

Lecture 6 Conservation Laws

We know about vectors and tensors, how they transform, some of their properties (principal axes, eigenvalues + eigenfunctions), and how to use them to describe flow (material + Eulerian coordinates, deformation + rotation, stream functions). We are now ready to put in the physics.

⇒ The big secret is the physics is really simple and the same for everything from your bank account to cars in a carpool to conservation of mass, momentum, and energy (to the limit of fluid dynamics).

Consider, for example, fluid density. A material element of fluid with density ρ must change its volume if its density changes.

Definition of divergence:

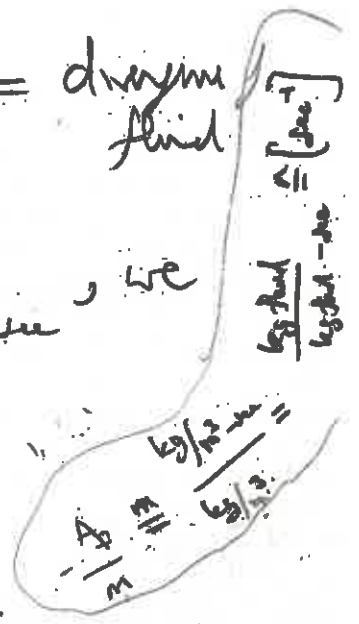
$$\frac{1}{\rho} \frac{D\rho}{Dt} \equiv -\frac{1}{\rho} \frac{D\rho}{Dt} = -\nabla \cdot \underline{u}$$

$$\nabla \cdot \underline{u} = \frac{\text{Volume chge}}{\text{unit volume}} = \text{diverge fluid}$$

If A is a fluid source in $\frac{\text{kg fluid}}{\text{kg fluid} \cdot \text{sec}}$, we

notes
This book
starting from
Lecture 1, not to
do the math.

can express conservation / mass:



(5-1a)
Conservation / fluid
mass

$$\frac{1}{\rho} \frac{D\rho}{Dt} + \nabla \cdot \underline{u} = \frac{A}{\rho}$$

$\left. \begin{matrix} \text{kg/m}^3 \cdot \text{sec} \\ \text{sec}^{-1} \end{matrix} \right\} = \text{kg/m}^3 \cdot \text{sec}$

This is the conservation / mass equation for a material element. If there are no fluid sources, so $A = 0$, material element expansion ($\nabla \cdot \underline{u}$) must decrease the density of the material (assuming of course some mass material is considered)

It's worth doing this in Eulerian coordinates

to gain added insight. For a volume fixed in

Spool: A

Control fixed in space

kg of fluid in volume (cont)

flow out of volume

addition of mass / subtraction

just logic of cons in control

$$\frac{d}{dt} \int_V \rho dv + \int_A \rho \underline{u} \cdot d\underline{A} = \int_V A_f dv$$

kg/m³

Since:

depending on t

$$\frac{d}{dt} \int_V \rho dv = \int_V \frac{\partial \rho}{\partial t} dv$$

fluid vol. in control

and:

$$\int_A \rho \underline{u} \cdot d\underline{A} = \int_V \nabla \cdot (\rho \underline{u}) dv$$

by Gauss law

Then

$$\int_V \frac{\partial \rho}{\partial t} dv + \int_V \nabla \cdot (\rho \underline{u}) dv = \int_V A_f dv$$

$$= \int_V \left(\frac{\partial \rho}{\partial t} + \nabla \cdot \rho \underline{u} - A_f \right) dv = 0$$

Since this must hold everywhere,

Continuity equation (no

$$\frac{\partial \rho}{\partial t} + \nabla \cdot \rho \underline{u} = A_f$$

Source sink (fluid)

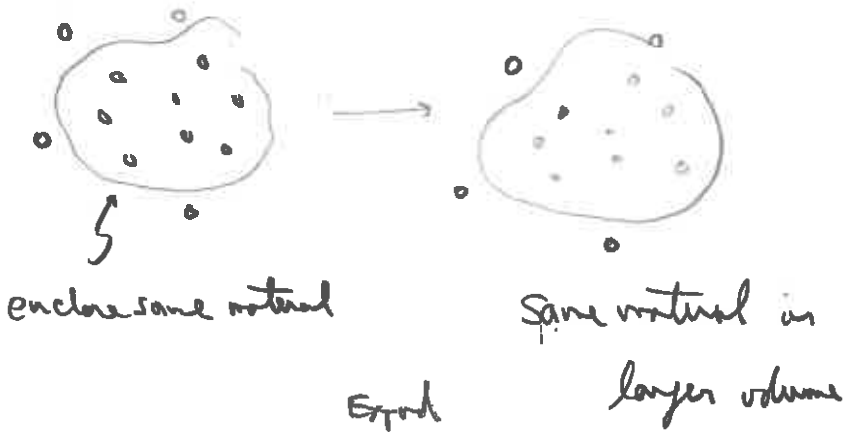
This is same as (5-16), since $\nabla \cdot \rho \underline{u} = \underline{u} \cdot \nabla \rho + \rho \nabla \cdot \underline{u}$

and $\frac{D\rho}{Dt} = \frac{\partial \rho}{\partial t} + \underline{u} \cdot \nabla \rho$ so: $\frac{D\rho}{Dt} + \rho \nabla \cdot \underline{u} = A_f$, as type

See next page

The Lagrangian or Eulerian description are the same and can go both ways:

$$\frac{1}{\rho} \frac{D\rho}{Dt} + \nabla \cdot \underline{u} = 0$$



Lagrangian

Note - we used this for derivation of streamlines

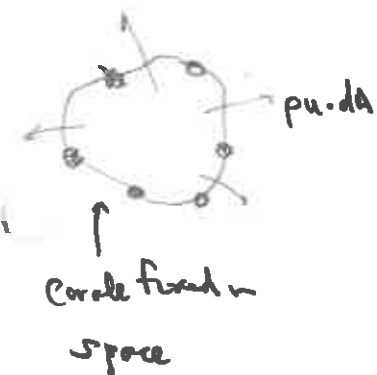
- Therefore density decreased

Chain Rule

$$\frac{D}{Dt} = \frac{\partial}{\partial t} + \underline{u} \cdot \nabla$$

$$\frac{1}{\rho} \frac{\partial \rho}{\partial t} + \frac{\underline{u} \cdot \nabla \rho}{\rho} + \nabla \cdot \underline{u} = 0$$

$$\frac{\partial \rho}{\partial t} + \underline{u} \cdot \nabla \rho + \rho \nabla \cdot \underline{u} = 0$$



$$\frac{\partial \rho}{\partial t} + \nabla \cdot \rho \underline{u} = 0$$

Eulerian

Flux out of fixed control determines change in density

Conservation of a chemical species

Now consider a fluid with a dissolved

chemical in it. Let $C \equiv \frac{\text{kg chemical}}{\text{kg fluid}} = \text{mass fraction } C$, then
 by mass Eulerian logic:

$$\int_V \frac{\partial \rho C}{\partial t} = - \int_A \rho \underline{u} \cdot d\underline{A} + \int_V A_c \left[\frac{\text{kg } C}{\text{m}^2 \cdot \text{sec}} \right] dV$$

$$\frac{\partial \rho C}{\partial t} + \nabla \cdot \rho \underline{u} C - A_c = 0$$

\downarrow
 $\partial_i \rho u_i$

$$\rho \frac{\partial C}{\partial t} + C \frac{\partial \rho}{\partial t} + C \partial_i \rho u_i + \rho u_i \partial_i C - A_c = 0$$

$$\rho \frac{DC}{Dt} + (C A_f - A_c) = 0$$

$$\rho \frac{DC}{Dt} = (A_c - C A_f)$$

(Equation 4.6)
 about
 $\rho \frac{DC}{Dt}$ more
 multiple velocity instead
 but operator of
 C (not ρ)

$\left[\frac{\text{kg } C}{\text{m}^2 \cdot \text{sec}} \right]$
 $\left[\frac{\text{kg fluid}}{\text{m}^2 \cdot \text{sec}} \right]$
 (same as A_c)

$$\frac{D}{Dt} \int_V \rho C dV = \int_V \rho \frac{DC}{Dt} dV$$

If no fluid source, $A_f \rightarrow 0$
 and increase in C reduced
 by introducing C at $\frac{\text{kg } C}{\text{m}^2 \cdot \text{sec}} = A_c$

If inject fluid at A_f $\frac{\text{kg fluid}}{\text{kg fluid} \cdot \text{sec}}$

$$A_c = C_{inj} A_f$$

some term w. $A_f (C_{inj} - C)$

Make sense!
 If $C_{inj} = C$, $A = 0$ or should be \rightarrow

↳ Generalization - all conservation equations are in same form
 So, we can generalize. Consider any conserved quantity Θ

Whether G is advected quantity kg i / kg fluid,
 momentum measured in momentum / kg fluid, or
 energy measured in energy with / kg fluid etc, we can write a
general conservation equation for Θ :-

$\Theta \equiv$ conserved quantity (e.g. defined mass, momentum, energy)

$G = \Theta / \text{kg fluid}$

$J_{\Theta} =$ non-advective flux / Θ
 across a unit area

$A_{\Theta} =$ source of Θ per unit volume of space \rightarrow

Then conservation of Θ can be written: add flux out (as diffusive)

$$\left(\frac{D}{Dt} \right) \int_{\mathcal{V}} \rho G dV = \int_{\mathcal{V}} \left(\rho \frac{DG}{Dt} + (\rho G A_f - A_{\Theta}) dV \right) + \int_{\mathcal{A}} J_{\Theta} \cdot dA = 0$$

Rule $\rightarrow \mathcal{V}$
more DG or in back let \rightarrow on part \leftarrow
source \rightarrow

$$\rho \frac{DG}{Dt} + \int_{\mathcal{V}} \nabla \cdot J_{\Theta} dV$$

So:

$$\int_V \left(\rho \frac{DG}{Dt} + \nabla \cdot \mathbf{J}_G + (G A_f - A_G) \right) dV = 0$$

Since This relation must hold everywhere

$\frac{G}{\text{kg m}^3}$

$\frac{\text{kg/m}^2}{\text{m}^2 \cdot \text{sec}}$

$\frac{G}{\text{m}^2 \cdot \text{sec}}$

$\frac{G}{\text{m}^2 \cdot \text{sec}}$

(5-2)
General Continuity
equation

$$\rho \frac{DG}{Dt} + \nabla \cdot \mathbf{J}_G + (G A_f - A_G) = 0$$

Now consider some particular examples:

A. Conservation of chemical mass (again)

$G = \text{kg chemical } c = M_c$

$G = \frac{\text{kg chemical } c}{\text{kg fluid}} = C = \frac{M_c}{M_f}$

diffusion tensor

$\frac{M_c}{M_f} = \frac{M_c}{M_f}$

$\frac{M_c}{M_f} = \frac{M_c}{M_f}$

$\frac{M_c}{M_f} = \frac{M_c}{M_f}$

chemical mass

$A_G = \frac{\text{kg chemical } c}{\text{unit volume}} / \text{unit time}$

$\mathbf{J}_G = \text{kg of } c \text{ diffusing across an area per unit time} = \underline{\underline{D \cdot \nabla pC}}$

$\frac{\text{m}^2}{\text{sec}} \frac{1}{\text{m}} \frac{\text{kg}}{\text{m}^3} = \frac{\text{kg}}{\text{m}^2 \cdot \text{sec}}$

where $C = \text{mass fraction of } c$

$\underline{\underline{D}} = \text{diffusion tensor in } \text{m}^2/\text{sec}$

(5-3)
conservation of
chemical mass

$$\rho \frac{DC}{Dt} + \nabla \cdot \underline{\underline{D}} \cdot \nabla pC + (c A_f - A_c) = 0$$

Part A-3 lets complete the story by sketching E conservation. (7)

B. The Energy Equation

$$G = \text{energy} = \left(\frac{1}{2} u_i^2 + e \right) \rho$$

$$= \left(\frac{1}{2} (u_x^2 + u_y^2 + u_z^2) + e \right) \rho$$

where e = internal energy
of molecules vibration + rotation

Energy

$$G = \frac{\text{Energy}}{V} = \frac{1}{2} u_i^2 + e$$

$$A_{\underline{E}} = \text{source of energy} = u_i \frac{\partial \tau_{ij}}{\partial x_j} + \rho u_i g_i$$

work rotation

$$J_{\underline{E}} = \dot{q} \quad (\text{difference heat flux})$$

$$\text{So } \rho \frac{D(\frac{1}{2} u_i^2 + e)}{Dt} + \nabla \cdot j - u_i \frac{\partial \tau_{ij}}{\partial x_j} - \rho u_i g_i = 0$$

rewriting terms in physical insight yields (Flow # 2):

$$\rho \frac{D(\frac{1}{2} u_i^2 + e)}{Dt} + \nabla \cdot j = \frac{\partial}{\partial x_j} (u_i \tau_{ij}) + \rho \nabla \cdot u - \rho$$

rate of change of kinetic potential, and internal energy
difference heat
work by surface force
work of volume expansion
work of viscous dissipation

If the flow speed u is small compared to
 the speed of sound, and ρ is substantially small,
 and we neglect changes in potential
 energy, and heat due to viscous stress and dilatational
 work are negligible, then:

$$\rho \frac{D\epsilon}{Dt} = -\nabla \cdot \mathbf{j}$$

Internal
 Energy available
 at constant
 pressure

but $\frac{D\epsilon}{Dt} = c_p \frac{DT}{Dt}$

and $\mathbf{j} = -k \nabla T$ (Fourier's law).

Then

$$\rho c_p \frac{DT}{Dt} = k \nabla^2 T \quad (\text{eq 4.88 in book})$$

(5.1)

$$\frac{DT}{Dt} = K \nabla^2 T$$

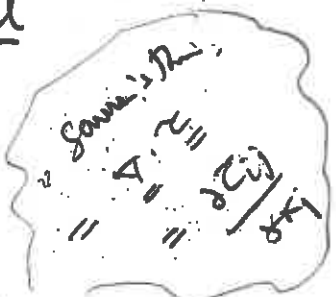
$K = \frac{k}{\rho c_p} = \text{Thermal diffusivity} \quad \frac{m^2}{s}$

Note heat flow relative to material matrix.

B. Conservation of fluid momentum

$$\underline{G} = \text{fluid momentum} = \rho \underline{u}$$

$$G = \frac{\text{fluid momentum}}{\text{kg fluid}} = \underline{u}$$



$$A_G = \frac{\text{unbalanced force}}{\text{unit volume}} = \int_{\partial V} \underline{\tau} \cdot d\underline{a} + \rho \underline{g}$$

mass, ∂V not V

$$J_G = \text{divergence of momentum} = 0$$

(could think of molecules divergence - but we will include ρm in A_G)

fluxes in by fluid / by surface

$$\rho \frac{D u_i}{D t} = \rho g_i + \frac{\partial \tau_{ij}}{\partial x_j} - \rho u_i A_f$$

if $A_f \neq 0$

we've used Gauss Thm.

(5-4)
conservation of momentum
(Cauchy's equation of motion)
Eq 4.15 in book

We will spend quite a bit of time

on this equation. We'll first illustrate conservation

of momentum in simple direct terms. Then derive the Navier-Stokes equation for viscous flow (incompressible, parabolic flow). Then specialize to Bernoulli equation. Then put in rotating frame, then consider vorticity.

II. Momentum Examples

(11)

A. Momentum Theorem for fluid Eulerian volume:

$$(1) \quad \rho \frac{D u_i}{D t} = \rho g_i + \frac{\partial \tau_{ij}}{\partial x_j}$$

Cauchy Equation
(Conservation of Momentum)

$$(2) \quad \rho \frac{\partial u_i}{\partial t} + \rho u_i \frac{\partial u_i}{\partial x_j} = \rho g_i + \frac{\partial \tau_{ij}}{\partial x_j}$$

add u_i times conservation of mass, and integrate over volume

$$(3) \quad + \quad u_i \left(\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x_j} (\rho u_j) \right) = 0$$

Conservation
(Continuity)

$$(4) \quad \int_V \left(\frac{\partial \rho u_i}{\partial t} + \frac{\partial \rho u_i u_j}{\partial x_j} - \rho g_i + \frac{\partial \tau_{ij}}{\partial x_j} \right) dV$$

" " " "

$$\frac{\partial}{\partial t} \int_V \rho u_i dV \quad \int_A \rho u_i u_j dA_j \quad \int_A \tau_{ij} dA_j$$

" " "

$$\boxed{\frac{\partial M_i}{\partial t} + \dot{M}_{out} = \bar{F}_{bi} + F_S}$$

body force

(5) Newton's Law

$$\underline{F} = \frac{dM}{dt} + \dot{M}_{out}$$

Force on volume = change in total momentum

Momentum theorem for fluid (Eulerian) volume
(3 indep. components)

(6)

$$\underline{T} = \frac{dH}{dt} + \dot{H}_{out}$$

Torque - just kind of momentum, $\underline{H} = \underline{r} \times \underline{u} dm$

B. Example - Wind force on a tank

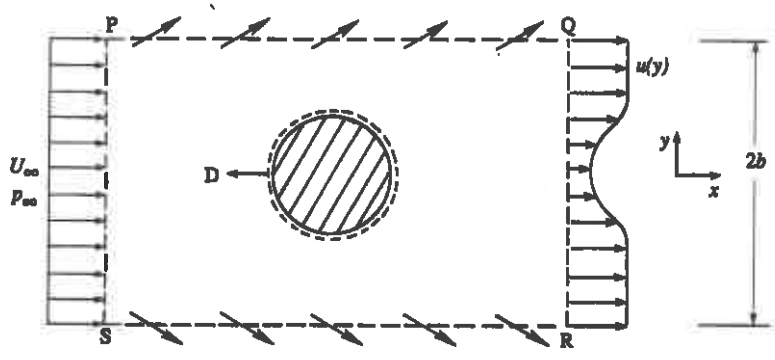


Figure 4.8 Momentum balance of flow over a body (Example 4.1).

$$\underline{F} = \frac{dM}{dt} + \dot{m}_{out}$$

boundaries distant enough that $p \sim p_{atm}$
 QR distant enough that streamlines \parallel and $v \sim 0$

steady state

conservation of momentum (steady state)

$$\therefore \underline{F}_x = \dot{m}_x^{out}$$

$$\dot{m}_x^{PS} = \int_{-b}^b U_{\infty} (\rho U_{\infty} dy) = -2b \rho U_{\infty}^2$$

$$\dot{m}_x^{QR} = \int_{-b}^b u (\rho u dy) = \rho \int_{-b}^b u^2 dy$$

Conservation of mass (flux out sides must equal flux diff):

conservation mass

$$\dot{m}^{PS} + \dot{m}^{SR} = \rho \int_{-b}^b (U_{\infty} - u) dy$$

$$\dot{m}_x^{PS} + \dot{m}_x^{SR} = \rho U_{\infty} \int_{-b}^b (U_{\infty} - u) dy$$

$$\dot{M}_x^{PS} + \dot{M}_x^{QR} + (\dot{M}_x^{PS} + \dot{M}_x^{SR}) = -2b \rho U_{\infty}^2 + \rho \int_{-b}^b u^2 dy + \rho U_{\infty} \int_{-b}^b (U_{\infty} - u) dy$$

$$\underline{F}_x = \rho \int_{-b}^b u (U_{\infty} - u) dy$$

Force on tank

Momentum diff downward of Tank

① note how we bulk conservation at conservation

② Logic - flux out sides!

B. Tank example by contour Integration

so back to (4) and consider steady state of rigid tank:

$$(7) \quad \int_V \frac{\partial}{\partial x_i} (\rho u_i u_j - \tau_{ij}) = 0$$

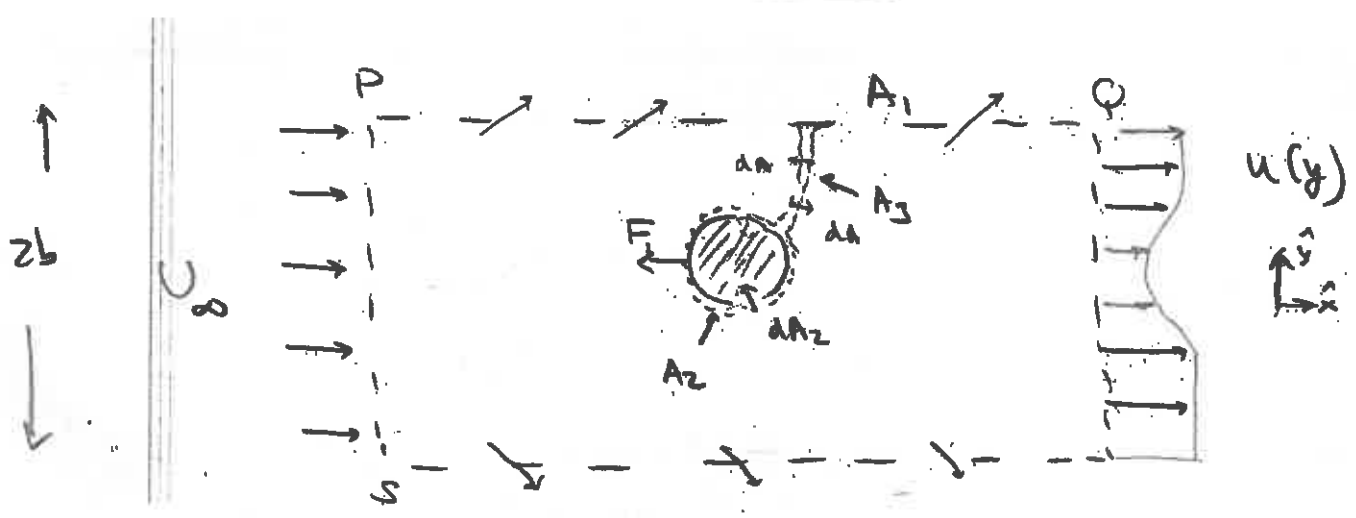
by Stokes theorem

$$(8) \quad \int_A (\rho \underline{u} \underline{u} - \underline{\tau}) \cdot d\underline{A} = 0$$

conservation of momentum

$$(9) \quad \int_A \rho \underline{u} \cdot d\underline{A} = 0$$

conservation of mass



$$0 = \int_{A_2} (\rho \underline{u} \cdot \underline{u} - \underline{\tau}) \cdot d\underline{A}_2 + \int_{A_3} (\rho \underline{u} \cdot \underline{u} - \underline{\tau}) \cdot d\underline{A}_3 + \int_{A_1} (\rho \underline{u} \cdot \underline{u} - \underline{\tau}) \cdot d\underline{A}_1$$

" $\int_{A_2} \underline{\tau} \cdot d\underline{A}_2$ "
 force on F_x

" $\int_{A_3} \underline{\tau} \cdot d\underline{A}_3$ "
 no volume enclosed $\rightarrow 0$

" $\int_{A_1} \underline{\tau} \cdot d\underline{A}_1$ "
 can't flow through wall $\rightarrow 0$

Examine last term:

$$F_{A_i} = - \int_{A_i} (\rho u u - \tau) \cdot dA_i$$

$\tau = \text{const pressure} - \text{cancel out around circuit}$

$$= - \int_{A_i} \rho u u \cdot A_i$$

A_i is outward pointing normal so

$$\int_{A_i} \rho u u \cdot dA_i = - \int_{A_i} \rho u^2 dy$$

$$\int_{A_i} \rho u u \cdot dA_i = \rho u_\infty \int_{y=-b}^b (u_\infty - u(y)) dy$$

momentum at side deficit momentum from flow out side out side

$$\int_{A_i} \rho u u \cdot dA = \rho \int_{y=-b}^b u^2(y) dy$$

$$\rho \int_{y=-b}^b (u_\infty u - u^2) dy$$

$$\therefore F_B = - \rho \int_{y=-b}^b u (u_\infty - u) dy$$

Momentum flux leaving system

by conservation of mass this is fluid exiting the side

Since it has a velocity check U_∞ to momentum is an inbuilt

Force on object in flow field can be measured by measuring y-velocity across a transect.

b. Sprinkler Example

Sprinkler example is beautiful example of method

flow of momentum

$T = \text{torque} = r \times F = Fr$

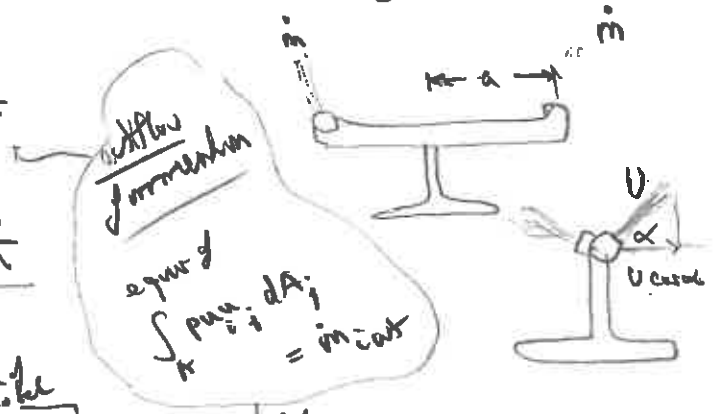
$H = \text{angular momentum} = \int \underline{r} \times \underline{u} \, dm$

$T = \frac{dH}{dt} + \dot{H}_{out}$

steady state

$\therefore T = \dot{H}_{out}$

transverse change in angular momentum



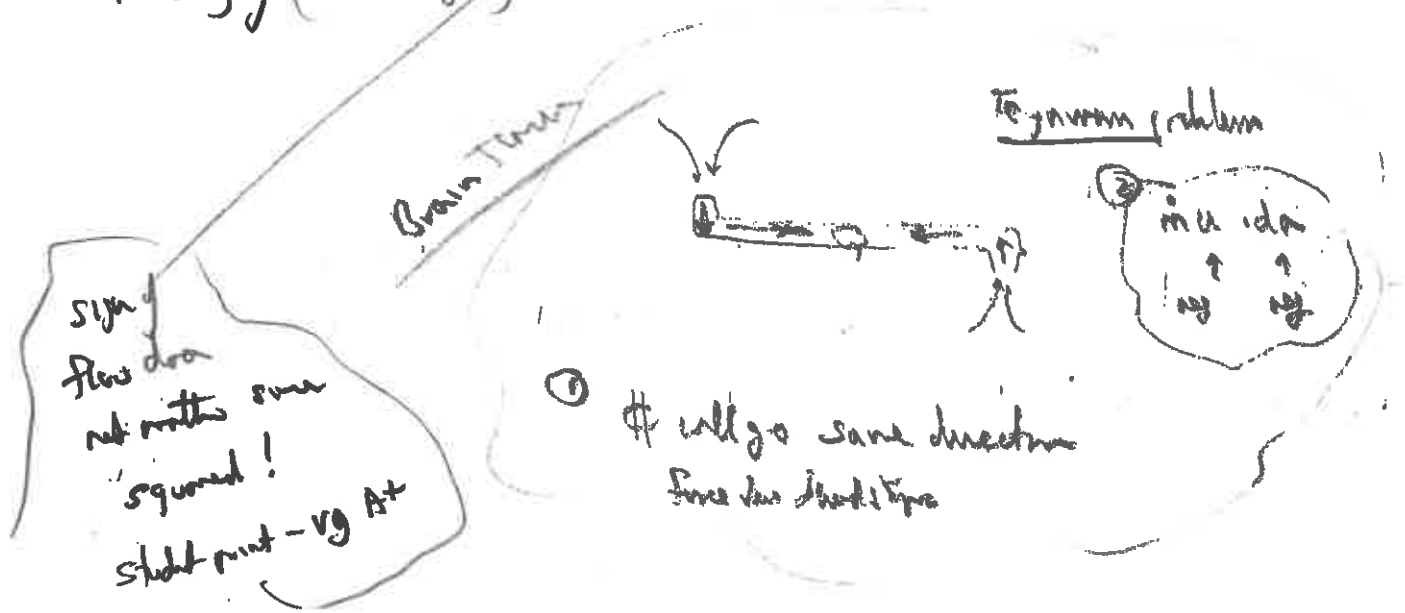
$\dot{m} = \rho A U = \text{kg/sec}$

$\dot{H}_z = [(\dot{m} U \cos \alpha) a]^2$

$T = 2 a \rho A U^2 \cos \alpha$

momentum flux leaving system (= force at two ends)

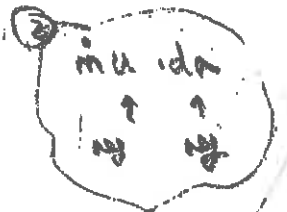
At steady state F must balance momentum flux. This is the meaning of (5-4 example).



sign of flow does not matter since squared!
 should not - v dy A dx

① it will go same direction force has that type

Tension problem



Flow 9 - Navier Stokes derivation

uniform flow reproduces so

ALL PHYSICS
MATHS

Students
misread volume
followed by mistake

$$\frac{D}{Dt} \int_V \rho dV = \int_V A \left[\frac{D\rho}{Dt} \right] dV$$

(Control volume = V)

or $\frac{D\rho}{Dt}$

COLS IN
ELASTIC
CURVATURE

Now in spatial coordinates the density / cons decreases if
the volume expands. By the generalized Leibniz rule

$\nabla \cdot \mathbf{u}$

ALL PHYSICS
MATHS

$$\frac{D}{Dt} \int_V \rho dV = \int_V \frac{\partial \rho}{\partial t} + \int_A \rho \mathbf{u} \cdot d\mathbf{A} = 0$$

By the Gauss Theorem (applied to $\rho \mathbf{u}$ or \mathbf{V})

$$\int_V \left(\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) \right) dV = 0$$

misread
divergence

Then

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) = 0$$

(Equation 2.5)

$$\frac{D\rho}{Dt} + \rho \nabla \cdot \mathbf{u} = 0$$

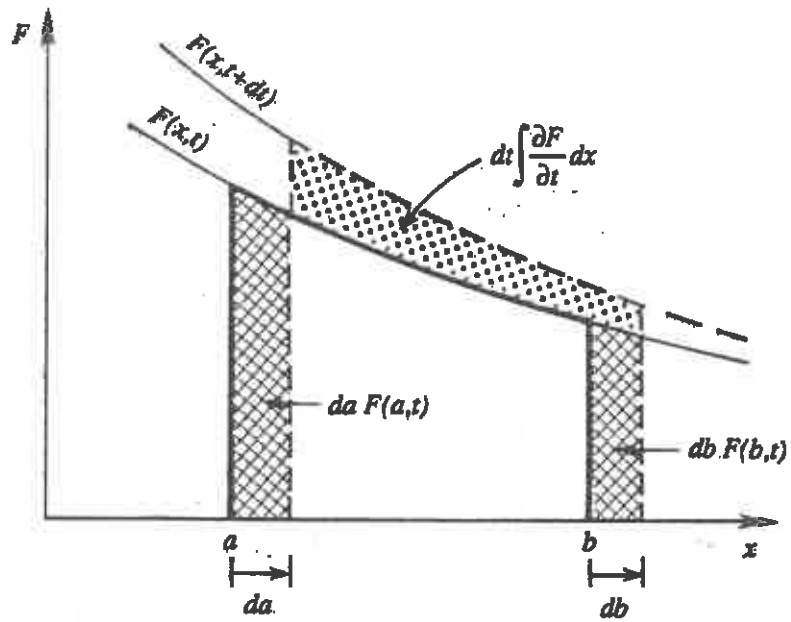


Figure 4.1 Graphical illustration of Leibniz's theorem.

$$\frac{d}{dt} \int_{V(t)} F(\mathbf{x}, t) dV = \int_{V(t)} \frac{\partial F}{\partial t} dV + \int_{A(t)} \mathbf{dA} \cdot \mathbf{u}_A F, \quad (4.3)$$

Gloss #2 Lecture 6

$$\rho \frac{D}{Dt} \left(\frac{1}{2} u_i^2 + e \right) + \nabla \cdot j - u_i \frac{\partial \tau_{ij}}{\partial x_j} - \rho u_i g_i = 0$$

$$\frac{\partial}{\partial x_j} (u_i \tau_{ij}) - \tau_{ij} e_{ij}$$

work by surface forces

$$\frac{D}{Dt} \rho g z = \rho \frac{\partial z}{\partial t} + \rho g u_i \delta_{iz}$$

solen (not δ_i)

Hauver + duken $\tau_{ij} = -p \delta_{ij} + 2\mu e_{ij} - \frac{2}{3} \mu (\nabla \cdot u) \delta_{ij}$

$$\tau_{ij} e_{ij} = -p \nabla \cdot u + 2\mu e_{ij} e_{ij} - \frac{2}{3} \mu (\nabla \cdot u)^2$$

viscous deformation vol

$$2\mu \left[e_{ij} - \frac{1}{3} (\nabla \cdot u) \delta_{ij} \right]^2$$

rate of viscous dissipation

Can show this is true by evaluating $[]^2 = \frac{1}{9} (\nabla \cdot u)^2 \delta_{ij} \delta_{ij} + e_{ij} e_{ij} - \frac{2}{3} (\nabla \cdot u)^2 = e_{ij} e_{ij} - \frac{1}{3} (\nabla \cdot u)^2$

So collecting terms:

$$\rho \frac{D}{Dt} \left(\frac{1}{2} u_i^2 + g z + e \right) + \nabla \cdot j = \frac{\partial}{\partial x_j} (u_i \tau_{ij}) + p \nabla \cdot u - \phi$$

rate of change of kinetic, potential and internal energy

efflux of heat

work by surface forces

Vol expansion work

viscous dissipation rate