \underbrace{\int_{\Omega^{CMC}} Q_\alpha \nabla \cdot \widetilde{U}|\eta d\Omega}_{T_2} + \underbrace{\int_{\Omega^{CMC}} \widetilde{N}|\eta \frac{\partial^2 Q_\alpha}{\partial^2 \eta} d\Omega}_{T_3} + \underbrace{\int_{\Omega^{CMC}} \widetilde{\omega_\alpha}|\eta d\Omega}_{T_4} + \underbrace{\int_{\Omega^{CMC}} \nabla \cdot (D_t \nabla Q_\alpha) d\Omega}_{T_5} \end{aligned}$$

T1: Convection; T2: Dilatation; T3: Micro-mixing;
T4: Chemistry; T5: Turbulent Scalar Flux

Modeled Terms

- ✓ $\widetilde{U}|\eta = \widetilde{U}$
- ✓ $\widetilde{N}|\eta$: amplitude mapping closure (AMC)
- ✓ $\widetilde{P}(\eta)$: presumed beta-function
- ✓ $\widetilde{\omega_\alpha}|\eta$: **first order CMC closure, detailed chemistry, ARM2 mechanism**
- ✓ a gradient model used for sub-grid scale scalar flux

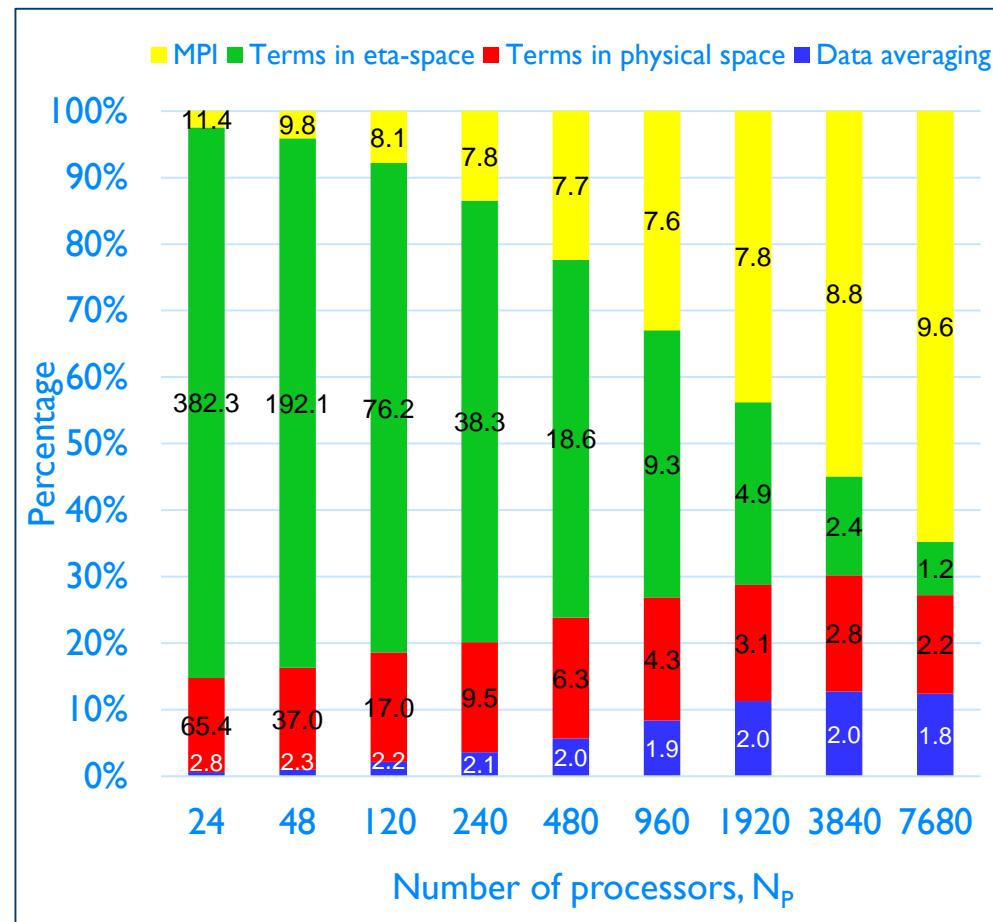
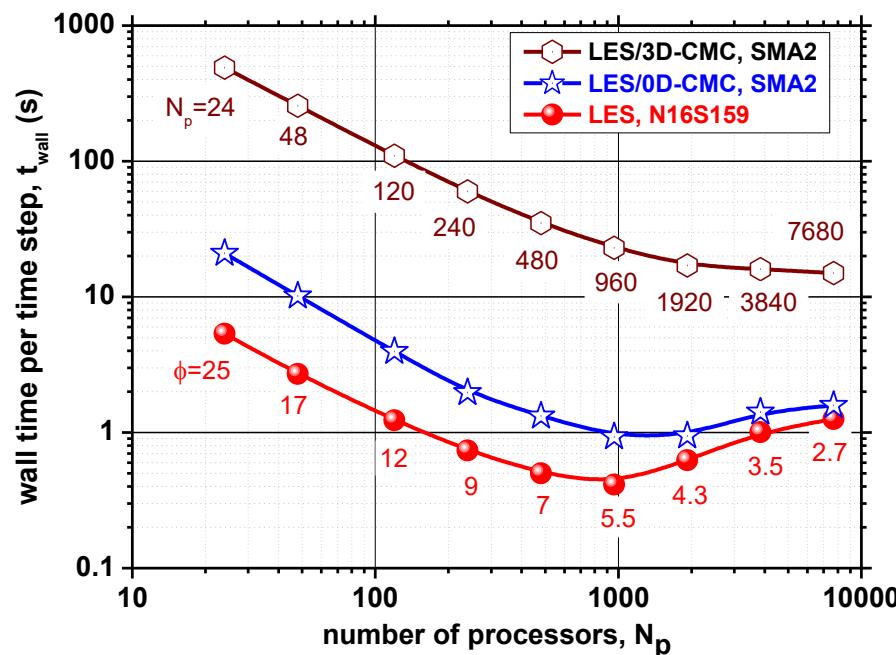
First-order CMC model

Zhang et al., Proc. Combust. Inst. 2015.

Zhang and Mastorakos, Flow Turb Combust 2016.

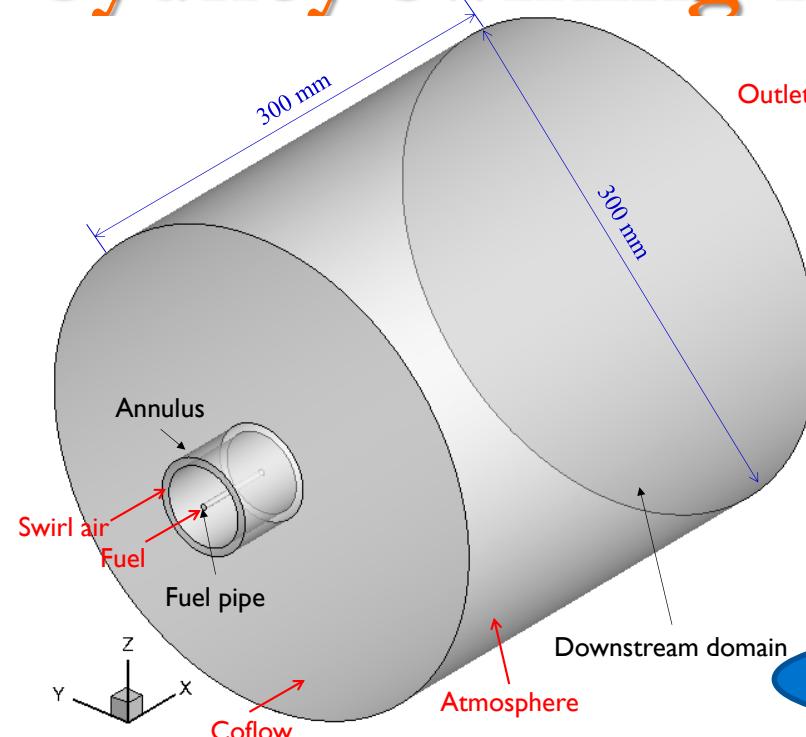
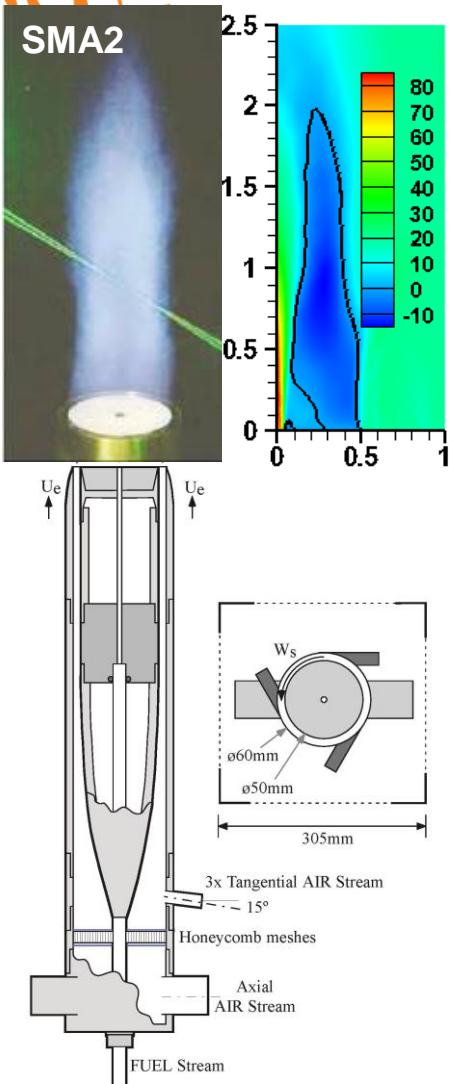
Garmory and Mastorakos, Proc. Combust Inst 2015. Klimenko and Bilger, Prog. Combust. Energy Sci. 1999.

Parallelization and computational cost



- Perfect loading balancing for parallelized CMC solver: round-robin parallel algorithm $p = \text{mod}(n-1, N_p) + 1$,
- Good scaling for $O(10^3)$ processors on ARCHER Cluster of UK National Supercomputer

Sydney swirling flames



- One of the target flames in **TNF workshop**, measured by researchers from University of Sydney and Sandia
- Swirl number $S_N = W_S/U_S = 1.59$
- Oxidizer: swirling air; fuel: non-swirling **CH₄/air** (1:2)
- Approximately **8,400,000** tetrahedral LES cells, About **120,000** polyhedral CMC cells

**Strong turbulence,
increased local extinction**

Cases	U_s (m/s)	W_s (m/s)	U_j (m/s)	$U_j / U_{j,SL}$	\dot{W} (kW)	Re_j
SMA2	16.3	25.9	66.3	31%	11.5	15,400
SMA3			132.6	62%	23.0	30,800
SMA4			225.0	104%	39.0	52,300

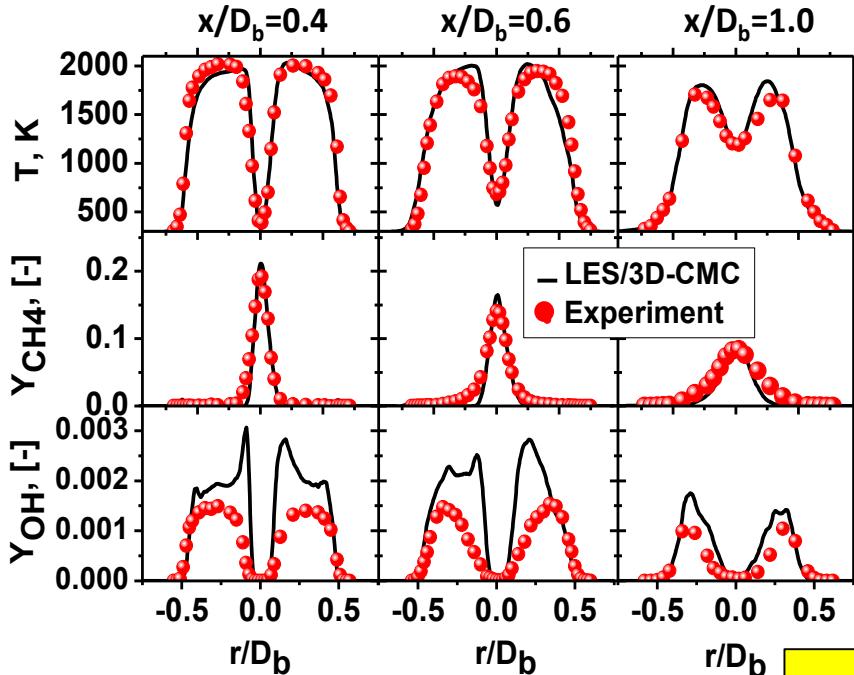
Sydney swirl burner

Masri et al, Combust. Theor. Model. 2007.

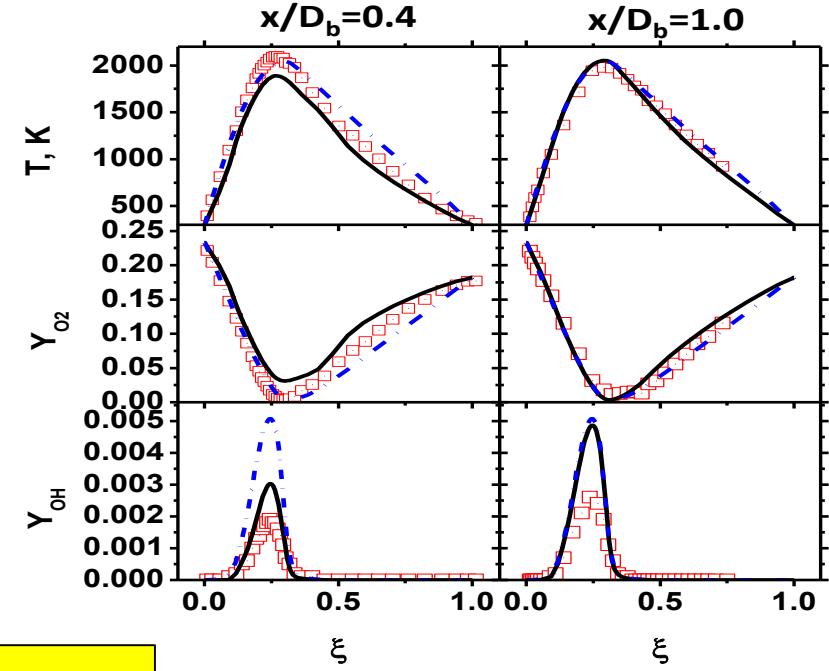
Masri et al, Combust. Flame 2004.

Al-Abdei and Masri, Combust. Theor. Model. 2003.

Reactive scalars



Symbols : Experimental data (Masri et al. CNF 2004)
 Solid lines : LES
 Dashed lines: 0D-CMC calculation with low scalar dissipation



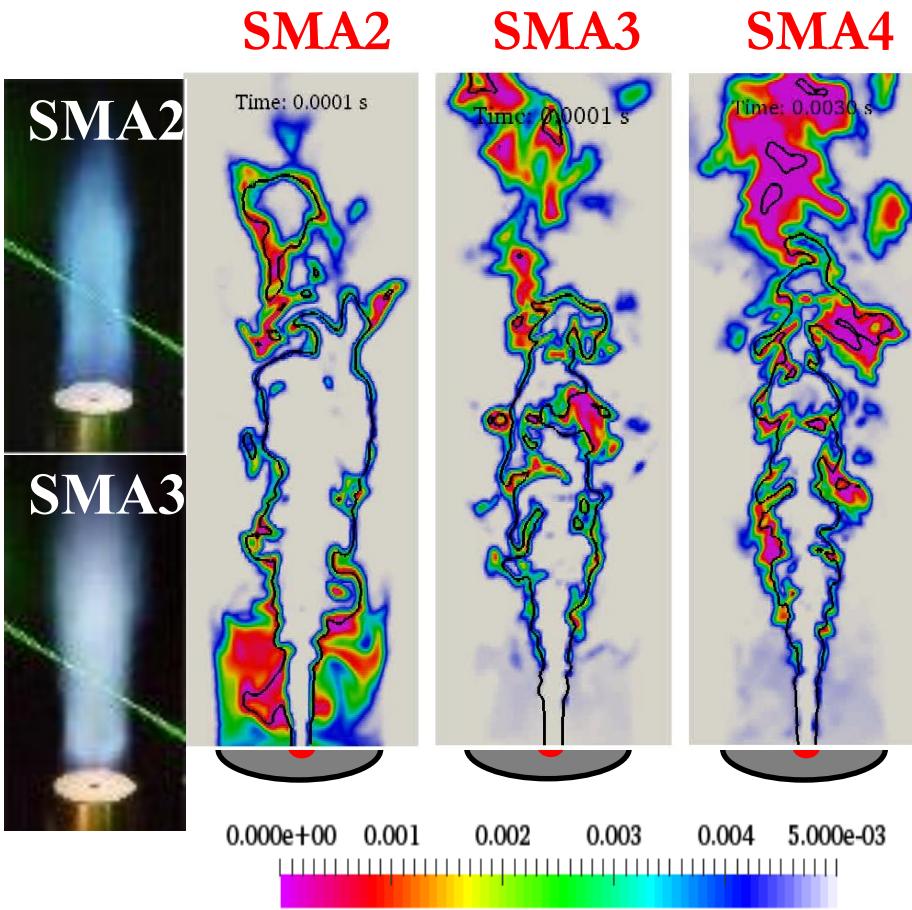
Unconditional scalars $\langle \tilde{f} \rangle$

$$\tilde{f} = \int_0^1 \tilde{f} | \eta P(\eta) d\eta$$

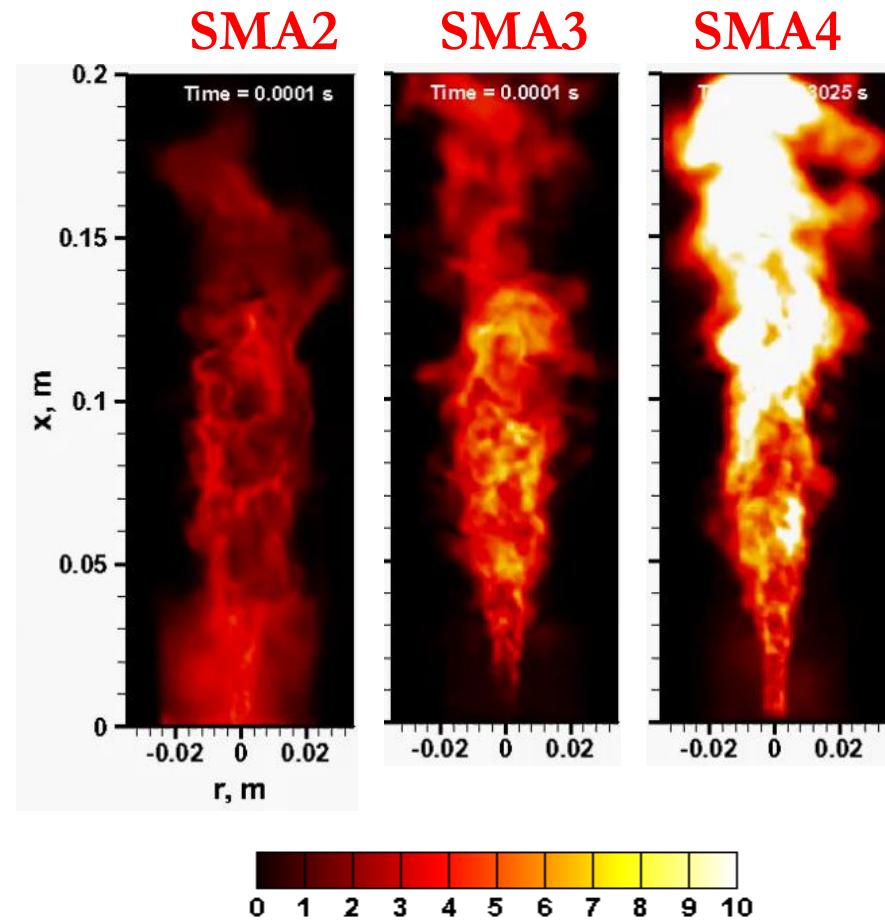
Conditional scalars $\langle \tilde{f} | \eta \rangle$

1. Unconditional means of temperature and methane mass fraction agree well with experimental data.
2. The conditional temperature and O₂ mass fraction are computed well, which are close to the equilibrium flame structures, but the OH is over-predicted in LES.
3. Overall, the current LES/CMC solver demonstrates good accuracy in predicting the reactive scalars of this simulated flame.

Time evolutions of OH and heat release rate



Resolved **OH** mass fraction (OH-PLIF-like)
(black iso-lines: stoi. mixture fraction 0.25)



Line-of-sight **integrated heat release rate** (MJ/m²s)
(OH* chemiluminescence-like)

Cambridge swirling flames

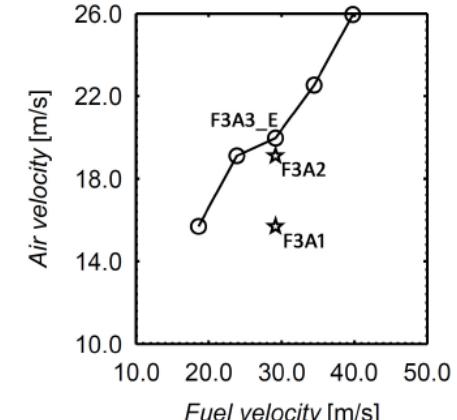
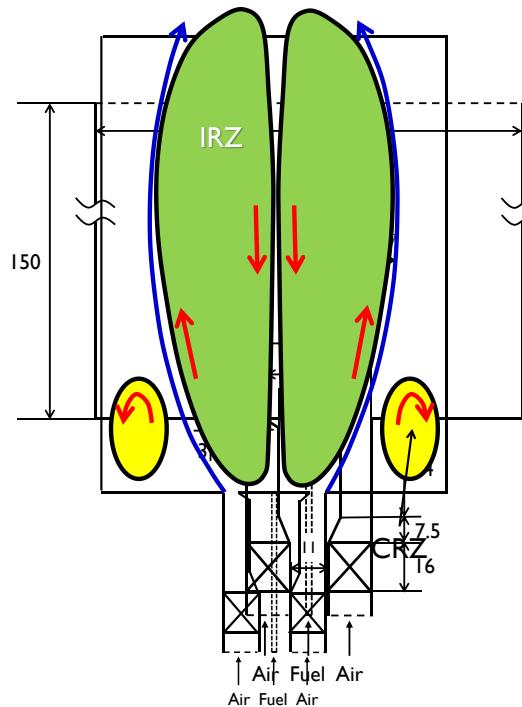


(a) Chamber



(b) Bluff body

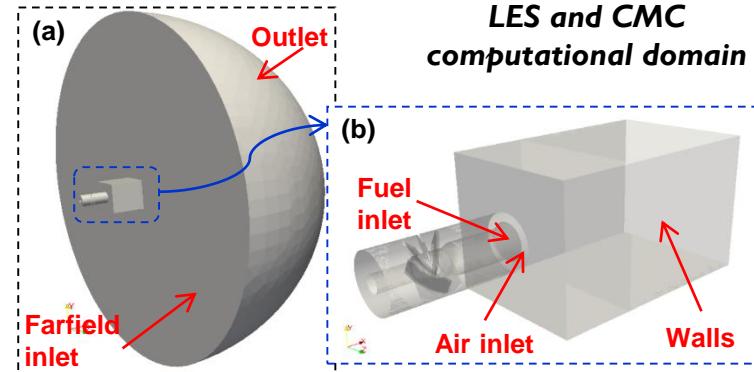
Schematic of experimental setup



Blow-off curve

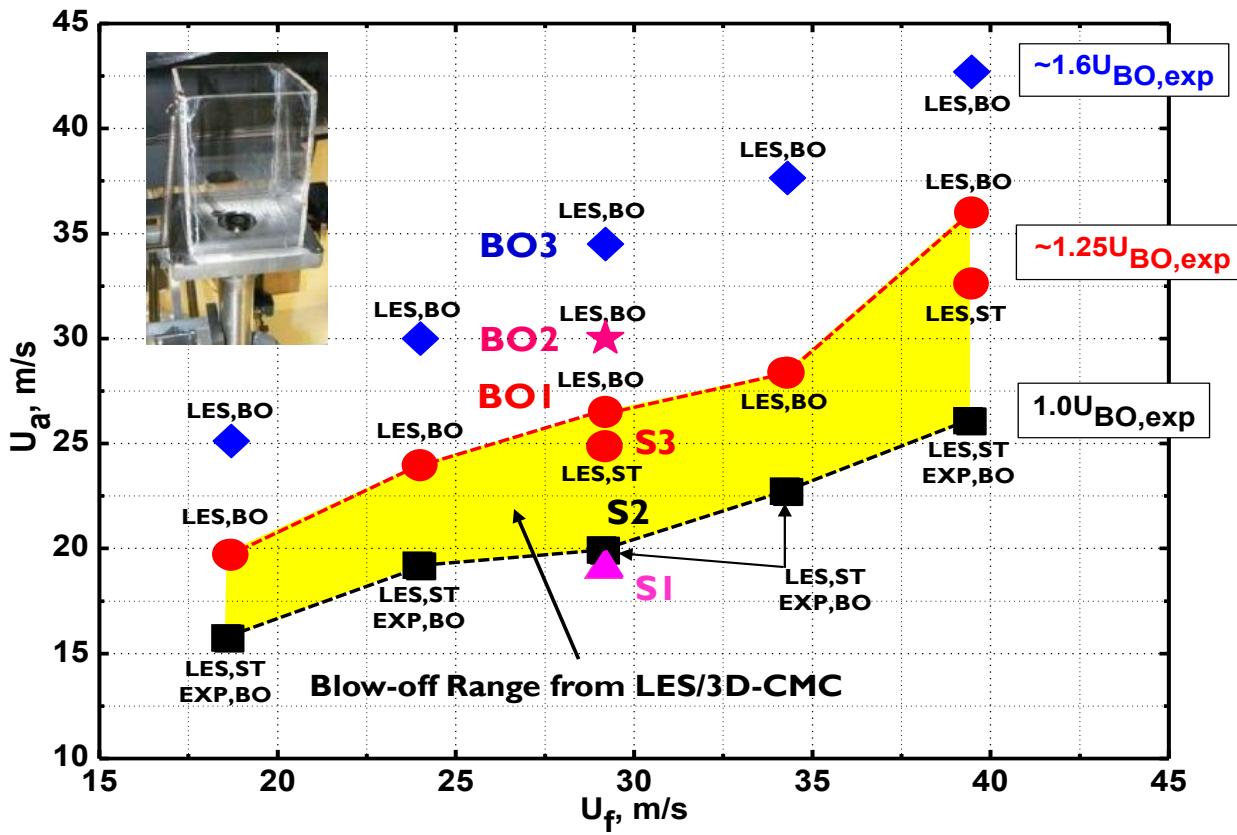
- Fuel/oxizider: pure methane (non-swirling) and air (swirling)
- The swirl number S_N is calculated following Beer and Chigier's method (Applied Science 1972):

$$S_N = \frac{2(1 - (D_{hub}/D_{sw})^3)}{3(1 - (D_{hub}/D_{sw})^2} \tan\theta = 1.23$$



LES and CMC
computational domain

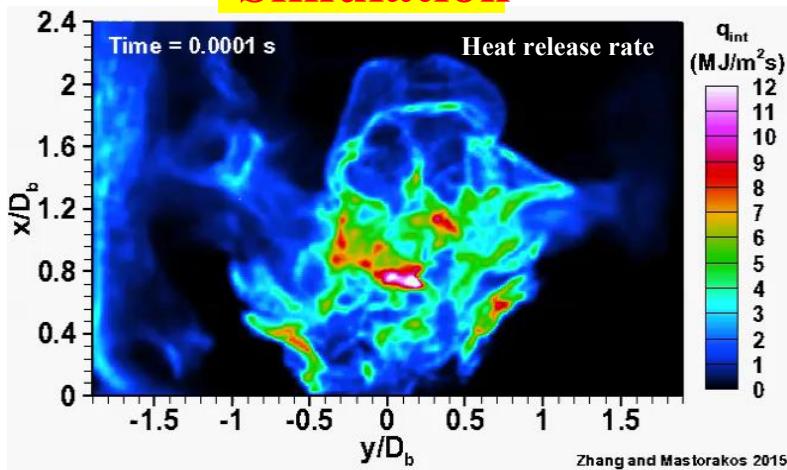
Prediction of global extinction (blow-off) condition in Cambridge burner



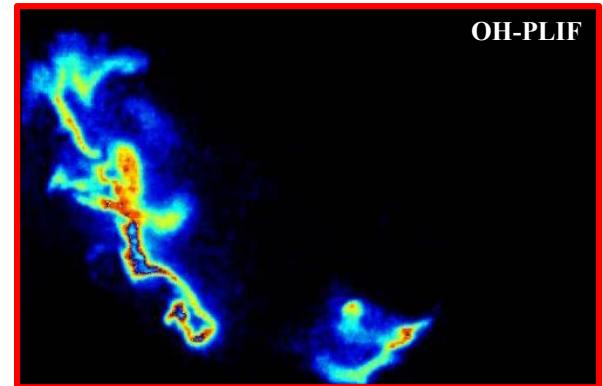
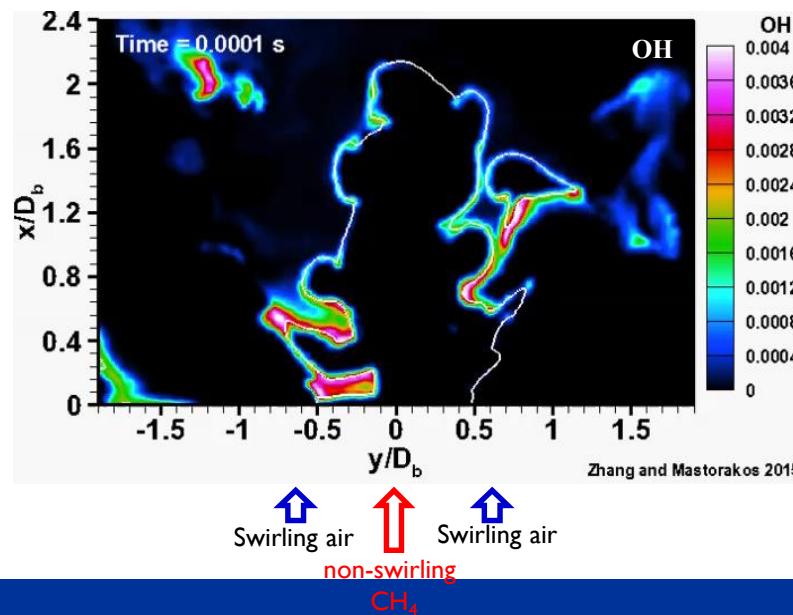
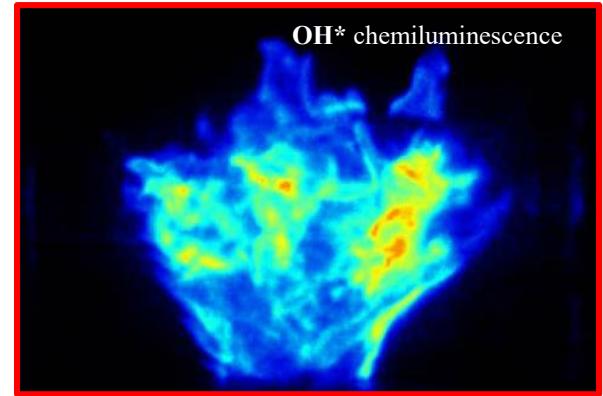
- Prediction of the full blow-off curve is still one of the targets of combustion CFD.
- Capturing the blow-off condition with LES has not been demonstrated yet.

Blow-off transient: LES vs. experiment

Simulation

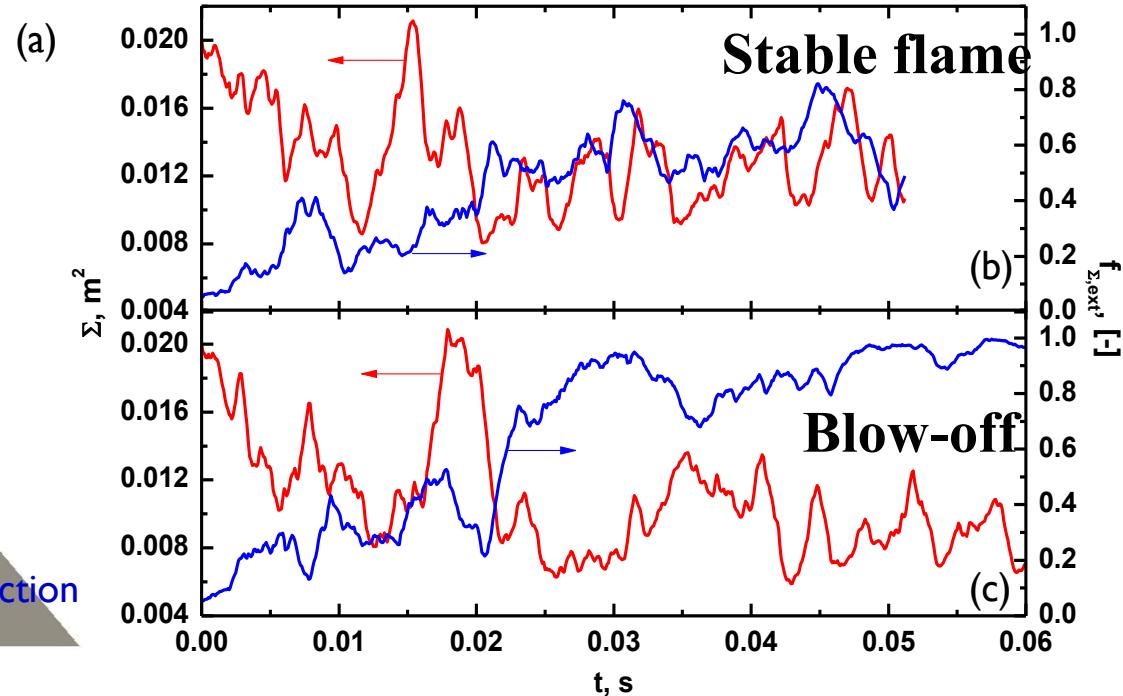
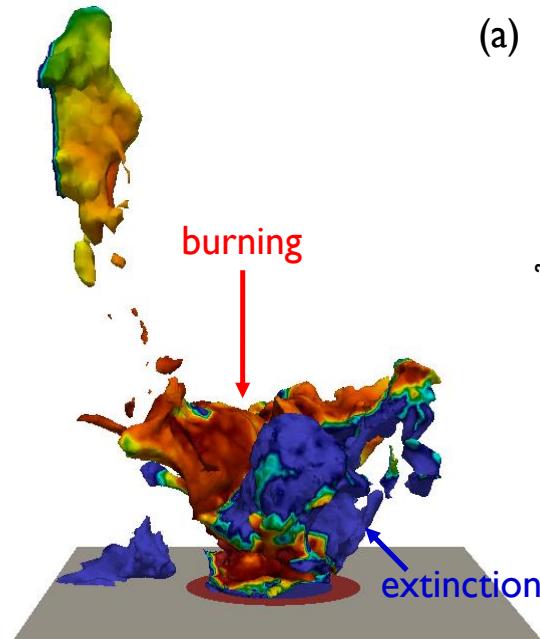


Experiment



(Cavaliere et al. Flow Turb. Combust. 2013)

Localized extinction during blow-off

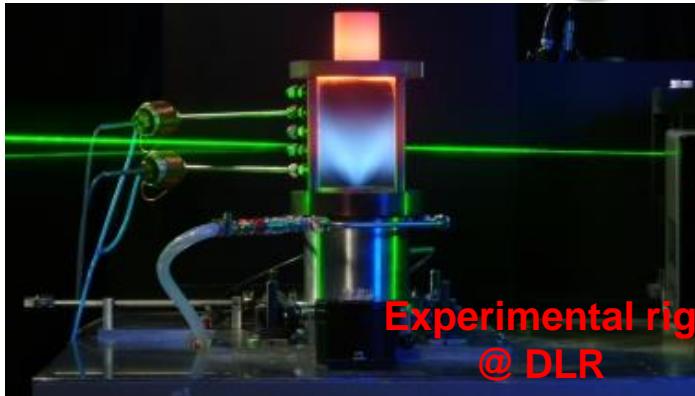


Metrics for quantifying flame extinction:

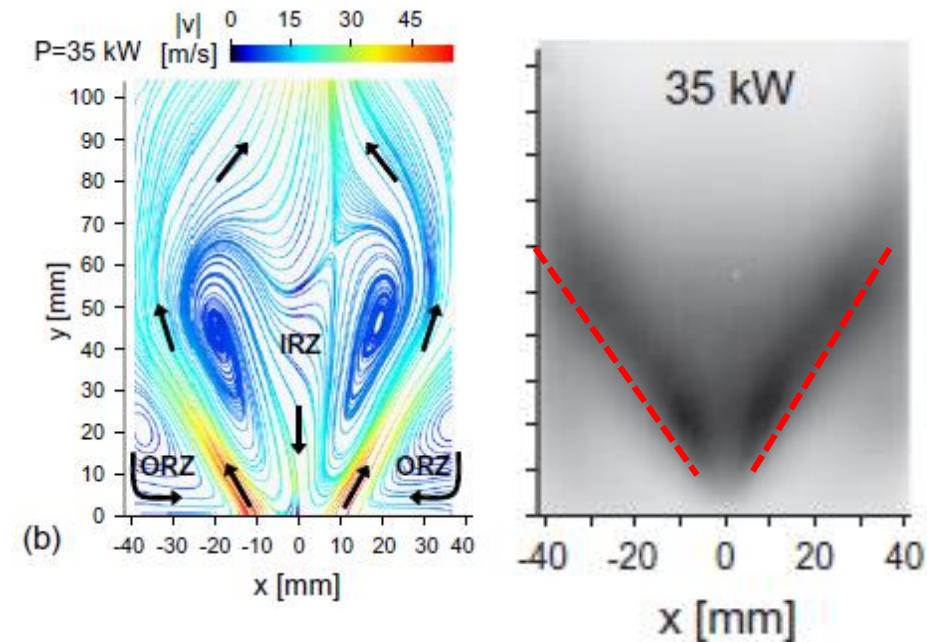
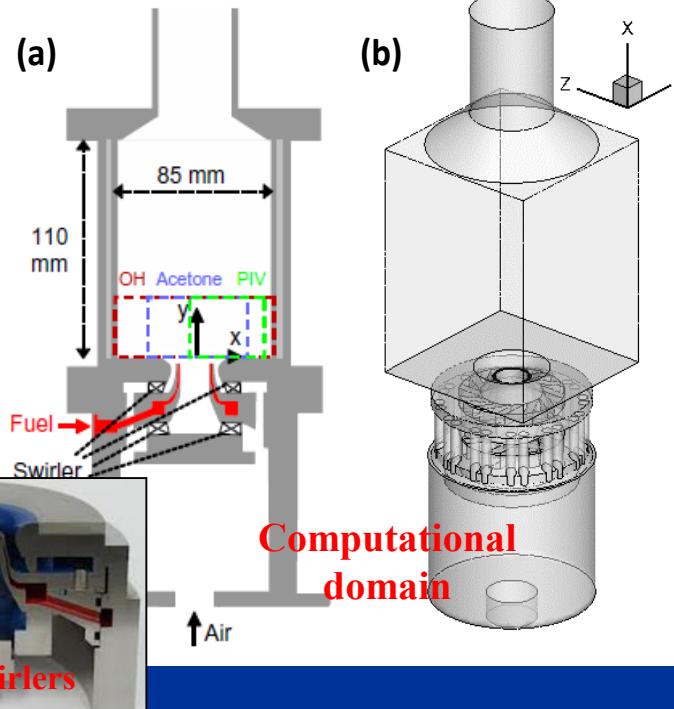
- ❖ Area of the stoichiometric mixture fraction iso-surface Σ
- ❖ Extinguished fraction $f_{\Sigma,ext}$

$$f_{\Sigma,ext} = \frac{\text{extinguished area}}{\text{total stoichiometric area}}$$

model gas turbine combustor



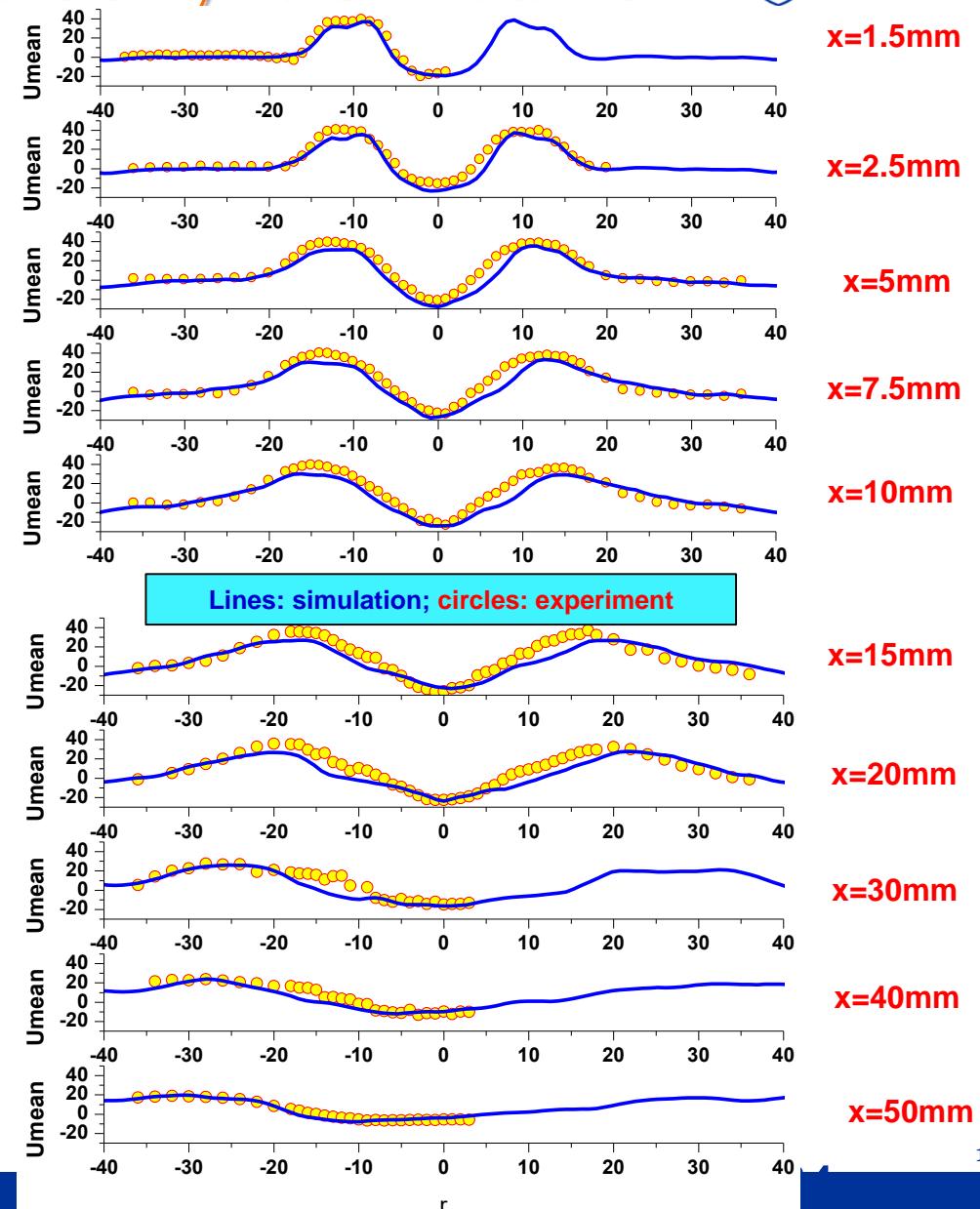
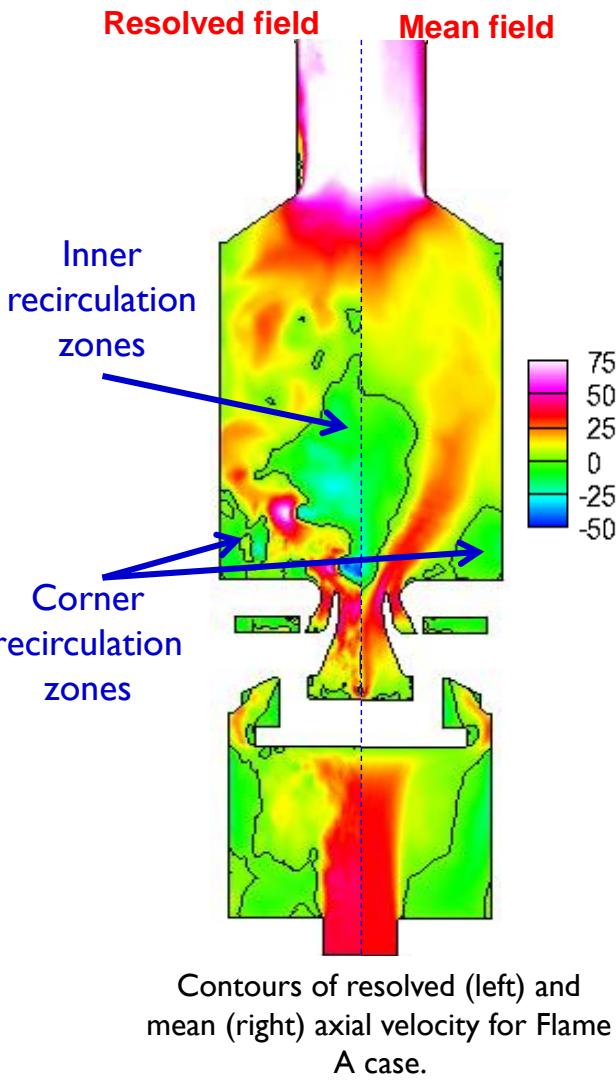
Case	Fuel mass flow rate (kg/s)	Air mass flow rate (kg/s)	Swirl number	Thermal power (kW)	Global equivalence ratio
A	0.000697	0.018	0.9	34.9	0.65
C	0.00015	0.0047	0.55	7.6	0.55



Experimental visualizations of flow pattern and reaction zone of Flame A case

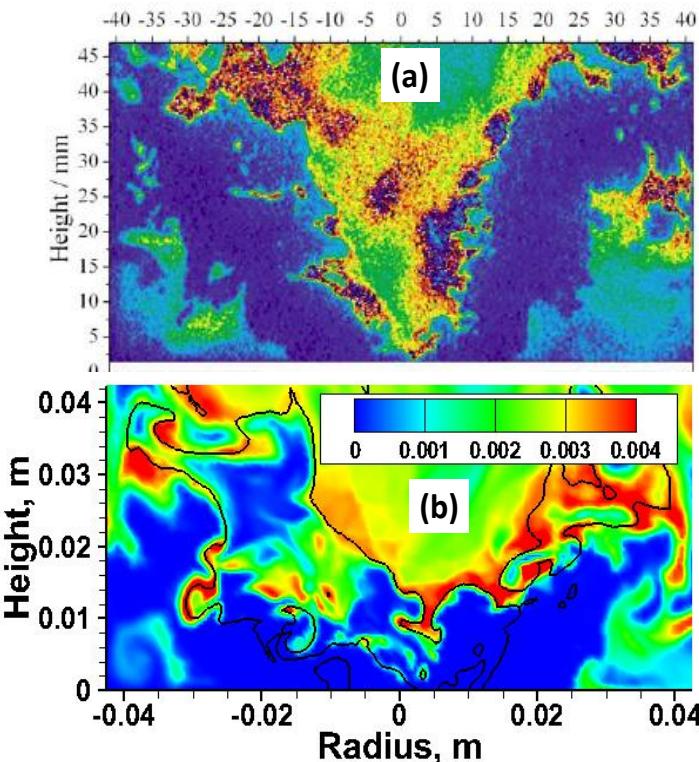
Weigand et al. Combust Flame 2006.
Stohr et al. Combust. Flame 2012.

Axial mean velocity distribution



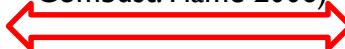
OH mass fraction vs. OH-PLIF

Flame A

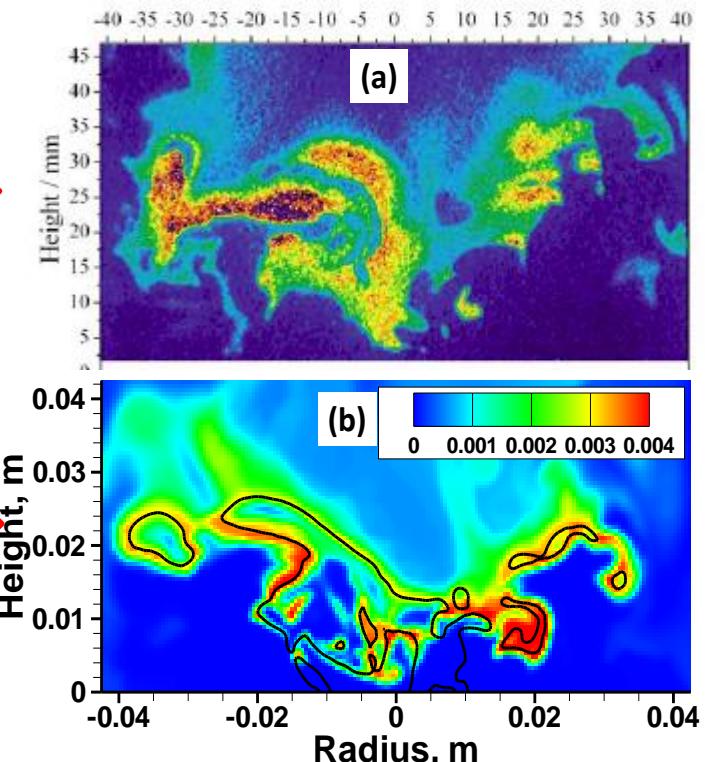


Experiment

(Weigand et al.
Combust. Flame 2006)



Flame C



Simulation



- ✓ Localized extinction?
- ✓ Lean blowout?
- ✓ Partially premixed?