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# Large Eddy Simulation/Conditional Moment Closure modeling of swirl-stabilized non-premixed flames with local extinction

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## Abstract

The Large Eddy Simulation (LES)/three-dimensional Conditional Moment Closure (3D-CMC) model with detailed chemistry and finite-volume formulation is employed to simulate a swirl-stabilized non-premixed flame with local extinction. The results demonstrate generally good agreement with the measurements concerning velocity, flame shape, and statistics of flame lift-off, but the penetration of fuel jet into the recirculation zone is under-predicted possibly due to the over-predicted swirl velocities in the chamber. Localized extinctions are seen in the LES, in agreement with experiment. The local extinction event is shown by very low heat release rate and hydroxyl mass fraction and reduced temperature, and is accompanied by relatively high scalar dissipation. In mixture fraction space, CMC cells with strong turbulence-chemistry interaction and local extinction show relatively large fluctuations between fully burning and intermediate distributions. The probability density functions of conditional reactedness, which shows how far the conditionally-filtered scalars are from reference fully burning profiles, indicate that for CMC cells with local extinction, some reactive scalars demonstrate pronounced bimodality while for those cells with strong reactivity the PDFs are very narrow.

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**Keywords:** Large Eddy Simulation; Conditional Moment Closure; Swirl non-premixed flames; Local extinction; Lift-off

## 1. Introduction

Swirl flows are virtually ubiquitous in gas turbines [1–3]. However, the strong turbulent fluctuations due to the intense shear can lead to finite rate chemistry effects [4] such as local extinction. Extensive efforts have been made to examine local extinction and re-ignition with experiment [4–11] and direct numerical simulations (DNS) [12,13].

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It is well accepted that accurately predicting extinction is an important measure for the performance of advanced turbulent combustion models. Previous efforts include calculations of the Sandia flames (D, E and F) using the Probability Density Function (PDF) method [14] and of the Sydney flames (L, B and M) using the Eulerian stochastic field method [15]. Various versions of the flamelet model [16,17] and higher order Conditional Moment Closure (CMC) models and doubly-conditioned CMC have also been used [18–20]. More recently, the first order CMC model was used to capture local extinction in Sandia F [21] and a swirl spray flame [22] in Large Eddy Simulations (LES). The Sydney swirl diffusion flames also show different levels of turbulence-chemistry interactions [4]. LES has been used for these flames with flamelet and PDF models [23,24]. However, LES focused on the local extinction occurring in swirling non-premixed flames by CMC and other models is still not extensive.

The objective of this study is to apply LES with a multi-dimensional CMC combustion sub-grid model to a swirl-stabilized non-premixed methane flame previously studied experimentally at Cambridge [25] and to analyze the localized extinction. The model is presented in Section 2, while the results and discussion are given in Section 3, followed by the conclusions in the final Section.

2. Modeling

2.1. LES and CMC modeling

The LES equations for mass, momentum, and mixture fraction are obtained through applying low-pass Favre filtering to their instantaneous governing equations. Here, the sub-grid scale stress tensor is closed by the constant Smagorinsky model. The mixture fraction sub-grid variance  $\tilde{\xi}^{n2}$  is modeled by  $\tilde{\xi}^{n2} = C_V \Delta^2 \nabla \tilde{\xi} \cdot \nabla \tilde{\xi}$ , where the constant  $C_V$  is 0.1 [26] and  $\Delta$  is the filter width, taken as the cube root of the LES cell volume. The filtered scalar dissipation rate  $\tilde{N}$  is determined from both resolved and sub-grid contributions as [27]

$$\tilde{N} = \tilde{N}_{res} + \tilde{N}_{sgs} = \underbrace{D \nabla \tilde{\xi} \cdot \nabla \tilde{\xi}}_{resolved} + \underbrace{C_N \mu_t \tilde{\xi}^{n2} / 2 \bar{\rho} \Delta^2}_{sub-grid} \quad (1)$$

in which  $D$  is the molecular diffusivity and  $\mu_t$  is the turbulent viscosity. Here  $C_N = 42$  is used, a constant determined through matching the computational and experimental results in Sandia flame D [21]. This has given good results for the statistics of extinction in Sandia F [21] and for capturing the global blow-off condition of a spray swirl flame [22].

The 3D-CMC equations for non-premixed combustion can be derived through filtering the equations of reactive scalars  $Y_\alpha$  [28,29]

$$\underbrace{\partial Q_\alpha / \partial t}_{T0-unsteady} + \underbrace{\nabla \cdot \widetilde{U} \eta Q_\alpha}_{T1-convection} = \underbrace{Q_\alpha \nabla \cdot \widetilde{U} \eta}_{T2-dilatation} + \underbrace{\widetilde{N} \eta \partial^2 Q_\alpha / \partial \eta^2}_{T3-micromixing} + \underbrace{\widetilde{\omega}_\alpha \eta}_{T4-chemistry} + \underbrace{e_f}_{T5-turbulent\ diffusivity} \quad (2)$$

where  $Q_\alpha = \widetilde{Y}_\alpha \eta$  is the filtered conditional mass fraction of the  $\alpha$ -th species and  $\eta$  is the sample space variable for  $\xi$ .  $\widetilde{U} \eta$ ,  $\widetilde{N} \eta$  and  $\widetilde{\omega}_\alpha \eta$  are the conditionally filtered velocity, scalar dissipation rate and reaction rate, respectively, and need to be closed. Term  $T5$  reads  $e_f = -\nabla \cdot [\bar{\rho} \tilde{P}(\eta) (\widetilde{U} Y_\alpha \eta - \widetilde{U} \eta Q_\alpha)] / \bar{\rho} \tilde{P}(\eta)$  and is modeled with the usual gradient model, i.e.  $e_f = \nabla \cdot (D_t \nabla Q_\alpha)$  [29], with  $D_t$  the turbulent diffusivity modeled as  $D_t = \mu_t / \bar{\rho} Sc_t$  with a turbulent Schmidt number  $Sc_t = 0.7$ .  $\tilde{P}(\eta)$  is the Filtered probability Density Function (FDF). Eq. (2) without the chemical source term is also used for the absolute enthalpy.

Equation (2) is conservative and hence can be discretized in a finite-volume formulation [30,31], which is advantageous for burners with complicated geometries and where CMC resolution may be needed in locations with strong extinction.  $Q_\alpha$  generally change slowly in physical space and thus Eq. (2) is discretized in a coarse CMC mesh independent of the fine LES mesh [28,29]. The unclosed quantities, i.e.  $\widetilde{U} \eta$ ,  $\widetilde{N} \eta$  and  $\widetilde{\omega}_\alpha \eta$ , must be modeled on the CMC mesh. For the conditional velocities  $\widetilde{U} \eta = \tilde{U}$  is assumed [28]. The Amplitude Mapping Closure (AMC) model [32] is used here to first calculate conditional scalar dissipation at the LES mesh, i.e.  $\widetilde{N} \eta^{LES} = N_0 G(\eta)$  where  $N_0$  and  $G(\eta)$  are  $N_0 = \tilde{N} / \int_0^1 G(\eta) \tilde{P}(\eta) d\eta$  and  $G(\eta) = \exp(-2[erf^{-1}(2\eta - 1)]^2)$ , respectively.  $\tilde{N}$  is determined from Eq. (1). Then conditional scalar dissipation at the CMC resolution necessary for solving Eq. (2),  $\widetilde{N} \eta^{CMC}$ , can be obtained through FDF-weighted integration over each CMC cell  $\Omega_{CMC}$  (which is composed of a number of LES cells)

$$\widetilde{N} \eta^{CMC} = \int_{\Omega_{CMC}} \bar{\rho} \tilde{P}(\eta) \widetilde{N} \eta^{LES} d\Omega / \int_{\Omega_{CMC}} \bar{\rho} \tilde{P}(\eta) d\Omega \quad (3)$$

To assist the result interpretation, steady solution of Eq. (2) with  $T_0$ ,  $T_3$ , and  $T_4$  only and a prescribed constant scalar dissipation parameterized by  $N_0$  is referred to as “0D-CMC” (i.e. the transient flamelet equation with unity Lewis number). Finally, first order closure is applied so  $\widetilde{\omega}_\alpha \eta = \omega_\alpha(Q_1, \dots, Q_n, Q_T)$ , where  $n$  is the number

of species.  $\tilde{P}(\eta)$  is presumed to have a  $\beta$ -function shape [33] and is calculated based on  $\tilde{\xi}$  and  $\tilde{\xi}^{\prime 2}$ . The unconditional filtered reactive scalars at the LES resolution,  $\tilde{Y}_\alpha$ , can be obtained by  $\tilde{Y}_\alpha = \int_0^1 \tilde{Y}_\alpha(\eta) \tilde{P}(\eta) d\eta$ .

## 2.2. Problem considered and numerical implementation

The flow considered (Fig. 1) was previously used to investigate extinction in a range of swirl-stabilized flames (premixed, non-premixed, spray) [25], but only the non-premixed flames are considered here. The experimental data include OH\* chemiluminescence and 5 kHz OH-PLIF that have shown flame sheet breaks, attributed to local extinctions. New velocity measurements are included here. The burner consists of (a) a pipe with inner diameter  $D_p = 0.037$  m, at the exit of which a conical bluff body of diameter  $D_b = 0.025$  m is fitted, and which includes a swirler with six 60° guide vanes placed 0.04 m upstream of the bluff body top; (b) a central fuel pipe with  $D_f = 0.004$  m carrying methane; and (c) a  $0.095 \times 0.095 \times 0.15$  m<sup>3</sup> rectangular combustion chamber. The bulk velocity of air  $U_{a,b}$  is 19.1 m/s while that of the methane  $U_{f,b}$  is 29.2 m/s. The swirl number  $S_N$  is 1.23 calculated following Beer and Chigier's formula [34]. The inlet temperatures for both gases are 298 K. The velocities were measured with one-component Laser-Doppler Anemometry with seeding in the air flow only and used 2000 data points resulting to a statistical uncertainty of 1–3%. Note that the absence of seeding in the fuel jet may cause uncertainty in the reported velocities at positions very close to the fuel nozzle, but the error is expected to be negligible elsewhere due to the dominance of the air flow.

The LES computational domain consists of the annulus/swirler, the chamber, and a hemispherical

far-field (not shown in Fig. 1). About 10 million tetrahedral cells are generated for LES with a minimum size  $\Delta_{min} = 0.0004$  m. The CMC domain includes the combustor, far-field and partial annulus section without swirler (starting 0.02 m upstream of bluff body). The CMC mesh consists of around 100,000 polyhedral cells and is refined in the fuel jet and flame regions, thereby leading to 100–200 LES cells within one CMC cell there. This CMC mesh refinement is expected to be useful here because fine local CMC resolution can better capture the response of the conditional reactive scalars to the local variations of scalar dissipation and because it would also aid to accurately capture the interaction of neighboring CMC cells through physical transport (i.e.  $T1$ ,  $T2$  and  $T5$  in Eq. (2)). The mixture fraction space is discretized with 51 nodes clustered around the stoichiometric mixture fraction  $\xi_{st}$ .

At first, an LES with a prescribed steady 0D-CMC solution is performed and this provides the initial fields for the final LES. For the LES boundary conditions, zero pressure gradient is enforced at all inlets. For velocity and mixture fraction, Dirichlet boundary conditions are used. For the air and fuel inlets, random noise with intensity 5% is added to the mean top-hat velocity profiles. At far-field boundaries, the fixed total atmospheric pressure is enforced while for velocities zero gradient extrapolation is applied. At the walls, no slip condition is enforced for the velocities and zero gradient for the mixture fraction.

In CMC,  $\eta = 0$  corresponds to air and  $\eta = 1$  to pure fuel, both at 298 K. The fully burning steady solutions with  $N_0 = 50$  s<sup>-1</sup> from a 0D-CMC calculation are used to initialize all the CMC cells. Inert mixing solutions are injected in the air and fuel inlets and zero gradient conditions are applied for all the solid walls in the CMC domain.

The OpenFOAM LES solver with low Ma number assumption and unstructured finite-volume method is used. The PISO algorithm is used for the velocity–pressure coupling and a second order implicit Crank–Nicholson scheme for time marching. The time step is  $t = 1.5 \times 10^{-6}$  s. A new in-house CMC solver is interfaced with the LES solver. The convective term is discretized with first-order upwind scheme and the diffusion term with second-order central differencing. Full operator splitting is applied here for Eq. (2) and  $\tilde{\omega}_\alpha(\eta)$  is solved with VODPK solver [35]. The ARM2 mechanism is used for the chemistry [36].

In terms of the communication between LES and CMC solvers, the former provides  $\tilde{U}$ ,  $\tilde{\xi}$ ,  $\tilde{\xi}^{\prime 2}$  and  $\tilde{N}$ , while the CMC solver sends back density  $\tilde{\rho}$  and temperature  $\tilde{T}$  for the LES to proceed [28,29]. Parallelized implementations are performed using Message Passing Interface (MPI) libraries for both solvers. The simulations were run on 80 2.6 GHz processors with 4 GB RAM

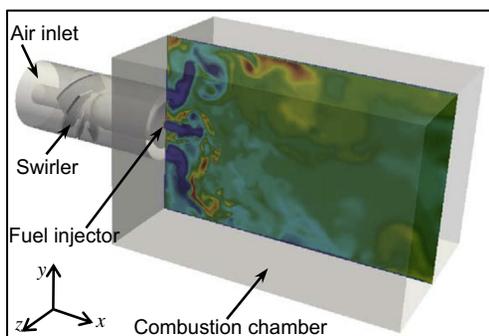


Fig. 1. Schematic of the burner. The colored contour shows an instantaneous distribution of the temperature (blue: 300 K, red: 2100 K). (For interpretation of the references to color in this figure legend, the reader is referred to the web version of this article.)

per processor. 0.001 s of physical time could be completed in about 24 h.

### 3. Results and discussion

#### 3.1. Velocity statistics and global flame characteristics

Radial profiles of normalized mean and r.m.s. axial velocity are shown in Fig. 2. The computational statistics were collected spanning 0.032 s; different sampling periods were examined to confirm statistical convergence. The flow reversals in both the Corner Recirculation Zone (CRZ) close to the chamber walls and the Inner Recirculation Zone (IRZ) are correctly captured. The IRZ extends from the bluff body top to about 0.18 m downstream (i.e. beyond the chamber). The decay of the fuel jet is over-predicted by the LES and thus its penetration into the IRZ is shorter (see  $x/D_b = 2.2$  in Fig. 2c) compared to the experiment. This is likely due to the fact that the simulation does not reproduce very well the swirl velocity (not shown here) at the chamber inlet. The simulations over-predict the r.m.s. close to the bluff body, but the overall shape of the distribution follows the experiment and the agreement improves downstream.

Instantaneous contours of selected scalars and scalar dissipation rates are plotted in Fig. 3 and the time-averaged heat release rate  $\tilde{q}$  and hydroxyl (OH) mass fraction  $\tilde{Y}_{OH}$  are compared to experiment in Fig. 4. The instantaneous resolved heat release rate  $\tilde{q}$  in Fig. 3(b) is calculated from  $\tilde{q} = \int_0^1 \tilde{q}|\eta| \tilde{P}(\eta) d\eta$  where  $\tilde{q}|\eta| = \sum_{\alpha=1}^n W_{\alpha} h_{\alpha} \omega_{\alpha} |\eta|$  and  $W_{\alpha}$  and  $h_{\alpha}$  are the molecular mass and specific enthalpy of the  $\alpha$ -th species, respectively. An instantaneous image of OH from the experiment [25] is also shown next to the computational  $\tilde{Y}_{OH}$  in Fig. 4 to show some qualitative features of the prediction. The white iso-lines showing the stoichiometric mixture fraction are highly distorted due to the strong turbulence. The flame is short ( $x < 0.05$  m), severely fragmented and con-

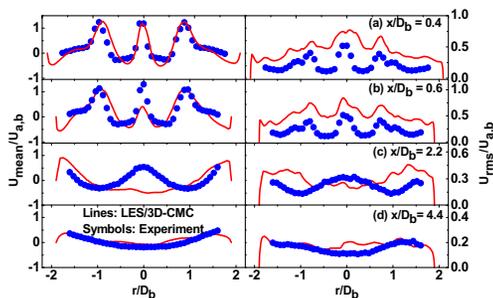


Fig. 2. Radial profiles of mean and r.m.s. axial velocity from the LES (lines) and experiment (symbols).

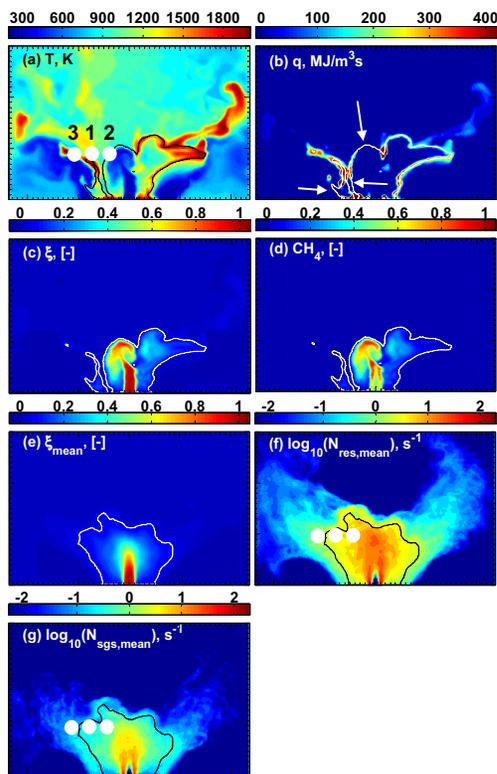


Fig. 3. Instantaneous (a) temperature, (b) heat release rate, (c) mixture fraction, (d)  $\text{CH}_4$  mass fraction, (e) mean mixture fraction, and logarithm of mean (f) resolved and (g) sub-grid scalar dissipation. Images shown are  $0.095 \times 0.06$  m. Lines: instantaneous or mean stoichiometric mixture fraction. Solid circles: Cells CMC1, CMC2 and CMC3. Arrows point to low heat release rate regions.

finned at the boundary of the IRZ and therefore is somewhat different from the unconfined Sydney swirl diffusion flames [4]. Negligible  $\tilde{q}$  along the  $\xi_{st}$  iso-line (pointed by arrows in Fig. 3b) can occasionally be observed. The mixture fraction decays very quickly due to the fast mixing in the IRZ. The methane ( $\text{CH}_4$ ) is consumed inside the IRZ. It can be seen from Fig. 4(b) that  $\tilde{Y}_{OH}$  is very localized, typically coincides with the  $\xi_{st}$  iso-lines, and qualitatively in line with the OH-PLIF image in Fig. 4(a), showing breaks along the front (pointed by arrows in Figs. 4a and b). The intermittent lift-off from the corner of bluff-body observed in the experiment is also seen in the LES (see the left flame branch in Fig. 4b). The time-averaged  $\tilde{Y}_{OH}$  and  $\tilde{q}$  in Fig. 4(d) and (f) shows a mean flame shape in reasonable agreement with the experiment. The time-averaged mixture fraction shows effective mixing (Fig. 3e) and the average scalar dissipation is high along the fuel jet (Fig. 3f and g), with the resolved higher

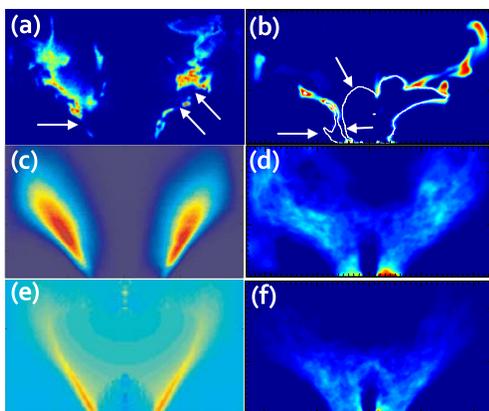


Fig. 4. Comparisons of the numerical predictions (right column: b, d, f) with experimental data [25] (left column: a, c, e): (a, b) instantaneous and (c, d) mean OH-PLIF from the experiment and simulated OH mass fraction, (e) mean OH\* chemiluminescence after inverse Abel transform from the experiment and (f) mean heat release rate from the simulation. Images shown are  $0.095 \times 0.05$  m. Arrows point to low OH regions.

than the estimated sub-grid in most locations. Immediately downstream of the fuel jet tip along the axis,  $\tilde{N}$  is high and thus in most instants the flame is quenched there.

Since the present simulations consider adiabatic wall conditions, we may expect that we over-predict the heat release rate and OH mass fraction near the bluff body surface (see Figs. 4d and f) compared to experiment. A more detailed comparison with the measured statistics of lift-off will be presented later once the nature of local extinction is discussed. In general, we consider the agreement relatively satisfactory and this helps build confidence for further analysis.

### 3.2. Identification and quantification of local extinction

Figure 5 shows  $\tilde{q}$ ,  $\tilde{Y}_{OH}$ ,  $\tilde{T}$ , and  $\tilde{N}_{res}$  on a  $y$ - $z$  slice ( $x/D_b = 0.59$ ). Numbers 1–5 mark different positions along the  $\xi_{st}$  iso-lines. It can be seen from Fig. 5(a) that in Points 1, 4, and 5,  $\tilde{q}$  is comparatively high, while in Points 2 and 3 it is very low. In local extinction, nearly frozen reactivity should be reached and  $\tilde{q}$  is indeed negligible in Points 2 and 3. From Figs. 5(b)–(d) it can be clearly seen that Point 3 corresponds to low  $\tilde{Y}_{OH}$  and  $\tilde{T}$  and high  $\tilde{N}_{res}$  and thus undergoes localized extinction. Point 2, however, has very low  $\tilde{q}$  but high  $\tilde{Y}_{OH}$  and  $\tilde{T}$  and nearly zero  $\tilde{N}_{res}$ . This state corresponds to very fast chemistry and very low mixing rate (thus very large Damköhler number) and so the local heat release rate is very small. It is interesting how, in the same instant, different locations can have such widely different Damköhler

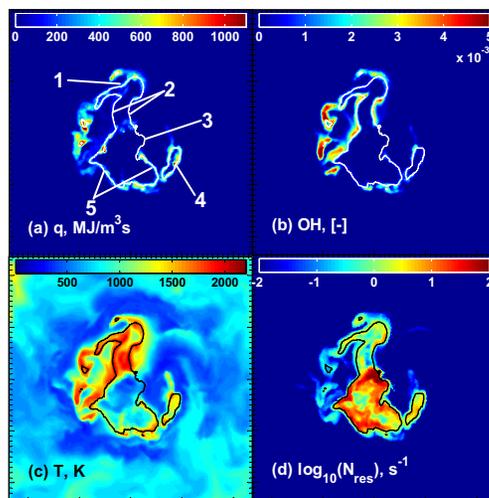


Fig. 5. Instantaneous contours in the  $y$ - $z$  plane ( $x/D_b = 0.59$ ) of (a)  $\tilde{q}$ , (b)  $\tilde{Y}_{OH}$ , (c)  $\tilde{T}$  and (d)  $\tilde{N}_{res}$ . Images shown are  $0.095 \times 0.095$  m. Lines: stoichiometric mixture fraction.

numbers. From an extinction identification perspective, exclusively taking the local low heat release rate as the criterion to identify the local extinction could therefore over-predict the degree of extinction. Points 1, 4 and 5 can be considered to be reactive (non-zero  $\tilde{q}$ ) although from Fig. 5(b) very small  $\tilde{Y}_{OH}$  can be seen. It is also seen that very low or medium scalar dissipation  $\tilde{N}_{res}$  occurs there. OH-based identification has been widely used for the qualitative or quantitative analysis in the both experimental and computational work [6,8–10,15], but perhaps a more rigorous criterion for localized extinction is that there is low  $\tilde{q}$ ,  $\tilde{Y}_{OH}$ , and  $\tilde{T}$ , and large  $\tilde{N}_{res}$ .

To indicate how the conditional reactive scalars  $Q_x$  evolve during localized extinction, the time series of  $\tilde{q}|_{\xi_{st}}$ ,  $\tilde{Y}_{OH}|_{\xi_{st}}$ ,  $\tilde{T}|_{\xi_{st}}$  and  $\tilde{N}|_{\xi_{st}}$  are plotted in Fig. 6. These data spanning about 0.005 s are extracted from one representative CMC cell in the flame region ( $x/D_b = 0.59$ ,  $y/D_b = 0$ ,  $z/D_b = 0.56$ ; it is marked by the middle solid circle in Fig. 3a and termed as CMC1 hereafter). As a reference, the extinction scalar dissipation rate at  $\eta = \xi_{st}$  from the steady flamelet calculation,  $N|_{\xi_{st},0D} = 18.3 \text{ s}^{-1}$ , is denoted in Fig. 6(d). To quantify the local extinction in  $\eta$ -space, thresholds for  $\tilde{q}|_{\xi_{st}}$ ,  $\tilde{Y}_{OH}|_{\xi_{st}}$ , and  $\tilde{T}|_{\xi_{st}}$  are defined as 10%, 10%, and 70% of their corresponding steady flamelet counterparts from a simulation with  $N_0 = 174 \text{ s}^{-1}$  (the indicated value of  $N|_{\xi_{st},0D} = 18.3 \text{ s}^{-1}$  corresponds to this  $N_0$  in the stand-alone 0D-CMC using the AMC shape for the scalar dissipation), which is very close to extinction. These thresholds are  $380 \text{ MJm}^{-3}\text{s}^{-1}$ , 0.00024 and 1245 K, respectively, and are included in

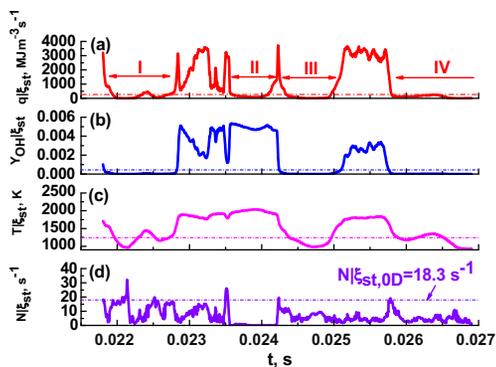


Fig. 6. Time series of conditionally filtered (a) heat release rate, (b) OH mass fraction, (c) temperature and (d) scalar dissipation rate at  $\eta = \xi_{st}$  for CMC1. Dash-dotted lines: the thresholds for defining extinction.

**Figs. 6(a)–(c).** From **Figs. 6(a)** and **(d)**, initially  $\widehat{q}|\xi_{st}$  corresponds to fully burning state and then sharply decreases with gradually increased  $\widehat{N}|\xi_{st}$ , which is very close to  $N|\xi_{st,0D}$ . This period (Period I) lasts about 0.001 s and local extinction occurs (very low  $\widehat{q}|\xi_{st}$  and  $Y_{OH}|\xi_{st}$  and low  $\widehat{T}|\xi_{st}$  with relatively high  $\widehat{N}|\xi_{st}$ ). Then at  $t = 0.0228$  s,  $\widehat{N}|\xi_{st}$  decays far below  $N|\xi_{st,0D}$  and  $Y_{OH}|\xi_{st}$  and  $\widehat{T}|\xi_{st}$  increase. From  $t = 0.0236$  s to  $t = 0.0242$  s (Period II), very low heat is liberated locally while the scalar dissipation is nearly zero. Period II has the fully-burning composition. At  $t = 0.0242$  s, CMC1 experiences a sharp increase of  $\widehat{N}|\xi_{st}$ , and the conditional reactive scalars seem quenched again. This extinction event lasts around 0.0008 s (Period III) during which  $\widehat{N}|\xi_{st} < N|\xi_{st,0D}$ . At  $t = 0.025$  s, CMC1 is ignited again and is reactive until  $t = 0.0258$  s when a peak  $\widehat{N}|\xi_{st}$  appears. Then CMC1 is extinguished (Period IV) again. Therefore, local extinction events occur in Periods I, III and IV, while for Period II we have combustion with very high Damköhler number. For Periods I, III and IV the quantities,  $\widehat{q}|\xi_{st}$ ,  $Y_{OH}|\xi_{st}$ , and  $\widehat{T}|\xi_{st}$  fall below their individual thresholds mentioned above. Re-ignition will not happen automatically as soon as  $\widehat{N}|\xi_{st}$  drops from the critical value, which is also observed in LES/CMC of the Sandia jet flames [21] and in DNS of planar non-premixed ethylene jet flames [13]. In Ref. [13], this delay is attributed to the increased scalar dissipation rate after the onset of local extinction. However, based on the present results (e.g. Period III) and Ref. [21], it can also be caused by the inadequate diffusion of heat and species from the neighbouring CMC cells and in this sense re-ignition events are affected by both local scalar dissipation and transport. In Ref. [13], this mechanism is loosely

referred to as re-ignition by premixed flame propagation, while in the present CMC equation such a mechanism is captured by the physical transport terms  $T1$  and  $T5$ . Further discussion on how LES/CMC captures re-ignition is given in [21]. Note that the minimum physical dimension of a local extinction in the present model is limited by the CMC cell size. The unstructured CMC grid methodology used here can allow greater resolution in regions where extinctions are expected.

$\widehat{q}|\eta$ ,  $\widehat{T}|\eta$ ,  $Y_{OH}|\eta$  and  $Y_{CH_4}|\eta$  from CMC1 are plotted in Fig. 7 for the whole simulated time. For reference, the 0D-CMC results with high ( $N_0 = 170s^{-1}$ ), intermediate ( $N_0 = 50s^{-1}$ ), and low ( $N_0 = 5s^{-1}$ ) scalar dissipation rates are also shown. A wide scatter (quantified by the large r.m.s. in Fig. 7) can be observed for all quantities, indicating the frequent transition between burning and frozen conditional distributions. During the extinction event, the conditional profiles of  $Y_{OH}|\eta$  and  $\widehat{q}|\eta$  are very close to the inert mixing lines (i.e. zero).  $\widehat{T}|\eta$  and  $Y_{CH_4}|\eta$  in contrast take time to diffuse to the inert values and therefore very cold temperatures and inert fuel  $\eta$ -space distributions are not reached.

In order to quantify the deviation from the fully burning state and hence the occurrence of local extinction in mixture fraction space, the conditional reactivity [11] is calculated for each CMC cell by  $\widehat{b}_x|\eta = (Y_x|\eta - Y_{x,m}|\eta)/(Y_{x,b}|\eta - Y_{x,m}|\eta)$ , where  $Y_{x,b}|\eta$  and  $Y_{x,m}|\eta$  come from fully burning 0D-CMC solutions with  $N_0 = 5s^{-1}$  (which is about 3% of the extinction value) and inert mixing, respectively. Clearly,  $\widehat{b}_x|\eta = 0$  when

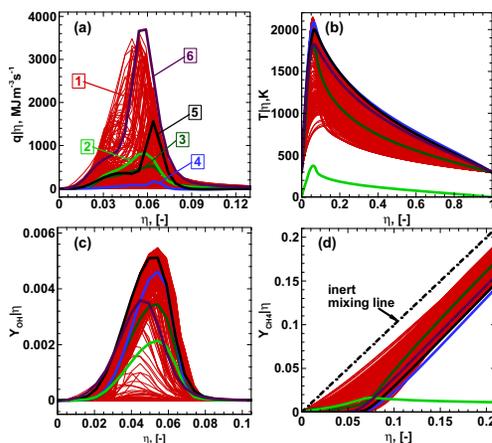


Fig. 7. Conditional (a) heat release rate, (b) temperature and mass fractions of (c) OH and (d) CH<sub>4</sub> for CMC1. Line 1 indicates instantaneous conditional profiles while Lines 2 and 3 their r.m.s. and mean, respectively. Lines 4–6 indicate stand-alone 0D-CMC solutions with low, medium and high scalar dissipation ( $N_0 = 5, 50$ , and  $170 s^{-1}$ ), respectively.

the quenching occurs, while it is close to or larger than unity when the conditional solutions are fully burning [11]. The PDFs of the conditional reactedness at stoichiometry for temperature and hydroxyl, nitric oxide (NO), carbon monoxide (CO) mass fractions for three CMC cells (CMC1, CMC2 and CMC3 marked by the three white solid circles in Fig. 3a) are shown in Fig. 8. Most samples of  $T|\xi_{st}$  fall within  $0.9 \leq b_T|\xi_{st} \leq 1.1$ , but there are still considerable data in  $b_T|\xi_{st} \leq 0.6$ , indicating that the conditional temperatures in these two cells experience frequent extinction (Fig. 8a), consistent with the existence of the peak at 0 for the OH (Fig. 8b). The PDFs of  $b_{Y_{NO}}|\xi_{st}$  and  $b_{Y_{CO}}|\xi_{st}$  in Fig. 8(c) and (d) are relatively broad. For NO, two peaks exist at  $b_{Y_{NO}}|\xi_{st} \approx 0.1$  and  $b_{Y_{NO}}|\xi_{st} \approx 0.5 \sim 0.6$ . For CMC1 and CMC2 that lie in the high scalar dissipation rate region (see Fig. 3f and g), the PDFs of stoichiometric reactedness for temperature and OH have obvious bimodality, while for NO and CO this feature is less pronounced. This is likely to be because the chemistry of NO and CO will freeze during an extinction event and hence these species will be evident for some time in  $\eta$ -space. CMC3 is located in a region with low scalar dissipation (see Fig. 3f and g) and so no local extinction happens and therefore the PDF of  $b_T|\xi_{st}$  is concentrated at high values and  $b_{Y_{OH}}|\xi_{st}$ ,  $b_{Y_{NO}}|\xi_{st}$  and  $b_{Y_{CO}}|\xi_{st}$  have wide distributions. The conditional mass fractions of OH, NO and CO have large fluctuations in spite of the relatively continuous strong reactivity in CMC3 due to the fluctuations of the scalar dissipation and the frequent quenching of the neighbouring cells.

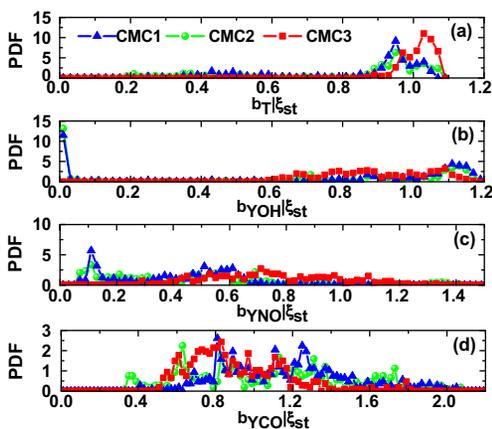


Fig. 8. Probability density functions of reactedness at  $\xi_{st}$  from (a) temperature, mass fractions of (b) OH, (c) NO and (d) CO for CMC1 (triangles), CMC2 (circles) and CMC3 (squares).

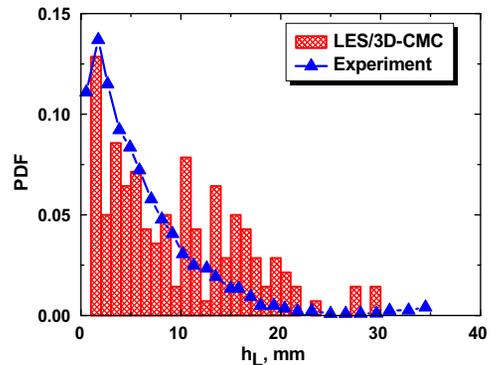


Fig. 9. Probability density functions of lift-off height. Histogram: LES/3D-CMC, line: experiment [25].

### 3.3. Lift-off

In the present swirl non-premixed flame, lift-off from the bluff-body surface (shown in Fig. 3b and b) is an important dynamic phenomenon caused by local extinction at the flame base. In this subsection, the statistics of the lift-off height  $h_L$  in the  $x$ - $y$  plane are investigated. To be consistent with the experiment [25],  $h_L$  is defined as the streamwise distance between the bluff body to the position along the  $\xi_{st}$  iso-line where  $\tilde{Y}_{OH}$  critically exceeds the 0D-CMC threshold (i.e. 0.00024). 200 samples are extracted equally from both left and right flame branches and 140 samples of those showed lifted flame. We consider here only snapshots with  $h_L > 0.0005$  m, which is about one nominal CMC cell size near the bluff body edge.

The PDFs of  $h_L$  from the measurement [25] and the LES are shown in Fig. 9. The mean lift-off height from LES/3D-CMC is 9.75 mm while that from measurement is 5 mm, but the overall shape is consistent and the long tail at large lift-off heights is reproduced. The fact that lift-off height at about 10–20 mm is more probable in the simulation results may be due to the overprediction of turbulence intensity close to the bluff body as shown in Fig. 2, thereby leading to more intensive extinction at those distances. The results in Fig. 9 further demonstrate the overall satisfactory capability of LES/3D-CMC in reproducing the local extinction in swirl non-premixed flames.

## 4. Conclusions

The LES/3D-CMC model with detailed chemistry is used to simulate a swirl-stabilized non-premixed flame with local extinction. The conservative 3D CMC equations are used to extend the applications of the CMC model to complex geometries. The LES/3D-CMC simulations reproduce reasonably the flow field and the

global flame characteristics (e.g. mean OH-PLIF, OH\* chemiluminescence, PDF of lift-off height). The occurrence of localized extinction is typically manifested by low heat release rate and OH mass fraction and low or medium temperature and is accompanied by high scalar dissipation rates. In mixture fraction space the CMC cells undergoing local extinction have relatively wide scatter between inert and fully burning solutions while for fully burning CMC cells the instantaneous CMC solutions follow closely fully-burning distributions. A reactedness index is introduced to quantify how far the conditional profiles deviate from the reference fully burning state. The PDFs of reactedness at the stoichiometric mixture fraction demonstrate some extent of bimodality, showing the events of local extinction and re-ignition and their relative occurrence frequency.

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