

Lecture 8

Quick update from last time:

(1) Navier-Stokes Equations

(2) Bernoulli's equation #2

plus energy Bernoulli addit

EINSTEIN

Essays in Science



physico-chemical factors (decomposition of the ground). We must concentrate our attention on the circumstances which affect the steepness with which the velocity falls at the wall.

In both cases the asymmetry in relation to the fall in velocity in question is indirectly due to the occurrence of a circular motion to which we will next direct our attention. I begin with a little experiment which anybody can easily repeat.

Imagine a flat-bottomed cup full of tea. At the bottom there are some tea leaves, which stay there because they are rather heavier than the liquid they have displaced. If the liquid is made to rotate by a spoon, the leaves will soon collect in the center of the bottom of the cup. The explanation of this phenomenon is as follows:—The rotation of the liquid causes a centrifugal force to act on it. This in itself would give rise to no change in the flow of the liquid if the latter rotated like a solid body. But in the neighborhood of the walls of the cup the liquid is restrained by friction, so that the angular velocity with which it circulates is less there than in other places near the center. In particular, the angular velocity of circulation, and therefore the centrifugal force, will be smaller near the bottom than higher up. The result of this will be a circular movement of the liquid of the type illustrated in fig. 1. which goes on increasing until, under the influence of ground friction, it becomes stationary. The tea leaves are swept into the center by the circular movement and act as proof of its existence.

THE CAUSE OF THE FORMATION OF MEANDERS IN THE COURSES OF RIVERS AND OF THE SO-CALLED BEER'S LAW

It is common knowledge that streams tend to curve in serpentine shapes instead of following the line of the maximum declivity of the ground. It is also well known to geographers that the rivers of the northern hemisphere tend to erode chiefly on the right side. The rivers of the southern hemisphere behave in the opposite manner (Beer's law). Many attempts have been made to explain this phenomenon, and I am not sure whether anything I say in the following pages will be new to the expert; some of the relevant considerations are in any case known. Nevertheless, having found nobody who thoroughly understood the elementary principles involved, I think it is proper for me to give the following short qualitative exposition of them.

First of all, it is clear that the erosion must be stronger the greater the velocity of the current where it touches the bank in question, or the more steeply it falls to zero at any particular point of the confining wall. This is equally true under all circumstances, whether the erosion depends on mechanical or on

The same sort of thing happens with a curving stream (fig. 2). At every section of its course, where it is bent, a centrifugal force operates in the direction of the outside of the curve (from A to B). This force is less near the bottom, where the speed of the current is reduced by friction, than higher above the bottom. This causes a circular movement of the kind illustrated in the diagram. Even where there is

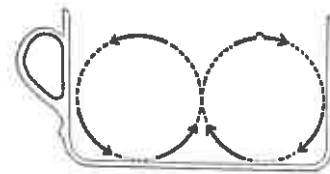


FIG. 1.

no bend in the river, a circular movement of the kind shown in fig. 2 will still take place, if only on a small scale and as a result of the earth's rotation. The latter produces a Coriolis-force, acting transversely to the direction of the current, whose right-hand horizontal component amounts to $2v\Omega \sin \phi$ per unit of mass of the liquid, where v is the velocity of the current, Ω the speed of the earth's rotation, and ϕ the geographical latitude. As ground friction causes a diminution of this force towards the bottom, this force also gives rise to a circular movement of the type indicated in fig. 2.

After this preliminary discussion we come back to

Navier-Stokes addition

(3)

$$\text{Substituting } \sigma_{ij} = 2\mu e_{ij} + \lambda e_{mm} \delta_{ij}$$

into

$$\tau_{ij} = p_n \delta_{ij} + \sigma_{ij}$$

with Stokes assumption that $\lambda + \frac{2}{3}\mu = 0$,

gives

$$(5-6) \quad \tau_{ij} = - \left\{ p_n + \frac{2}{3}\mu \nabla \cdot \underline{u} \right\} \delta_{ij} + 2\mu e_{ij}$$

Substituting into

Cauchy eqn

$$\rho \frac{D u_i}{D t} = \rho g_i + \frac{\partial \tau_{ij}}{\partial x_j}$$

$$2\mu \partial_j (e_{ij} = \frac{1}{2}(\partial_i u_j + \partial_j u_i))$$

$$\mu (\partial^2 u_i + \partial_i \nabla \cdot \underline{u})$$

$$\mu \nabla^2 \underline{u} + \mu \nabla \nabla \cdot \underline{u}$$

yields:

(5-8)
Navier
Stokes equation
(incompressible)

$$\rho \frac{D \underline{u}}{D t} = - \nabla p_n + \rho \underline{g} + \mu \nabla^2 \underline{u} + \frac{1}{3} \mu \nabla \nabla \cdot \underline{u}$$

if flow is far from boundaries, viscous effect $\rightarrow 0$

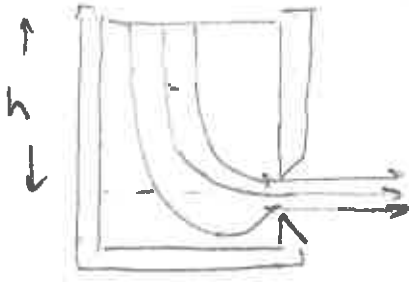
and

Euler
equation

(5-9)

$$\rho \frac{D \underline{u}}{D t} = - \nabla p + \rho \underline{g}$$

(2) Orifice in a Tank



$B =$ constant along streamlines

$$= \frac{q^2}{2} + \frac{p}{\rho} + gz$$

$$= \frac{p_{atm}}{\rho} + gh \quad \text{at top}$$

$$= \frac{p_{atm}}{\rho} + \frac{u^2}{2} \quad \text{at jet}$$

because
same p_{atm}
or reference

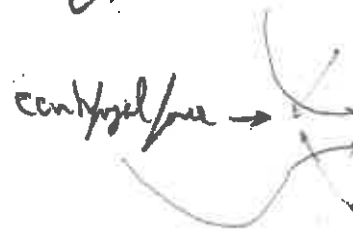
$$\therefore u = \sqrt{2gh}$$

$$\dot{m} = \text{mass flux out} = \rho A_c u$$

$$\dot{m} = \rho A_c \sqrt{2gh}$$

note centrifugal force of converging streamlines

$$\text{coefficient } A_c^{eff} \approx 0.62 A_c$$



→ [Also note Energy Derivation - total energy
 $h + \frac{u^2}{2} + gz$ const along streamline]

Coriolis + centrifugal → Navier-Stokes
 vorticity
 "Euler"
 Lecture 8: Rotating frames

Last time looked at
 the many forms of the
 conservation of momentum &

①

The final issue we need to consider (for much of the earth, but not the mantle) is the earth's rotation. Rotation gives rise to the Coriolis force, which as anyone who has tried a rotating platform at a carnival knows, is tricky a force to be reckoned with. Since our reference is the rotating surface of the earth, the Coriolis force for oceanography and atmospheric dynamics is an important, in fact dominant, control.

The book does a good job of describing how the Navier-Stokes equation in a rotating frame changes from:

Navier-Stokes
 (8-1)

$$\rho \frac{D\mathbf{u}}{Dt} = -\nabla p + \rho \mathbf{g} + \mu \nabla^2 \mathbf{u}$$

where \mathbf{u} is the velocity of a material element in the fixed frame, to



(8-2) (454 book)

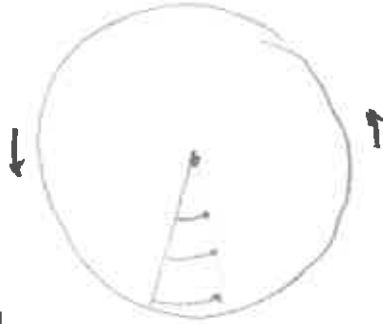
$$\rho \frac{D\mathbf{u}}{Dt} + \underbrace{2\Omega \times \mathbf{u}}_{\text{Coriolis accel}} - \underbrace{\Omega^2 \mathbf{R}}_{\text{centrifugal accel}} = -\nabla p + \rho \mathbf{g} + \mu \nabla^2 \mathbf{u}$$

where \mathbf{u} is the velocity of a material element in the rotating frame. The material frame physics is the same in both frames, but acceleration terms

arise because \underline{u} is now measured in the rotating frame.

Consider a rotating platform

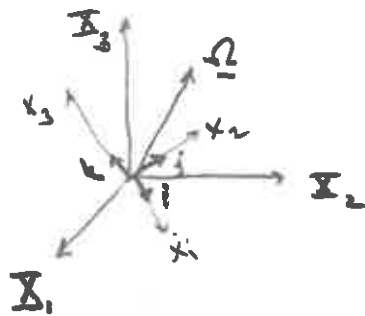
- As you walk out you must accelerate to "walk the straight line" in the rotating frame



- your acceleration is perceived as a Coriolis force!

- We add the force to the Navier Stokes equations so it applies in the rotating frame

Consider a point \underline{P} in fixed + rotating frames

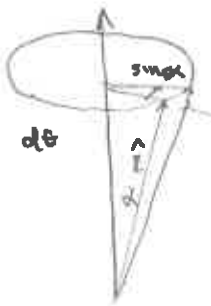


$$\underline{P}_R = P_{1R} \hat{i} + P_{2R} \hat{j} + P_{3R} \hat{k}$$

$$\left(\frac{d\underline{P}_R}{dt} \right)_F = \frac{d}{dt} (P_1 \hat{i} + P_2 \hat{j} + P_3 \hat{k})$$

$$= \hat{i} \frac{dP_1}{dt} + \hat{j} \frac{dP_2}{dt} + \hat{k} \frac{dP_3}{dt}$$

$$+ P_1 \frac{d\hat{i}}{dt} + P_2 \frac{d\hat{j}}{dt} + P_3 \frac{d\hat{k}}{dt}$$



$$\frac{d\hat{i}}{dt} = \sin \alpha \frac{d\theta}{dt}$$

$$\underline{\Omega} \times \hat{i}$$

$$= \frac{d\underline{P}}{dt} + \underline{\Omega} \times \underline{P}$$

(2-5)

$$\boxed{\left(\frac{d\underline{P}}{dt} \right)_F = \left(\frac{d\underline{P}}{dt} \right)_R + \underline{\Omega} \times \underline{P}}$$

Apply to $\underline{P} = \underline{r}$

$$\underline{u}_F = \underline{u}_R + \underline{\Omega} \times \underline{r}$$

Apply to u_F

$$\left(\frac{du_F}{dt} \right)_F = \left(\frac{du_R}{dt} \right)_R + \underline{\Omega} \times \underline{u}_F$$

$$= \frac{d}{dt} (\underline{u}_R + \underline{\Omega} \times \underline{r})_R + \underline{\Omega} \times (\underline{u}_R + \underline{\Omega} \times \underline{r})$$

$$= \left(\frac{du_R}{dt} \right)_R + 2 \underline{\Omega} \times \underline{u}_R + \underline{\Omega} \times (\underline{\Omega} \times \underline{r})$$

← axis rotation

$$\downarrow$$

$$- (\underline{\Omega} \cdot \underline{\Omega}) \underline{r} + \left(\frac{\underline{\Omega} \cdot \underline{r}}{R} \right) \underline{\Omega}$$

0

$$\underline{A} \times (\underline{B} \times \underline{C}) = (\underline{A} \cdot \underline{C}) \underline{B} - (\underline{A} \cdot \underline{B}) \underline{C}$$

$$\left(\frac{du_F}{dt} \right)_F = \left(\frac{du_R}{dt} \right)_R + 2 \underline{\Omega} \times \underline{u}_R - \underline{\Omega}^2 \underline{r}$$

$$a_F = a_R + 2 \underline{\Omega} \times \underline{u}_R - \underline{\Omega}^2 \underline{r}$$

$\nu = \mu / \rho$

Thus the Navier Stokes equation in the Fixed Frame

$$\left(\frac{Du}{Dt} \right)_F = -\frac{1}{\rho} \nabla p + \underline{g} + \nu \nabla^2 \underline{u}$$

becomes in rotatg frame

effective gravity force

in inertial frame

(2.4)

$$\left(\frac{Du}{Dt} \right)_R = -\frac{1}{\rho} \nabla p + \left(\underline{g} + \underline{\Omega}^2 \underline{r} \right) + \nu \nabla^2 \underline{u} - 2 \underline{\Omega} \times \underline{u}$$

Coriolis acceleration is just that required for a projectile to follow the path we all know it should.

Consider standing on the pole of rotation and firing south.

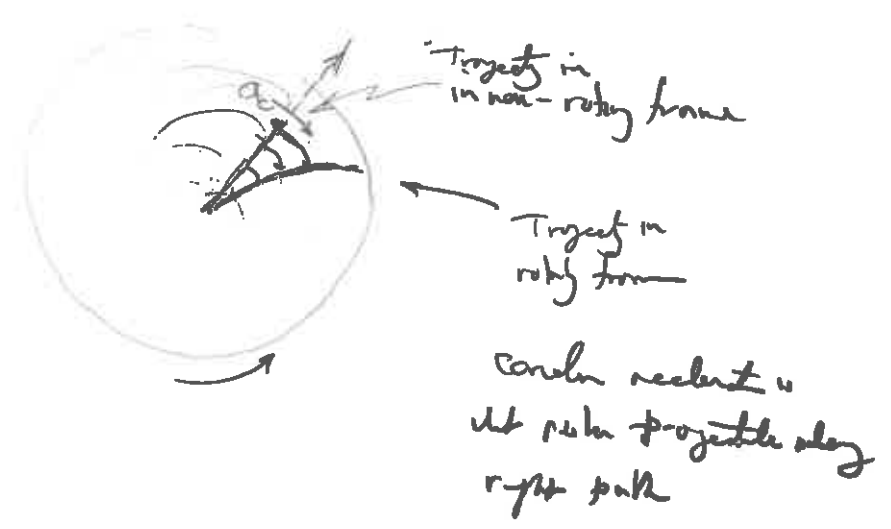
The projectile velocity u_r does not change. $\therefore x_s = ut$.

As projectile moves south rotation of earth \uparrow . $a_c = 2\Omega u_r$

So $v_\theta = a_c t$, $x_\theta = \int a_c t dt = \frac{1}{2} a_c t^2 = \Omega u t^2$

$\therefore \alpha = \frac{\Delta x_\theta}{\Delta x_s} = \Omega t = \text{just amt earth turned in } t.$

\therefore earth turns under projectile. "Acceleration" arises from fact $u_\theta \uparrow$ with distance south. Acceleration defines the "fictitious" force need to "push" the projectile along the proper path.



Vorticity

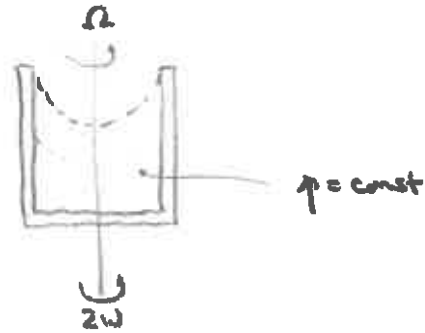
Incompressible
Navier Stokes

$$\frac{D\mathbf{u}}{Dt} = -\frac{1}{\rho} \nabla p + \mathbf{g} + \nu \nabla^2 \mathbf{u} - 2\mathbf{\Omega} \times \mathbf{u}$$

Euler

$$\frac{D\mathbf{u}}{Dt} = -\frac{1}{\rho} \nabla p + \mathbf{g} - 2\mathbf{\Omega} \times \mathbf{u}$$

Solid body rotation



From basic
physics



$$F = \frac{\rho u_0^2}{r}$$

$$\frac{\partial p}{\partial r} = -\rho g \frac{\partial z}{\partial r}$$

$$\frac{\rho u_0^2}{r} - \rho g \frac{\partial z}{\partial r} = 0$$

force balance

substitute $u_0 = \frac{1}{2} \omega r$

$$\int_{r_1}^{r_2} \frac{1}{4} \omega^2 r \, dr = \int_{z_1}^{z_2} g \, dz$$

$$\frac{1}{8} \frac{\omega^2}{g} (r_2^2 - r_1^2) = (z_2 - z_1)$$

Now from Euler equation

$$\frac{D\mathbf{u}}{Dt} = \frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} = -\frac{1}{\rho} \nabla p + \mathbf{g}$$

In polar coordinates (r, θ, z) with $u_r = 0, u_\theta = \frac{1}{2} \omega_0 r$

$$\begin{aligned} \frac{\partial u_r}{\partial t} + \mathbf{u} \cdot \nabla u_r &= -\frac{u_\theta^2}{r} = -\frac{1}{\rho} \frac{\partial p}{\partial r} - \nu \left(\nabla^2 u_r - \frac{u_r}{r^2} - \frac{2}{r^2} \frac{\partial u_\theta}{\partial \theta} \right) \\ \frac{\partial u_\theta}{\partial t} + \mathbf{u} \cdot \nabla u_\theta + \frac{u_r u_\theta}{r} &= \frac{1}{\rho r} \frac{\partial p}{\partial \theta} + \nu \left(\nabla^2 u_\theta + \frac{2}{r^2} \frac{\partial u_r}{\partial \theta} - \frac{u_\theta}{r} \right) \\ \frac{\partial u_z}{\partial t} + (\mathbf{u} \cdot \nabla) u_z &= -\frac{1}{\rho} \frac{\partial p}{\partial z} + \nu \nabla^2 u_z - g \end{aligned}$$

From p 732 $\sigma_{r\theta} = \mu \left[\frac{1}{r} \frac{\partial u_r}{\partial \theta} + r \frac{\partial}{\partial r} \left(\frac{u_\theta}{r} \right) \right] = 0$

and if solid body rotation then $u_\theta = \omega_0 r, u_r = 0, \sigma_{r\theta} = 0$

Irrrotational Vortex

$$u_\theta = \frac{\Gamma}{2\pi r}$$

$$u_\theta = \frac{C}{r} \quad \text{and} \quad u_r = 0$$

$$\omega = \nabla \times \underline{u}$$

$$\frac{\partial \sigma_{ij}}{\partial x_j} = -\mu (\nabla \times \omega)_i$$

$$= \frac{1}{r} \left(\frac{\partial}{\partial r} (r u_\theta) - \frac{\partial u_r}{\partial \theta} \right)$$

$$= \frac{1}{r} \left(\frac{\partial}{\partial r} \left(\frac{rC}{r} \right) \right) = 0$$

since $\omega = 0$ there is no viscous force. \therefore again inviscid Euler equation holds

$$-\frac{u_\theta^2}{r} = -\frac{1}{\rho} \frac{\partial p}{\partial r}$$

$$0 = -\frac{\partial p}{\partial z} - \rho g$$

So

$$dp = \frac{\rho \Gamma^2}{4\pi^2 r^3} dr - \rho g dz$$

Consider $\frac{1}{r}$ flow:



$$p_1 - p_2 = -\frac{\rho}{2} (u_{\theta 2}^2 - u_{\theta 1}^2) - \rho g (z_2 - z_1)$$

$$\frac{p_1}{\rho} + \frac{u_{\theta 1}^2}{2} + g z_1 = \frac{p_2}{\rho} + \frac{u_{\theta 2}^2}{2} + g z_2$$

Bernoulli equation valid between any two points (crossed flow just along streamlines)

$$z_2 - z_1 = \frac{\Gamma}{4\pi^2 g} \left(\frac{1}{r_1^2} - \frac{1}{r_2^2} \right)$$

Toilet drain

Kerim: Circulation Theorem

$$\frac{D\Gamma}{Dt} = 0$$

Circulation is conserved in a material element!

$$\frac{D\Gamma}{Dt} = \frac{D}{Dt} \int_C u_i dx_i$$

Circulation around a material contour does not change

$$= \int_C \frac{Du_i}{Dt} dx_i + \int_C u_i \frac{D}{Dt} dx_i$$

momentum equation

$$\frac{Du_i}{Dt} = -\frac{1}{\rho} \frac{\partial p}{\partial x_i} + g_i + \frac{1}{\rho} \partial_j \sigma_{ij}$$

$$\int_C \frac{Du_i}{Dt} dx_i = - \int_C \frac{1}{\rho} \frac{\partial p}{\partial x_i} dx_i + \int_C g_i dx_i + \int_C \frac{1}{\rho} \partial_j \sigma_{ij} dx_j$$

$$\frac{D\Gamma}{Dt} = \int_C \underline{g} \cdot d\underline{x} - \int_C \frac{dp}{\rho} + \int_C \frac{1}{\rho} (\underline{\nabla} \cdot \underline{\sigma}) \cdot d\underline{x} + \int_C u_i \frac{D}{Dt} dx_i$$

$$\underline{g} = -\underline{\nabla} \Pi$$
$$\Pi_A - \Pi_B$$
$$\text{if } A=B \Rightarrow 0$$

barotropic

$$p^{-1} \equiv \frac{dp}{\rho}$$
$$P_B - P_A \Rightarrow 0$$

||
σ of viscous stress can be neglected

$$d\underline{u} = \frac{D}{Dt} d\underline{x}$$
$$\int_C d\left(\frac{1}{2} u_i^2\right)$$

since $u_i = \text{same at start + end} \Rightarrow 0$

Create or destroy vorticity

Non-conservative body forces

Non-barotropic pressure-ρ relation

Viscous stresses

Implications

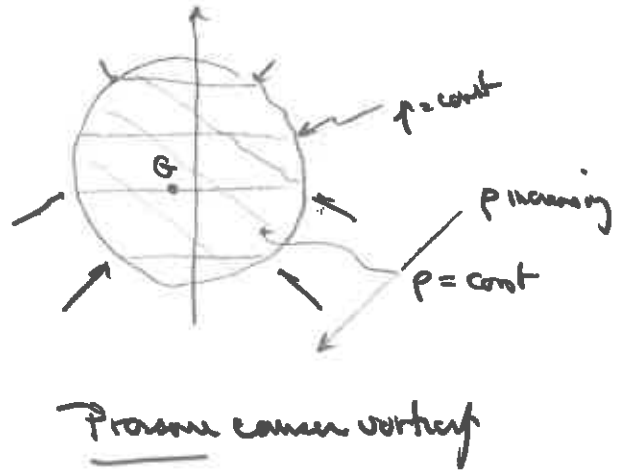
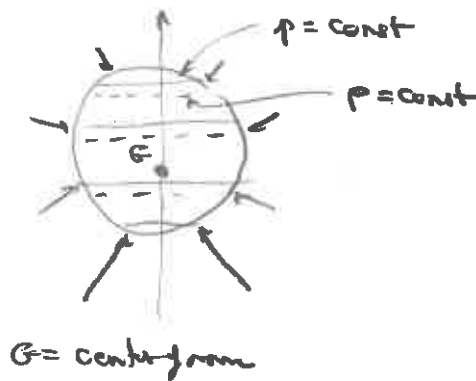
- (1) Circulation is preserved
- (2) Viscous effects cause diffusion of vorticity in + out of circuit
- (3) Conservative body forces act through center of mass + do not produce vorticity

(4) Barotropic flow - inviscid

lines of const ρ
 parallel to lines of constant p
 so that ∇p acts at c of m

or
 $\rho(\rho)$ of
 (perfect gas $p/\rho r = \text{const}$)

not barotropic = baroclinic



Geophysical flows which are dominated by baroclinicity are full of vorticity

Henry - $\nabla_H p = 0$ necessary condition for static fluid
 $\nabla_H p \neq 0$ generates vorticity

Helmholtz Theorems

- (1) Vortex lines move with the fluid
- (2) Strength of vortex tube (its circulation) = const along length
- (3) Vortex tube cannot start or end (end at bdy or be closed loop - vortex ring)
- (4) strength of a vortex tube remains constant in time



vortex tube enclosing material particles

S on boundary - circulation around edge of $S = 0$

as tube moves S will move but $\oint \mathbf{v} \cdot d\mathbf{s}$ with fluid $\oint \mathbf{v} \cdot d\mathbf{s} = 0$

\therefore no vorticity out of tube surface

\therefore vorticity contained in tube!

Vorticity equation

$$\frac{D\underline{\omega}}{Dt} = (\underline{\omega} \cdot \nabla) \underline{u} + \nu \nabla^2 \underline{\omega}$$

$$= (\omega + 2\Omega) \cdot \nabla u + \frac{1}{\rho^2} \nabla \rho \times \nabla p + \nu \nabla^2 \omega$$

stretching + tilting

baroclinic secondary vorticity

diffusion

$$(\underline{\omega} \cdot \nabla) \underline{u} = \left[\underline{\omega} \cdot \left(i_s \frac{\partial}{\partial s} + i_n \frac{\partial}{\partial n} + i_m \frac{\partial}{\partial m} \right) \right] \underline{u} = \omega \frac{\partial u}{\partial s}$$

along a vortex line s

$\frac{\partial u}{\partial s}$ is stretching along vortex line

$\frac{\partial}{\partial n}, \frac{\partial}{\partial m}$ represent turning or tilting of vortex line about m and n axes

$$\frac{D\underline{\omega}}{Dt} = \omega \frac{\partial u}{\partial s}$$

Stretching plays crucial role in dynamics of turbulent and geophysical flows

$$\frac{D\underline{\omega}_s}{Dt} = \omega \frac{\partial \omega_s}{\partial s}$$

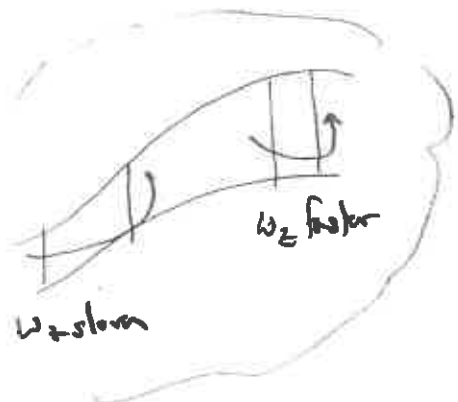
vorticity changes as stretch vortex lines. stretches + angular speed!

$$\frac{D\underline{\omega}_n}{Dt} = \omega \frac{\partial \omega_n}{\partial s}$$

tilting shifts vorticity into other components

$$\frac{D\underline{\omega}_m}{Dt} = \omega \frac{\partial \omega_m}{\partial s}$$

$$2(\underline{\Omega} \cdot \nabla) \underline{u}$$



if we orient z axis with $\underline{\Omega}$

$$\frac{D\underline{u}}{Dt} = 2 \underline{\Omega} \frac{\partial \underline{u}}{\partial t}$$

$$\frac{Dw_z}{Dt} = 2 \underline{\Omega} \frac{\partial w}{\partial t}$$

stretching in z increases w_z

Does any vertical fluid line contain "planetary vorticity" just trying to vertical fluid line is enough - vorticity tubes don't need to tilt

$$\frac{Dw_x}{Dt} = 2 \underline{\Omega} \frac{\partial w}{\partial t}$$

$$\frac{Dw_y}{Dt} = 2 \underline{\Omega} \frac{\partial w}{\partial t}$$

Tilting of vorticity line changes relative vorticity in x & y directions

Kevin's circulation Theorem in rotating frame

$$\frac{D \Gamma_a}{Dt} = 0$$

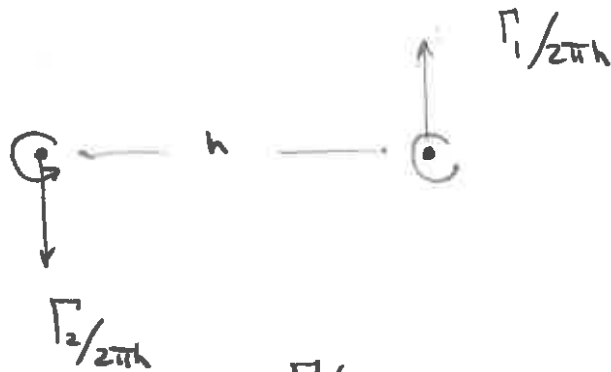
$$\Gamma_a = \int_A (\underline{u} + 2 \underline{\Omega}) \cdot d\underline{A} = \Gamma + 2 \int_A \underline{\Omega} \cdot d\underline{A}$$

$\underbrace{\hspace{10em}}$
planetary vorticity

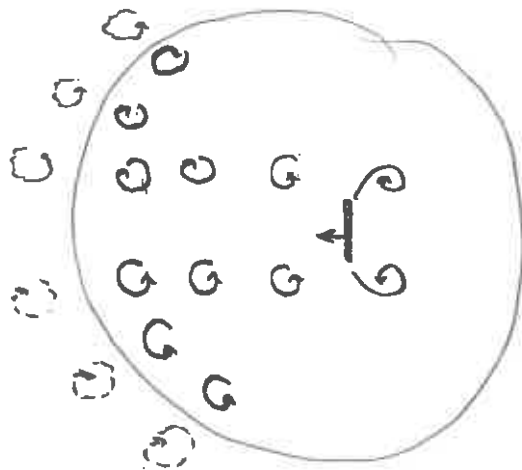
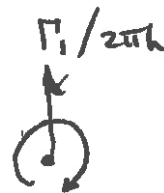
Interaction of Vortices -

vortices induce circulation at a distance ($\frac{1}{r}$) and interact

same sign rotate around each other



opposite sign translate



randomly generated vortices

Smoke Ring + vortex sheet

2 rings equal size + same rotation

- front slows - sucking of behind + ↑ in radius
- back speeds " of in front + ↓ in radius
- pattern $R\omega$ - yoyo

Vortex sheet - decaying of velocity - aircraft wing

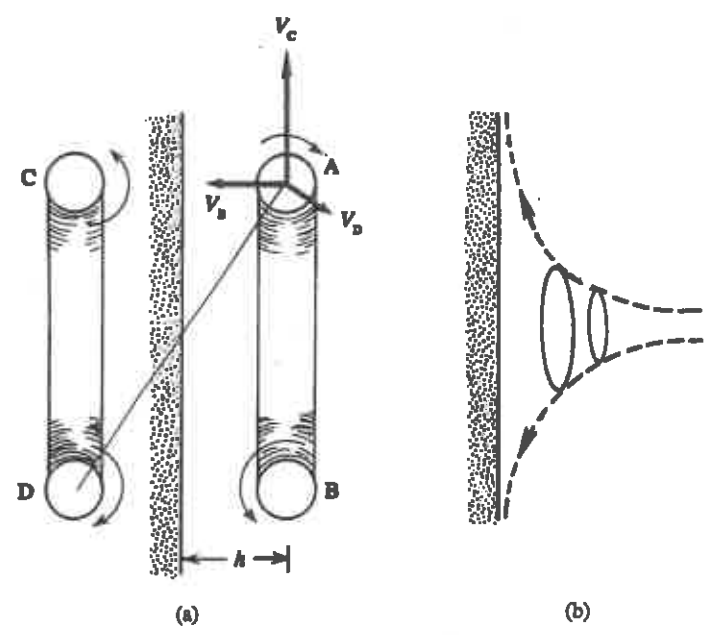


Figure 5.15 (a) Torus or doughnut-shaped vortex ring near a wall and its image. A section through the middle of the ring is shown. (b) Trajectory of vortex ring, showing that it widens while its translational velocity toward the wall decreases.

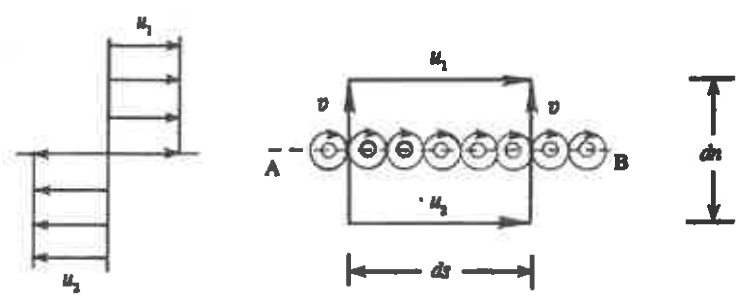


Figure 5.16 Vortex sheet.

Chapter 6 - Irrotational flow

Irrotational flow stay irrotational

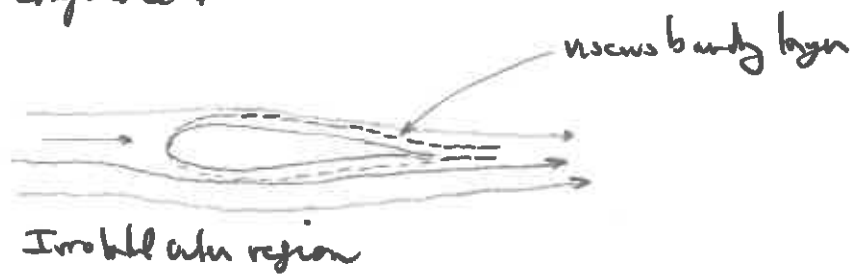
Rotated " " rotational

∴ Irrotational flow will become rotational at a body

÷ 2 regions - near body - "Inner region"

- far from body - "outer region"

Eddies congregate!



can use complex variables and law of superposition to compute ideal streamlines + vortex lines

streamline
velocity field

$$u = \frac{\partial \phi}{\partial x} = \frac{\partial \psi}{\partial y}$$

$$v = -\frac{\partial \phi}{\partial y} = \frac{\partial \psi}{\partial x}$$

$$\frac{\partial \phi}{\partial x} = \frac{\partial \psi}{\partial y}$$

$$\frac{\partial \phi}{\partial y} = -\frac{\partial \psi}{\partial x}$$

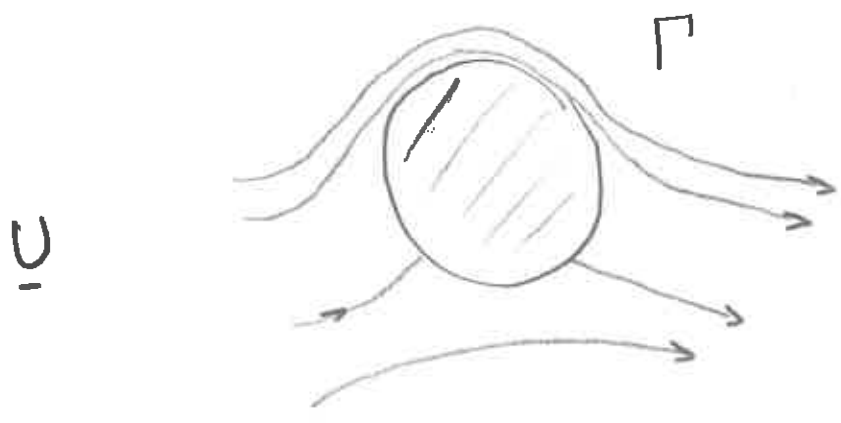
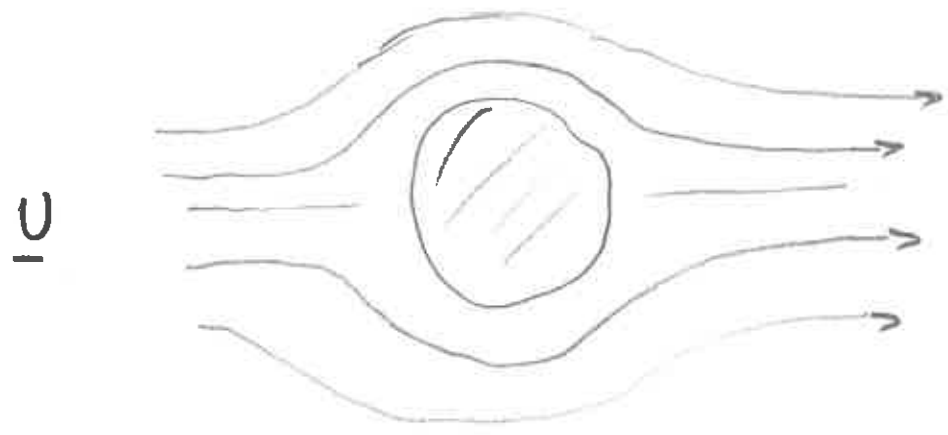
Cauchy-Riemann condition

Add vortex circulation to flow around a circular body

$$w = U \left(z + \frac{a^2}{z} \right) + \frac{i \Gamma}{2\pi} \ln \frac{z}{a}$$

$$\phi = U \left(r - \frac{a^2}{r} \right) \cos \theta + \frac{\Gamma}{2\pi} \ln \frac{r}{a}$$

$$z = x + iy = r e^{i\theta}$$



with fluid vortex added

Apply Bernoulli's principle to circuit around sphere

Kutta-Zhukovsky Lift Theorem

$$L = \rho U \Gamma$$

Lift

Lift is proportional to the circulation!

viscosity generates Γ but magnitude of Γ indep of visc + depends on U , shape + "attached" of the foil

Magnus effect \downarrow

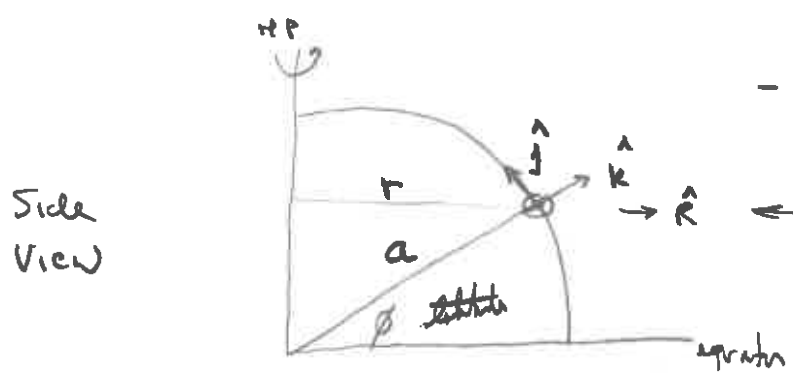
for circular body - must rotate to get lift.

soft ball diameter \rightarrow

delay of separation of eddies also imp't

March 8, 2016

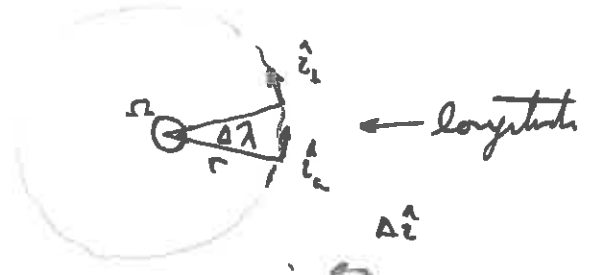
Mark Wysocki



- Metrology convention

← unit vector in r direction

Top view

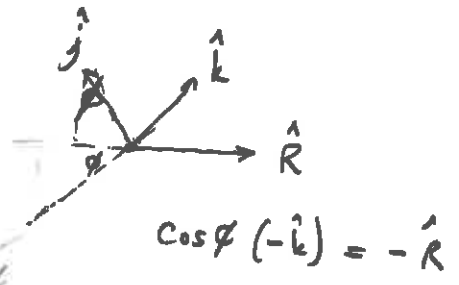


← Longitude

i-hat point east

$$\Delta \hat{i} = \hat{i} \Delta \lambda$$

$$\sin \phi \hat{j} = -\hat{R}$$



$$\cos \phi (-\hat{k}) = -\hat{R}$$

$$u \frac{D\hat{i}}{Dt}$$

i-hat = const - velocity u / time unit vector

$$u \left(\frac{\partial \hat{i}}{\partial t} + u \frac{\partial \hat{i}}{\partial x} + v \frac{\partial \hat{i}}{\partial y} + w \frac{\partial \hat{i}}{\partial z} \right)$$

$$u \frac{\Delta \hat{i}}{\Delta x} = -r \Delta \lambda$$

$$= -u \frac{r \Delta \lambda}{r a \Delta \lambda} = -\frac{u}{r} \hat{R}$$

$$\frac{D\hat{i}}{Dt} = \frac{u^2}{a} \tan \phi \hat{j} - \frac{u^2}{a} \hat{k}$$

$$\hat{R} = (-\sin \phi \hat{j} + \cos \phi \hat{k})$$

$$\frac{\partial \hat{i}}{\partial x} = \frac{1}{r} u (\sin \phi \hat{j} - \cos \phi \hat{k})$$

← = a cos phi

$$\frac{\partial \hat{i}}{\partial x} = \frac{u}{a} \tan \phi \hat{j} - \frac{u}{a} \hat{k}$$

$$= \frac{\partial \hat{i}}{\partial x} = \frac{u}{a \cos \phi} (\sin \phi \hat{j} - \cos \phi \hat{k})$$

Typical m on Synoptic scale

48 hr - W coast to E coast

$$U = 10 \text{ m s}^{-1} \quad \text{typical}$$

$$a = 637 \times 10^6 \text{ m}$$

$$\phi = 45^\circ$$

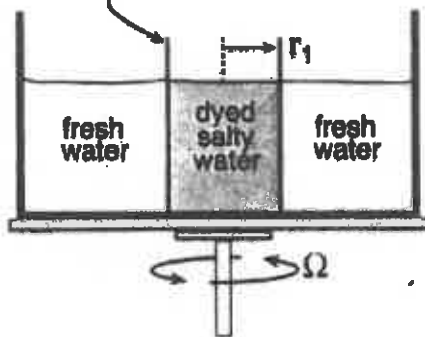
$$\begin{aligned} u \frac{D_i}{Dt} &= \frac{U}{a} \tan \phi j - \frac{U}{a} k^2 \\ &= 1.57 \times 10^{-6} \text{ m s}^{-2} - 1.5 \times 10^{-6} \text{ m s}^{-2} \end{aligned}$$

$\approx \phi$ in mid latitude
cancel!

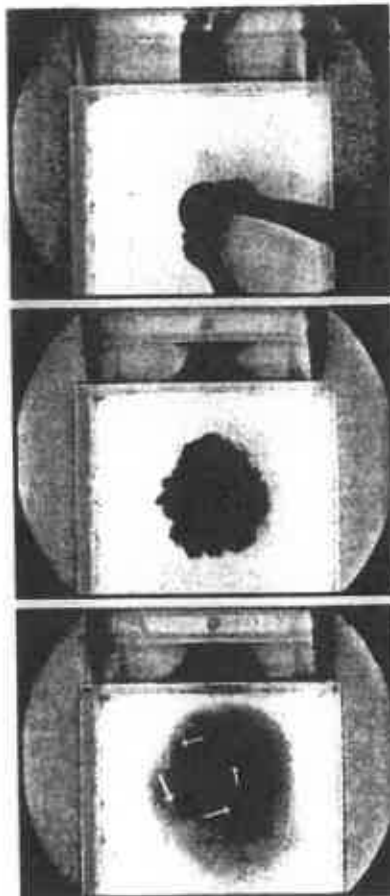
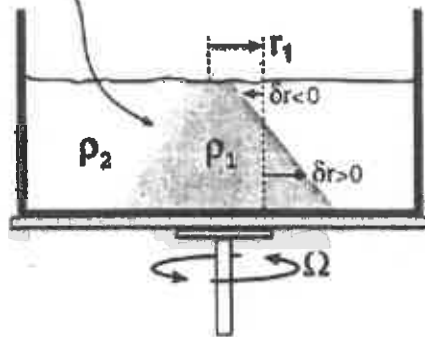
GCMs must take into account.

Geostrophic flow - ∇p balanced by Coriolis force

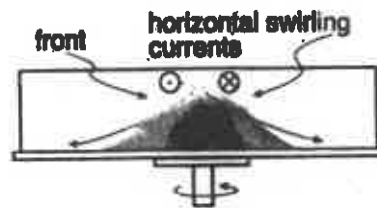
metal cylinder



frontal surface



mirror reflecting side view



salt water sinking under gravity and rotation

