# Modeling Quiescent Phase Transport of Air Bubbles Induced by Breaking Waves

BY FENGYAN SHI, JAMES T. KIRBY AND GANGFENG MA

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CENTER FOR APPLIED COASTAL RESEARCH

Ocean Engineering Laboratory University of Delaware Newark, Delaware 19716

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Fengyan Shi, James T. Kirby and Gangfeng Ma

Center for Applied Coastal Research, University of Delaware, USA.

#### Abstract

Simultaneous modeling of both the acoustic phase and quiescent phase of breaking wave-induced air bubbles involves a large range of length scales from microns to meters and time scales from milliseconds to seconds, and thus is computational unaffordable in a surfzone-scale computational domain. In this study, we use an air bubble entrainment formula in a two-fluid model to predict air bubble evolution in the quiescent phase in a breaking wave event. The breaking wave-induced air bubble entrainment is formulated by connecting the shear production at the air-water interface and the bubble number intensity with a certain bubble size spectra observed in laboratory experiments. A two-fluid model is developed based on the partial differential equations of the gas-liquid mixture phase and the continuum bubble phase, which has multiple size bubble groups representing a polydisperse bubble population. An enhanced 2-DV VOF (Volume of Fluid) model with a  $k - \epsilon$ turbulence closure is used to model the mixture phase. The bubble phase is governed by the advection-diffusion equations of the gas molar concentration and bubble intensity for groups of bubbles with different sizes. The

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model is used to simulate air bubble plumes measured in laboratory experiments. Numerical results indicate that, with an appropriate parameter in the air entrainment formula, the model is able to predict the main features of bubbly flows as evidenced by reasonable agreement with measured void fraction. Bubbles larger than an intermediate radius of O(1mm) make a major contribution to void fraction in the near-crest region. Smaller bubbles tend to penetrate deeper and stay longer in the water column, resulting in significant contribution to the cross-sectional area of the bubble cloud. An under-prediction of void fraction is found at the beginning of wave breaking when large air-pockets take place. The core region of high void fraction predicted by the model is dislocated due to use of the shear production in the algorithm for initial bubble entrainment. The study demonstrates a potential use of an entrainment formula in simulations of air bubble population in a surfzone-scale domain. It also reveals some difficulties in use of the two-fluid model for predicting large air pockets induced by wave breaking, and suggests that it may be necessary to use a gas-liquid two-phase model as the basic model framework for the mixture phase and to develop an algorithm to allow for transfer of discrete air pockets to the continuum bubble phase. A more theoretically justifiable air entrainment formulation should be developed. *Keywords:* air bubble, breaking wave, RANS model

#### 1 1. Introduction

The simulation of breaking wave-induced bubbly flows is a great challenge 2 due to the complexity of air entrainment and bubble evolution processes, and 3 to the range of spatial and temporal scales involved. According to previous 4 studies based on field or laboratory experiments (e.g., Thorpe, 1982; Gar-5 rett et al., 2000; Terrill et al., 2001; Deane and Stokes, 2002), the lifetime 6 of wave-generated bubbles can be categorized into two phases. The first 7 phase is called the acoustic phase, during which bubbles are entrained and 8 fragmented inside the breaking wave crest. The second phase happens after bubble creation processes cease and the newly formed bubbles evolve under 10 the influence of turbulent diffusion, advection, buoyant degassing, and disso-11 lution. Because this phase is acoustically quiescent, it is called the quiescent 12 phase. The duration of the acoustic phase is very short and the time scale 13 of bubble fragmentation is typically tens of milliseconds (Leighton, et al., 14 1994). Therefore, Direct Numerical Simulations (DNS) of the acoustic phase 15 require higher resolution in both time and space in order to capture the de-16 tails of the air entrainment process, making computations so expensive that 17 the main use of this kind of model will be limited to applications to studies 18 of bubble creation mechanisms. 19

Instead of a direct simulation of the air entrainment process, the use of an initial air entrainment formulation in modeling of bubbly flows was reported recently (Moraga et al., 2008; Shi et al., 2008). The idea was to prescribe air bubbles entrained during the acoustic phase in a two-phase model using a <sup>24</sup> bubble entrainment formulation. The model fed with the initially entrained
<sup>25</sup> bubbles simulates bubble plumes and requires much less spatial and temporal
<sup>26</sup> resolution than needed to capture the air entrainment process. The initial
<sup>27</sup> bubble number density and bubble size distribution were formulated based
<sup>28</sup> on theoretical and observational studies.

In a simulation of air bubbles entrained by naval surface ships, Moraga et 29 al. (2008) presented a sub-grid model that detects the location of the air bub-30 ble entrainment region. The localized region of high void fraction is bounded 31 by the surface at which the downward liquid velocity reaches a certain value 32 (0.22 m/s was used in Moraga et al.'s application). The initial bubble size 33 distribution in the localized region follows the bubble size spectrum mea-34 sured by Deane and Stokes (2002) who suggested that, at the beginning of 35 the quiescent phase, the size spectrum follows a certain power-law scaling 36 with bubble radius. Deane and Stokes (2002) found two distinct mechanisms 37 controlling the size distribution, depending on bubble size. For bubbles larger 38 than the Hinze scale (about 1 mm in Deane and Stokes (2002)), turbulent 39 fragmentation determines bubble size distribution, resulting in a bubble den-40 sity proportional to  $r_b^{-10/3}$ , where  $r_b$  is bubble radius. Bubbles smaller than 41 the Hinze scale are generated by jet and drop impact on wave face, with a 42 bubble density proportional to  $r_b^{-3/2}$ . The Hinze scale, which separates the 43 two processes, is the scale where turbulent fragmentation ceases, and is re-44 lated to the turbulent dissipation rate and the surface tension. A parallel 45 study was carried out by Shi et al. (2008), who used the same strategy to 46

avoid modeling of the bubble entrainment process, but applied a different 47 air entrainment formulation for breaking wave-induced air bubbles. The ini-48 tial air bubble entrainment is formulated by connecting the flow shear stress 40 at air-water interface and the bubble number intensity with the bubble size 50 spectra as observed by Deane and Stokes (2002). The model was used to 51 simulate wave transformation, breaking, and bubble generation and evolu-52 tion processes over a barred beach in the Large Wave Flume at Oregon State 53 University. Although there were no data for bubble quantities for compari-54 son, the model results showed that the evolution pattern of void fraction at 55 the water surface is consistent with bubble foam signatures sensed by video 56 systems during the laboratory experiments. The study showed the potential 57 to use an air entrainment formulation in modeling of air bubbles inside the 58 surfzone. 59

Models based on the volume or ensemble averaged two-fluid approach 60 seem best suited for practical use in modeling air bubbles in large-scale sys-61 tems such as breaking wave-induced bubbles in coastal water because of their 62 efficiency (Sokolichin et al., 2004). Carrica et al. (1998) reported a multi-63 phase model for simulating bubbly two-phase flow around a surface ship. The 64 bubble phase is modeled using the integrated Boltzmann transport equation 65 for the bubble size distribution function (Guido-Lavalle et al., 1994) and the 66 momentum equations for the gaseous phase. The liquid phase is modeled 67 using mass and momentum equations for liquid along with a turbulence clo-68 sure. The gas-liquid interactions are represented by drag, pressure, lift and 69

buoyancy forces. The model accounts for intergroup bubble transfer through 70 bubble coalescence, dissolution and breakup. The recent work of Moraga 71 et al. (2008) followed the approach of Carrica et al. (1998). A similar ap-72 proach is used by Buscaglia et al. (2002) who developed a double-averaged 73 multiphase model without taking into account the momentum balance in 74 the bubble phase. The exclusion of momentum equations for the bubble 75 phase makes the model more efficient, especially in a simulation involving 76 a number of bubble groups with different sizes. Shi et al. (2008) used the 77 method of Buscaglia et al. (2002) in the preliminary investigation of air 78 bubbles generated by breaking waves inside the surfzone. Although Carrica 79 et al.'s approach is more rigorous in theory in terms of the Favre-averaging, 80 Buscaglia et al.'s method still remains a valuable alternative as a computa-81 tional efficient model for practical purposes. 82

The focus of the present study is to estimate bubble population evolution 83 and spatial distribution in a breaking wave event. Due to the complexity 84 of wave breaking processes and the lack of sufficient knowledge of bubble 85 entrainment and water-bubble interaction, we intend to develop a simple 86 and physically based model. We will show developments of the model based 87 on Buscaglia et al. (2002) and components representing bubble coalescence, 88 breakup and bubble-induced turbulence effects. The model is tested against 89 laboratory data reported by Lamarre and Melville (1991), referred to here-90 after as LM91. 91

#### 92 2. TWO FLUID MODEL

Buscaglia et al. (2002) derived a two-fluid model using a double-averaging 93 approach. The first average was performed at spatial scales of the order 94 of the bubble-to-bubble spacing  $L_{bb}$  and resulted in mass and momentum 95 conservation equations for a gas-liquid mixture. The second average was 96 carried out using Reynolds averaging over the gas-liquid mixture equations 97 at larger turbulence scales. The governing equation for the bubble phase was 98 the Reynolds-averaged mass balance equation, taking into account bubble gc diffusion due to turbulence. The two-fluid model of Buscaglia et al. (2002) 100 involves a liquid chemistry process which incorporates oxygen and nitrogen 101 dissolution in applications to bubble plumes. Two bubble groups, i.e., oxygen 102 group and nitrogen group, with a uniform bubble size were considered. No 103 bubble breakup or coalescence is taken into account in their model. 104

In this section, we review the basic equations of the two-fluid model derived by Buscaglia et al. (2002). Some modifications and additions are made in order to represent polydisperse bubble population, bubble-induced turbulence, bubble breakup and coalescence.

#### 109 2.1. Mixed Fluid Phase

The double-averaged equations include mass conservation and momentum equations for the mixture fluid phase:

$$\nabla \cdot \mathbf{u}_m = 0 \tag{1}$$

$$\frac{\partial \mathbf{u}_m}{\partial t} + \mathbf{u}_m \cdot \nabla \mathbf{u}_m + \frac{1}{\rho_0} \nabla P_m = \frac{1}{\rho_0} \nabla \cdot (2\mu_t \mathbf{S}) - \frac{\rho_m}{\rho_0} g \mathbf{k}$$
(2)

where  $\mathbf{u}_m, P_m$  and  $\rho_m$  represent the mixture quantities of fluid velocity, pres-112 sure and density, respectively.  ${\bf k}$  is a vertical unit vector.  $\rho_0$  is a reference 113 density which has replaced  $\rho_m$  in all terms but the gravity term using the 114 Boussinesq approximation. It is noted that the Boussinesq approximation 115 is invalid for the mixture fluid with a high and inhomogeneous distribution 116 of void fraction. It is assumed in the present study that high void frac-117 tion is localized within a limited region so that the pressure gradient caused 118 the spatial variation in density would not affect much the overall wave form 119 evolution. The assumption is confirmed to be appropriate in the numerical 120 results shown in section 3.2. 121

 $\mathbf{S}$  represents the rate of strain tensor of the mean flow defined by

$$\mathbf{S} = \frac{1}{2} (\nabla \mathbf{u}_m + \nabla^T \mathbf{u}_m), \tag{3}$$

 $\mu_t$  is the eddy viscosity coefficient which is related to turbulent kinetic energy, k, and turbulent dissipation,  $\epsilon$ , in the  $k - \epsilon$  turbulence equations shown in Section 2.3. The relation between k and  $\epsilon$  can be expressed by

$$\mu_t = \rho_0 C_\mu \frac{k^2}{\epsilon} \tag{4}$$

where  $C_{\mu}$  is an empirical coefficient and  $C_{\mu} = 0.09$  was used as suggested by Rodi (1980). The last term in (2) represents the buoyancy force which can be evaluated by

$$\frac{\rho_m}{\rho_0} g \mathbf{k} = (1 - \alpha_b) g \mathbf{k} \tag{5}$$

where  $\alpha_b$  is the volume fraction of bubbles following the definition in Drew and Passman (1998).

#### 132 2.2. Bubble Phase

In this study, we do not employ the multicomponent gas model and asso-133 ciated chemistry framework of Buscaglia et al. (2002). Instead, we consider 134 the gas to be a single, inert component, and we neglect dissolution of the 135 gas phase in water. The bubble population is split into NG groups based on 136 bubble radius. The equations for the bubble phase include the equations of 137 the gas molar concentration and bubble number density with different bubble 138 sizes. Bin i of the bubble population is calculated using simple advection-139 diffusion equations given by 140

$$\frac{\partial C_{b,i}}{\partial t} + \nabla \cdot (C_{b,i} \mathbf{u}_g) = \mathcal{E}_{c,i} + \mathcal{S}_{c,i} + \nabla \cdot (\mathcal{D}_g \nabla C_{b,i})$$
(6)

141

$$\frac{\partial N_{b,i}}{\partial t} + \nabla \cdot (N_{b,i} \mathbf{u}_g) = \mathcal{E}_{n,i} + \mathcal{S}_{n,i} + \nabla \cdot (\mathcal{D}_g \nabla N_{b,i})$$
(7)

where  $C_{b,i}$  and  $N_{b,i}$  represent, respectively, the gas molar concentration and bubble number per unit volume for bubble size *i*. The total gas molar con<sup>144</sup> centration and bubble number per unit volume are, respectively,

$$C_b = \sum_{i=1}^{NG} C_{b,i},\tag{8}$$

145 and

$$N_b = \sum_{i=1}^{NG} N_{b,i}.$$
 (9)

 $_{^{146}}$   $~\mathbf{u}_g$  is the bubble advection velocity which can be calculated by

$$\mathbf{u}_g = \mathbf{u}_m + w_s(r_b)\mathbf{k} \tag{10}$$

<sup>147</sup> in which  $w_s(r_b)$  is the bubble-slip velocity, assumed to depend on the bubble <sup>148</sup> radius following Clift et al. (1978):

$$w_{s} = \begin{cases} 4474 \text{ m/s} \times r_{b}^{1.357} & \text{if } 0 \leq r_{b} \leq 7 \times 10^{-4}m \\ 0.23 \text{ m/s} & \text{if } 7 \times 10^{-4} < r_{b} \leq 5.1 \times 10^{-3}m \\ 4.202 \text{ m/s} \times r_{b}^{0.547} & \text{if } r_{b} > 5.1 \times 10^{-3}m \end{cases}$$
(11)

 $\mathcal{E}_{c,i}$  and  $\mathcal{E}_{n,i}$  are source terms associated with bubble entrainment.  $\mathcal{S}_{c,i}$  and  $\mathcal{S}_{n,i}$  are source/sink terms associated with inter-group adjustment of bubble quantity between different component *i* caused by bubble size changes due to pressure change, bubble breakup and coalescence, and will be described in the following sections.  $\mathcal{D}_g$  is the dispersion coefficient associated with the turbulence and bubble-bubble interaction. In the isotropic model proposed  $_{155}$  by Carrica et al. (1998),

$$\mathcal{D}_g = \frac{\mu_t}{\rho_0 S_g} \tag{12}$$

where  $S_g$  is the Schmidt number for gas (Buscaglia et al., 2002). The gas volume fractions used in (5) can be calculated using

$$\alpha_b = \frac{\mathcal{R}T_g \sum_i C_{b,i}}{P_g} \tag{13}$$

where  $\mathcal{R}$  is the universal gas constant, 8.314 J/mol K.  $T_g$  is the absolute gas temperature,  $P_g$  is gas pressure, assumed equivalent to  $P_m$ . The bubble radius can be calculated using

$$r_{b,i} = \left(\frac{3\nu_{b,i}}{4\pi}\right)^{1/3} \tag{14}$$

where  $\nu_b(i)$  is the bubble volume of component *i* which can be obtained by

$$\nu_{b,i} = \frac{C_{b,i} \mathcal{R} T_g}{P_g N_{b,i}} \tag{15}$$

In Shi et al. (2008), both the gas molar concentration equation (6) and bubble number intensity equation (7) for each group were solved in order to take into account the intergroup transfer caused by ambient pressure change. In applications of surface wave breaking in shallow water, both spatial and temporal variations in pressure field are small with respect to the atmospheric pressure at the water surface. For example, in the following application of a laboratory experiment,  $\Delta P_m/P_0 < 0.07$ , where  $P_0$  is atmospheric pressure, resulting in at most 2% radius variation due to pressure changes. It was found no intergroup transfer caused by pressure changes in the laboratory case. Although both of (6) and (7) are implemented in the model for general applications, only (7) was solved for the bubble phase in the present application for a purpose of efficiency. The void fraction is calculated by

$$\alpha_b = \sum_i N_{b,i} \nu_{b,i} \tag{16}$$

where  $\nu_{b,i}$  may be obtained using the relation between  $\nu_{b,i}$  and  $r_{b,i}$ , i.e., equation (14), under the assumption that bubble size is independent of gas pressure and temperature.

#### 177 2.3. Turbulence Model

Previous studies on turbulence modeling for two-phase flows indicated sig-178 nificant challenges in developing a suitable coupled regime between turbulent 179 eddies and air bubbles with less knowledge in physical mechanism and scarce 180 experimental studies (Banerjee, 1990). Turbulence plays an important role 181 in the non-linear process of bubble breakup and coalescence, whose feedback, 182 in turn, will affect the turbulent kinetic energy production (Sheng and Irons, 183 1993, Smith, 1998). In applications using transport equations for turbulence 184 quantities, such as the  $k - \epsilon$  model, a simple extension for the water-bubble 185 two phase flows is to modify the  $k - \epsilon$  model by adding some source terms in 186 the balance equations for k and  $\epsilon$ . This is based on the assumption that the 187

shear-induced and bubble-induced turbulence effects are decoupled, so that the bubble-induced turbulence can be evaluated separately based on semiempirical formulations (Kataoka and Serizawa, 1989, Lopez de Bertodano et al., 1994). The  $k - \epsilon$  equations may be written as

$$\frac{\partial k}{\partial t} + \nabla \cdot (k\mathbf{u}_m) = \nabla \cdot \left[ (\mu_0 + \frac{\mu_t}{\sigma_k}) \nabla k \right] + \mu_t |\mathbf{S}|^2 - \epsilon + S_k$$
(17)

192 and

$$\frac{\partial \epsilon}{\partial t} + \nabla \cdot (\epsilon \mathbf{u}_m) = \nabla \cdot \left[ (\mu_0 + \frac{\mu_t}{\sigma_\epsilon}) \nabla \epsilon \right] + C_{1\epsilon} \mu_t |\mathbf{S}|^2 \frac{\epsilon}{k} - C_{2\epsilon} \frac{\epsilon^2}{k} + S_\epsilon \qquad (18)$$

where  $\mu_0$  is the molecular kinematic viscosity;  $\sigma_k, \sigma_\epsilon, C_{1\epsilon}$  and  $C_{2\epsilon}$  are empirical coefficients with recommended values (Rodi, 1980)

$$\sigma_k = 1.0, \quad \sigma_\epsilon = 1.3, \quad C_{1\epsilon} = 1.44, \quad C_{2\epsilon} = 1.3$$
(19)

 $S_k$  and  $S_{\epsilon}$  represent source/sink terms associated with bubble-induced turbulence effects. In this study, Kataoka and Serizawa's (1989) approach is employed, which uses

$$S_k = -C_k \alpha_g \nabla p \cdot \mathbf{w}_s \tag{20}$$

198

$$S_{\epsilon} = C_{\epsilon} \cdot \frac{\epsilon}{k} S_k \tag{21}$$

where  $\mathbf{w}_s = w_s \mathbf{k}$  and the slip velocity for a bubble radius of 1mm was adopted in this study, and values of coefficients  $C_k$  and  $C_{\epsilon}$  are taken as 1.0.

#### 201 2.4. Bubble Entrainment

Studies of bubble characteristics under breaking waves have indicated 202 that the initial bubble entrainment and distribution are related to turbu-203 lence in the entraining fluid (Thorpe, 1982; Baldy, 1993; Garrett et al., 2000; 204 Mori et al., 2007). Baldy (1993) suggested that the bubble formation rate 205 depends on turbulent dissipation  $\epsilon$  and the bubble formation energy. He gave 206 a dimensional parameter-based source function which is linearly proportional 207 to  $\epsilon$ . Garrett et al. (2000) pointed out that, based on dimensional analy-208 sis, the bubble size spectrum should behave according to  $e^{-1/3}r^{-10/2}$  for a 209 given air volume entrained by breaking waves. Laboratory experiments by 210 Cox and Shin (2003) showed the dependence of void fraction on turbulence 211 intensity in the bore region of surf zone waves. Mori et al. (2007) show 212 a linear relationship between the void fraction and turbulence intensity in 213 their experimental study. Although the construction and parameterization 214 of a quantitative source function may be uncertain because of the lack of de-215 tailed observation, there is a general belief that bubble formation and initial 216 size distribution are related to turbulence. It is our understanding that, in a 217 wave breaking event, bubble generation is dependent on the intensity of wave 218 breaking and types of breakers. For plunging breakers, the major entrained 219 air volume is from an air pocket formed by a plunging jet impinging ahead 220 of the wave face. The injected air packet is broken up by turbulence into 221 small bubbles. For spilling breakers, air bubbles are entrained by a surface 222 roller and penetrate into the water column. At the beginning of the quiescent 223

phase, a statistical equilibrium in bubble size distribution is achieved with an
initial size spectrum of a power law (Garrett et al., 2000, Deane and Stokes,
2002).

The complexity of the bubble entrainment process and lack of knowledge 227 of the bubble entrainment mechanism make the formulation of a bubble 228 entrainment source function difficult. It is natural to start with a simple 229 source function to model bubbles entrained by breaking waves. In this study, 230 we model the initial bubble entrainment by connecting the production of 231 turbulent kinetic energy at the air-water interface and the bubble number 232 intensity with certain bubble size spectra observed by Deane and Stokes 233 (2002). The increment of initial bubble number per unit radius increment 234 can be written as 235

$$dN_{b,i} = a_b P_r D_i dt, \qquad P_r > P_{r0} \tag{22}$$

where  $a_b$  is a constant to be determined,  $P_r$  is the shear production term, i.e.,  $P_r = \mu_t |\mathbf{S}|^2$ ,  $P_{r0}$  is a threshold for the onset air entrainment,  $D_i$  is the bubble size probability function. Based on Deane and Stokes (2002), the bubble density per unit radius increment can by calculated by

$$N = \begin{cases} N_H \left(\frac{r_b}{r_H}\right)^{-3/2}, & r_{b,min} \le r_b \le r_H \\ N_H \left(\frac{r_b}{r_H}\right)^{-10/3}, & r_H < r_b \le r_{b,max} \end{cases}$$
(23)

240 where  $r_{b,min}$  and  $r_{b,max}$  represent respectively the minimum and maximum

bubble radius considered,  $N_H$  is the bubble density per unit radius increment 241 at the Hinze scale  $r_H$  (Hinze, 1955). Based on the formula of Hinze (1955), 242 the Hinze scale is a function of the turbulent dissipation, surface tension and 243 the critical Weber number. The relationship between bubble size distribution 244 and the intermittent dissipation rate or the average dissipation rate were 245 discussed in Garrett et al. (2000). In this study, we adopted  $r_H = 1 mm$ , 246 which was measured in Deane and Stokes (2002) rather than computed from 247 the model.  $r_H$  values estimated from computed dissipation rates in the model 248 fall in the range of 1 - 1.5mm in the region of established breaking described 249 below, and are thus consistent with the value  $r_H = 1mm$  which we apply 250 uniformly. The probability function  $D_i$  may be obtained using the bubble 251 density N normalized by the maximum N which is the value at  $r_{b,min}$ : 252

$$D_{i} = \begin{cases} r_{b,min}^{3/2} r_{b,i}^{-3/2}, & r_{b,min} \leq r_{b,i} \leq r_{H} \\ r_{b,min}^{3/2} r_{H}^{11/6} r_{b,i}^{-10/3}, & r_{H} < r_{b} \leq r_{b,max} \end{cases}$$
(24)

Figure 2 demonstrates an example of  $D_i$  with 20 bins of bubbles,  $r_{b,min} = 0.1 \ mm$  and  $r_{b,max} = 10 \ mm$ . This example will be used in the following application in Section 3.

According to (22), the source term 
$$\mathcal{E}_n$$
 can be written as

$$\mathcal{E}_{n,i} = a_b P_r D_i dr_{b,i} \qquad P_r > P_{r0} \tag{25}$$

where  $dr_{b,i}$  is the radius spacing of bin *i*. For given  $r_{b,i}$ ,  $\mathcal{E}_{n,i}$  and  $P_g$ , the source

term in the molar concentration equation  $\mathcal{E}_{c,i}$  is calculated using the ideal gas law (14) and and the relation between  $r_{b,i}$  and  $\nu_{b,i}$  (15). In our applications, the molar concentration was not calculated as described in Section 2.2.

#### 261 2.5. Bubble Coalescence and Breakup

Since only (7) was applied as the governing equation for the bubble phase in our application, only the source term  $S_{n,i}$  was taken into account for the intergroup transfer due to the bubble coalescence and breakup. It can be written as

$$\mathcal{S}_{n,i} = \chi_i^+ - \chi_i^- + \beta_i^+ - \beta_i^- \tag{26}$$

where  $\chi_i^{\pm}$  and  $\beta_i^{\pm}$  represent source/sink due to the coalescence and breakup, respectively. According to Prince and Blanch (1990), the coalescence source which represents the gain in bubble group *i* due to coalescence of smaller bubbles is given by

$$\chi_{i}^{+} = \frac{1}{2} \sum_{k,l < i} T_{kl} \zeta_{kl} X_{ikl}$$
(27)

where  $T_{kl}$  is the collision rate of bubble group k and l which can be evaluated by

$$T_{kl} = \frac{\sqrt{2}}{4} \pi (2r_{b,k} + 2r_{b,l})^2 \epsilon^{1/3} \left[ (2r_{b,k})^{2/3} + (2r_{b,l})^{2/3} \right]^{1/2} N_{b,k} N_{b,l}$$
(28)

 $\zeta_{kl}$  is the coalescence efficiency which represents the probability of coalescence when collision occurs. Based on Lou (1993),  $\zeta_{kl}$  is given by

$$\zeta_{kl} = \exp\left\{-\frac{\left[0.75(1 + (r_{b,k}^2/r_{b,l}^2)(1 + r_{b,k}^3/r_{b,l}^3)\right]^{1/2}}{(\rho_g/\rho_0 + 0.5)^{1/2}(1 + r_{b,k}/r_{b,l})^3}W_{kl}^{1/2}\right\}$$
(29)

where  $\rho_g$  is air density;  $W_{kl}$  is the Weber number (see Luo, 1993 or Chen et al., 2005). The last item in (27),  $X_{ikl}$ , is the number of bubble transfered from the coalescence of two bubbles from group k and l to group i

$$X_{ikl} = \frac{\nu_k + \nu_l - \nu_{i-1}}{\nu_i - \nu_{i-1}} \quad \nu_{i-1} < \nu_k + \nu_l < \nu_i \tag{30}$$

277

$$X_{ikl} = \frac{\nu_{i+1} - (\nu_k - \nu_l)}{\nu_{i+1} - \nu_i} \quad \nu_i < \nu_k + \nu_l < \nu_{i+1}$$
(31)

<sup>278</sup> The sink caused by coalescence in group i can be calculated by

$$\chi_i^- = \sum_{k=1}^{NG} T_{ik} \zeta_{ik} \tag{32}$$

<sup>279</sup> The source term of bubble breakup is calculated by

$$\beta_i^+ = \sum_{k=i}^{NG} \phi_k X_{ik} \tag{33}$$

where  $\phi_k$  is a breakup kernal function given by Luo and Svendsen (1996),

$$\phi_k = c_b \alpha_b N_{b,k} \left(\frac{\epsilon}{4r_{b,k}^2}\right)^{1/3} \int_{\xi_{min}}^1 \frac{(1+\xi)^2}{\xi^{11/3}} \times \exp\left[-\frac{12c_f \sigma}{\gamma \rho_0 \epsilon^{2/3} (2r_b)^{5/3} \xi^{11/3}}\right] d\xi \quad (34)$$

in which  $c_b$ ,  $\gamma$  and  $c_f$  are constants and  $c_b = 0.923$ ,  $c_f = 0.2599$ , and  $\gamma = 2.04$ in this study,  $\sigma$  is surface tension,  $\xi$  is the dimensionless eddy size and  $\xi_{min}$ is the minimum value of  $\gamma$  which can be obtained using the minimum eddy size given by van den Hengel et al. (2005):

$$\lambda_{min} = 11.4 \left(\frac{\mu_0^3}{\epsilon}\right)^{1/4} \tag{35}$$

We assume that the breakup splits a bubble into two identical daughter bubbles thus that  $X_{ik}$  can be written as

$$X_{ik} = 2\frac{\nu_k/2 - \nu_{i-1}}{\nu_i - \nu_{i-1}} \quad \nu_{i-1} < \nu_k/2 < \nu_i \tag{36}$$

287

$$X_{ik} = 2\frac{\nu_{i+1} - \nu_k/2}{\nu_{i+1} - \nu_i} \quad \nu_{i+1} > \nu_k/2 > \nu_i \tag{37}$$

288

$$X_{ik} = 0 \qquad \text{otherwise} \tag{38}$$

#### <sup>289</sup> The sink term of bubble breakup is calculated using

$$\beta_i^- = \phi_i \tag{39}$$

It should be mentioned that there are other bubble breakup and coalescence models to choose for this study. For example, our newly developed 3D model (Ma et al., 2010, in preparation) with the similar air entrainment approach utilizes the model of Martínez-Bazán (1999). Because of the purpose of this paper, the differences in using different models and effects of bubble <sup>295</sup> breakup and coalescence are not discussed. Interested readers can refer to
<sup>296</sup> Lasheras et al. (2002) and Chen et al. (2005).

297 2.6. Model implementation

We use the 2-D VOF model RIPPLE (Kothe et al., 1991) as the basic 298 framework for the computational code. The VOF model is a single phase 290 model and has been enhanced with several different turbulence closure models 300 such as  $k - \epsilon$  model (Lin and Liu, 1998) and multi-scale LES (Large Eddy 301 Simulation) model (Zhao et al., 2004, Shi et al., 2004). In this study, we 302 adopted the  $k - \epsilon$  approach with extra source terms,  $S_k$  and  $S_{\epsilon}$ , to account 303 for bubble-induced turbulence effects. The buoyancy force was added in the 304 model using formula (5) in which the void fraction  $\alpha_b$  may be evaluated 305 using (16). The governing equation for the bubble number intensity (7) of 306 each bubble group was implemented using the standard numerical scheme 307 for advection-diffusion equation which exists in the VOF code. 308

# 309 3. APPLICATION TO BREAKING WAVE EXPERIMENT OF 310 LAMARRE AND MELVILLE (1991)

We test the capabilities of the present model by comparing to experimental data on an isolated breaking event, as studied in laboratory conditions by Rapp and Melville (1990) and LM91. The wave breaking event in this study is generated in a narrow flume and is mainly two-dimensional, aside from complex flow structures generated in the breaking wave crest, and is thus reasonably well suited for study by the present two-dimensional model.

#### 317 3.1. Model setup and test runs

LM91 conducted measurements of air bubbles entrained by controlled 318 breaking waves in a wave flume 25 m long and 0.7 m wide filled with fresh 319 water to a depth of 0.6 m. Breaking waves were produced by a piston-320 type wave maker generating a packet of waves with progressively decreasing 321 frequency (Rapp and Melville, 1990), leading to a focussing of wave energy 322 at a distance  $x_f = 8.46 \ m$  from the wave paddle. The wave packet was 323 composed of N = 32 sinusoidal components of slope  $a_i k_i$  where  $a_i$  and  $k_i$  are 324 the amplitude and wave number of the ith component. Based on the linear 325 composition, the surface displacement is 326

$$\eta(x,t) = \sum_{i=1}^{N} a_i \cos[k_i(x-x_f) - 2\pi f_i(t-t_f)]$$
(40)

where  $f_i$  is the frequency of the *i*th component;  $x_f$  and  $t_f$  are the location and 327 time of focusing, respectively. In the experiments, the discrete frequencies  $f_i$ 328 were uniformly spaced over the band  $\Delta f = f_N - f_1$  with a central frequency 329 defined by  $f_c = \frac{1}{2}(f_N - f_1)$ . The wave packet envelope steepness may be 330 evaluated by  $\Delta f/f_c$ . In the numerical study, we use a computational domain 331 with dimensions of 30 m in the horizontal direction and 0.8 m in the vertical 332 direction. The coordinates are specified in  $x = -10 \sim 20m$  and  $z = -0.6 \sim$ 333 0.20m with the still water level at z = 0 and with x = 0 corresponding to 334 the wavemaker position. An internal wave maker (Lin and Liu, 1999) was 335 applied at x = 0 and generates the wave packet based on (40). A sponge 336

layer with a width of 5 m was used at each end of the domain to avoid wave reflection from the boundaries. The computational domain is discretized into  $1501 \times 201$  nonuniform cells with a minimum spacing of 0.01 m in 4 m $\leq x \leq 10 m$ , and 201 cells in z direction with a minimum spacing of 0.0025 m at z > 0 m, as shown in Figure 1.

Based on sensitivity tests on bubble group numbers, we adopted 20 groups 342 of bubbles with the smallest radius  $r_{b,min} = 10^{-1} mm$  and the largest radius 343  $r_{b,max} = 10 \ mm$ . The other 18 bubble radii were obtained by equal splitting in 344 the logarithm of bubble radius between  $10^{-1}$  and 10 mm and are respectively 345 0.13, 0.16, 0.21, 0.26, 0.34, 0.43, 0.55, 0.70, 0.89, 1.13, 1.43, 1.83, 2.33, 2.98,346 3.79, 4.83, 6.16, and 7.85 mm. The bubble size probability density function 347 with respect to the 20 bubble sizes can be obtained based on (24) and is 348 shown in Figure 2. 349

In the laboratory experiments,  $f_c = 0.88$  and  $\Delta f/f_c = 0.73$  were used and constant amplitude, i.e., $a_i = a_c$ , was specified. In the numerical study, we carried out two cases, one with  $a_c k_c = 0.38$ , corresponding to the case where void fraction was measured and used for analysis in LM91, the other with  $a_c k_c = 0.352$ , corresponding to the case for which photographs of the bubble cloud are given by Rapp and Melville (1990). Here,  $k_c$  is the central wave number corresponding to  $f_c$ .

In order to calibrate the model, we performed a series of model test runs with different adjustable parameters,  $a_b$ ,  $P_{r0}$ , and  $S_g$ , towards the model results best suited to the measured data. In LM91, void fractions of > 20%

for several test cases were observed for up to half a wave period after breaking. 360 The void fractions near the surface can reach  $40 \sim 50\%$  (measured at 0.26T361 , where T is the wave period, shown in Figure 3a in LM91). Among the 362 adjustable parameters,  $a_b$  was found to be the most sensitive parameter for 363 the overall void fraction level due to its representation of the air entrainment 364 Therefore, we focussed on the adjustment of  $a_b$  and adopted fixed rate. 365 parameters  $P_{r0} = 0.02 \ m^2/s^3$  and  $S_g = 0.7$  in all test runs. Figure 3 shows 366 the maximum void fractions with respect to different  $a_b$  chosen for test cases. 367 A nearly linear relation between  $a_b$  and the maximum void fraction was 368 observed. The parameter  $a_b = 1.45 \times 10^9$  predicted the maximum void 369 fraction of 40% which generally suits for the maximum void fraction level 370 observed in the laboratory experiments. It was used to generate numerical 371 results for comparisons with the measured data. 372

#### 373 3.2. Model results

Figure 4 shows predicted wave breaking patterns and contours of the void 374 fraction above 0.1%, with comparisons to photo images (first and third col-375 umn) of the bubble cloud taken in the laboratory experiments in the case 376 of  $ak_c = 0.352$ . The model predicts a wave break point which is about one 377 wave length upstream of the focal point  $x_f = 8.46 m$ , as observed in the 378 laboratory experiments. The predicted water surface evolution agrees well 379 with that shown in the images. Patterns of predicted void fraction contours 380 generally match the images of the bubble cloud, although the predicted void 381

fraction distribution does not contain some of the structural detail apparent in the photographed bubble cloud. Some bubble cloud deepening patterns shown in the images in  $t = 19.30 \sim 19.50 \ s$  were not observed in the modeled distribution of void fraction. These patterns may be caused by three dimensional effects including obliquely descending eddies (Nadaoka et al., 1989).

Predicted air entrainment is directly connected to shear production in 388 the entrainment formula (25). Results for bubble number density (above 389  $5 \times 10^6$ ) and the corresponding shear production above the threshold  $P_{r0} =$ 390 0.02  $m^2/s^3$  are shown in Figure 5. The water surface elevation and elapsed 391 time shown in the figure were normalized respectively by central wave number 392  $k_c$  and wave period T associated with the central frequency  $f_c$ .  $t_b$  is the 393 time at breaking, which was approximately determined by the time when 394 the turbulent production started to increase significantly. When the wave 395 starts to break, bubbles are entrained at the wave crest and bubble number 396 density is localized in a small area. As the breaking bore moves forward, the 397 wave height drops rapidly, accompanied by more intense shear production 398 leading to more significant bubble entrainment. The shear production above 399 the air entrainment threshold is persistently located at the breaking wave 400 crest with a moderate time variation in its value, while the bubble number 401 density varies by an order of magnitude over an elapsed time corresponding 402 to a wave period. The distribution of bubble number intensity indicates that 403 bubbles spread downstream and form a long tail of the bubble cloud beneath 404

405 the wave surface.

Following LM91, who used 0.3% void fraction as a threshold to evaluate 406 the volume of entrained air, we show snapshots of distribution of void fraction 407 above the threshold 0.3% in Figure 6. The figure shows that, at beginning 408 of wave breaking, air is entrained at the wave crest and the area bounded 409 by the threshold 0.3% is small. The bounded area increases as the wave 410 moving forward and reaches a maximum around the half wave period. After 411 the maximum area is reached, the overall void fraction decreases due to the 412 degassing process. The higher void fraction can be found at the leading bore 413 followed by a long tail of lower void fraction. 414

In LM91, several moments of the void fraction field were computed from void fraction measurements according to the following definitions,

$$A = \int_{A} dA, \tag{41}$$

417

$$V = \int_{A} \alpha_b dA, \tag{42}$$

418 and

$$\bar{\alpha}_b = V/A,\tag{43}$$

where A is the total cross-sectional area of the bubble plume above a void fraction threshold, V is the volume of air entrained per unit width, and  $\bar{\alpha}_b$ is the void fraction averaged over A. LM91 fitted functional expressions to computed values of A, V and  $\bar{\alpha}$  which are used here for comparison to the  $_{423}$  model results. The data for V was fitted by

$$V/V_0 = 2.6 \exp(-3.9(t - t_b)/T),$$
 (44)

where  $V_0$  is a reference value of the volume of air per unit width. In LM91 and Lamarre and Melville (1994),  $V_0$  was evaluated as  $V_0 = V(t = t_b + 0.2T)$ or by the maximum V.

427 The data for A was fitted by

$$A/V_0 = 325 \left( (t - t_b)/T \right)^{2.3}, \tag{45}$$

<sup>428</sup> and  $\bar{\alpha}_b$  was fitted by

$$\bar{\alpha}_b(\%) = 0.8 \left( (t - t_b)/T \right)^{-2.3}.$$
 (46)

Note that, in the formula of LM91,  $()^{-2/3}$  was given in (46) and is believed to be a typo.

We calculated A, V and  $\bar{\alpha}_b$  from numerical results in the same way as in LM91. A and V were normalized by  $V_0$  which is the maximum V in the numerical results. Figure 7 shows the normalized A, V and  $\bar{\alpha}_b$  computed from the model results with comparisons to the data fitted curves. The threshold used in the calculations is 0.3%. Figure 7 (a) shows that the model predicted a parabolic-like evolution of the void fraction area A, which has a similar trend as shown by the data fitted curve. The area A was over-predicted at

the beginning of wave breaking, and a more moderate increase in A can be 438 found in the early time in the wave period, compared with the data fitted line. 439 The comparison of the normalized air volume  $V/V_0$  shown in Figure 7 (b) 440 indicates an underprediction of air entrainment at the beginning of breaking, 441 which is consistent with the absence of a large entrained pocket of air in 442 the numerical simulation. In the first half wave period, the volume  $V/V_0$ 443 decreases more slowly than indicated by the data, resulting in overprediction 444 of  $V/V_0$  around the middle of the wave period. At later times, the decay rate 445 of the air volume  $V/V_0$  agrees reasonably well with data. Figure 7 (c) shows 446 the void fraction averaged over the area A in comparison to the data-fitted 447 curve (46). Again, an underprediction of the average void fraction can be 448 found at the beginning of wave breaking, followed by overpredictions at later 449 times. In general, the model predictions of the magnitude and evolutionary 450 trend of the average void fraction are in reasonable agreement with the data. 451 The underprediction at the beginning of wave breaking was expected because 452 the model does not account for large air pockets in the continuum phase. 453 The horizontal and vertical centroids of the void-fraction distribution in 454

$$(x_m, z_m) = \frac{\int_A \alpha_b(x, z) dA}{\int_A \alpha_b dA}$$
(47)

where x is the horizontal distance from  $x_b$  and z is the depth from the free surface. The top panel of Figure 8 shows the horizontal centroid normal-

the bubbly plume were also calculated using

455

ized by the wave length  $\lambda_c$  corresponding to the central frequency  $f_c$ . The 458 horizontal centroid moves at roughly the phase speed (slope of the dashed 459 line) in the early stage after wave breaking and gradually slows down in the 460 later time. Compared with the measurements in LM91 (Figure 3e in LM91), 461 the model predicted the tendency of the  $x_m$  evolution but over-predicted the 462 speed of the horizontal centroid in the later time. The normalized vertical 463 centroid  $z_m$  is shown in the bottom panel of Figure 8. The vertical centroid 464 is roughly constant, which is consistent with the measurements (Figure 3f in 465 LM91). It was explained by Lamarre and Melville (1991) that the downward 466 advection of fluid may balance the upward motion of the bubbles themselves. 467

According to observations in laboratory experiments, bubbles with differ-468 ent sizes make different contributions to the void fraction, with larger bubbles 469 contributing more directly to higher void fraction values. The contributions 470 of different size bubbles to void fraction can be demonstrated by the void 471 fraction distribution calculated from each bubble group. Figure 9 shows 472 snapshots of the void fractions contributed by group bins  $r_b = 0.10, 0.55, 1.43$ 473 and 6.16 mm at  $(t - t_b)/T = 0.62$ . Note that the radius bins are not evenly 474 split based on bubble radius and thus the void fraction calculated from each 475 bin may not represent the exact contribution from bubbles at specific size. 476 However, the figure shows orders of magnitude differences in the void frac-477 tions contributed from different bins and indicates that bubbles larger than 478 O(1) mm make a major contribution to void fraction. Smaller bubbles do 479 not contribute much to the total volume of air but contribute significantly 480

<sup>481</sup> to the cross-sectional area of the bubble cloud.

Two major discrepancies between the model and the data were observed 482 in the predicted void fraction distribution with comparison to the bubble 483 plume shape captured in the laboratory experiments (LM91; Lamarre and 484 Melville, 1994). First, the core region of the high void fraction measured 485 in the bubble plume is basically located in front of the wave surface peak 486 where large air pockets take place. The model predicted the core region at 487 the wave surface peak where the shear production is maximum. Second, the 488 bubble plume represented by the predicted void fraction does not look like 489 a semicylindrical plume as described in Lamarre and Melville (1994). The 490 maximum void fraction does not appear at the air-water interface as shown in 491 the measurements. The dislocation of the predicted core region of the high 492 void fraction is probably due to the air entrainment algorithm formulated 493 by the shear production which reaches maximum around the wave crest as 494 shown in Figure 5. The model does not predict correctly large air pockets 495 on the surface of the breaking bore, resulting in the under-prediction of void 496 fraction at the air-water interface. 497

It is interesting to look at the moments calculated from different void fraction thresholds, as the moments calculated using a larger void fraction threshold reflect the evolution of larger bubble populations. Figure 10 shows the moments calculated from the void fraction thresholds 3% and 10% in addition to that from 0.3%. Apparently, the area bounded by a larger threshold is generally smaller than the area bounded by a smaller one as demonstrated

in Figure 10 (a). A parabolic-like evolution can be also found in the larger 504 threshold cases. The area bounded by the threshold 10% starts to decrease 505 earlier compared with the areas with 3% and 0.3% threshold, indicating the 506 stronger degassing effects for larger bubbles. The decay rates of air volume 507 calculated using different thresholds are shown in Figure 10 (b). As expected, 508 the air volume from a larger threshold decays faster, especially for the case 509 with a 10% threshold. The averaged void fractions  $\bar{\alpha}_b$  with different void 510 fraction threshold are shown in Figure 10 (c). For the 10% threshold, the 511 averaged void fraction decays faster at the beginning of breaking, indicating 512 that more larger size bubbles are contained in the sectional area bounded 513 by the large threshold at the beginning and escape the water column due to 514 degassing. 515

The evolution of bubble cloud can be measured by the retention time 516 of bubbles in the water column. Figure 11 shows the evolution of bubble 517 number integrated over the water column between  $(x - x_b)/\lambda_c = 0$  and 1.0 518 for bubble group bins  $r_b = 0.10, 0.34, 0.89, 2.33$ , and 6.16 mm. The bubble 519 numbers are normalized by the maximum bubble number during the time 520 period of  $(t - t_b)/T = 0 \sim 1.0$  for each bin. The figure shows that bubble 521 counts for different bubble sizes reach maxima at different times and decay 522 at different rates. For larger bubbles, such as  $r_b = 2.33$  and 6.16 mm, bubble 523 numbers reach their maxima when the turbulent production becomes most 524 intense at  $(t - t_b)/T = 0.2$ , and then bubble numbers decay rapidly because 525 the degassing process dominates over the bubble entrainment. For smaller 526

<sup>527</sup> bubbles, such as  $r_b = 0.1$  and 0.34 mm, weak degassing causes accumulation <sup>528</sup> of bubbles and results in the maxima at later times. The figure also indicates <sup>529</sup> that a significant amount of smaller bubbles are retained in the water column <sup>530</sup> at the end of a wave period.

Field and laboratory experiments (e.g., Deane and Stokes, 2002) revealed 531 that the bubble size spectrum changes in both time and space during the 532 evolution of bubble population. Figure 12 demonstrates the bubble size 533 spectrum at four depths,  $dk_c = 0.0749, 0.1233, 0.1720$ , and 0.2207, where 534 d is the depth below the surface of the wave crest at  $(t - t_b)/T = 0.62$  and 535  $(x - x_b)/\lambda_c = 0.5530$ . In general, all slopes basically follow the input spec-536 trum (Deane and Stokes, 2002) with slight increases with depth. The figure 537 also indicates that bubbles with smaller sizes tend to penetrate deeper and 538 stay longer in the water column, resulting in significant contribution to the 539 cross-sectional area of the bubble cloud as measured in the laboratory exper-540 iments. After initial bubble entrainment, the bubble size spectrum depends 541 on bubble evolution processes in the quiescent phase such as bubble further 542 breakup, coalescence, degassing, dissolution, advection and diffusion under 543 turbulent flow. The effects of those physical processes are modeled by the 544 individual algorithms described in Section 2. Because of the lack of detailed 545 measurements, those individual processes are not addressed in the paper. 546

#### 547 4. CONCLUSION

The present work has been motivated by the need of an efficient physics-548 based numerical model for prediction of air bubble population in a surfzone-549 scale domain. Directly modeling of air bubble entrainment and evolution 550 at this scale is computationally unaffordable. In this study, we proposed a 551 two-fluid model in which the air entrainment is formulated by connecting 552 the shear production at air-water interface and the bubble number intensity 553 with a certain bubble size spectra as observed by Deane and Stokes (2002). 554 The model fed with the initially entrained bubbles basically simulates bubble 555 plumes, and requires much less spatial and temporal resolution than needed 556 to capture detailed air entrainment process. 557

The two-fluid model was developed based on Buscaglia et al. (2002). A 2-D VOF RANS model with the  $k - \epsilon$  turbulence closure was used to model the gas-liquid mixture phase. The bubble phase was modeled using the equation of bubble number density equation for a polydisperse bubble population. The two-fluid model takes into account bubble-induced turbulence effects and intergroup transfer through bubble coalescence and breakup processes.

The model was used to simulate breaking wave-induced bubble plumes measured by LM91. The air entrainment parameter calibrated using the maximum void fraction measured in the laboratory experiments resulted in reasonable agreements between the predicted and the measured moments of the void fraction field defined by LM91. The model predicted a paraboliclike evolution of the bubble area bounded by the 0.3% threshold of void

fraction. The decay rates of air volume and averaged void fraction are gener-570 ally consistent with the laboratory experiments. The model results revealed 571 that bubbles larger than 1 mm make a major contribution to void fraction, 572 while smaller bubbles contribute significantly to the cross-sectional area of 573 the bubble cloud but do not contribute much to the total volume of air. A 574 stronger degassing effect on larger bubbles is evidenced by the earlier drop 575 of the bubble plume area and the faster decay of the air volume bounded by 576 a larger void fraction threshold compared with those bounded by a smaller 577 threshold. 578

The model with the calibrated parameter  $a_b$  underpredicted the void fraction at the beginning of breaking. The core region of the high void fraction was predicted at the wave surface peak where the shear production reaches maximum while the measurements show the core region in front of the wave surface peak where large air pockets occur. The maximum void fraction was not predicted at the air-water interface as in the measurements.

A primary source of discrepancies between observations and model behav-585 ior is the single-phase model used for the mixture phase and the algorithm 586 used in the air entrainment formulation. The VOF model employed here does 587 not account for the entrainment of identifiable gas pockets during the early 588 stages of breaking, and the contribution of these pockets to initial average 589 void fraction is absent. Our present research utilizes a model which incor-590 porates a discrete air phase which can contribute to directly represented air 591 pockets entrained by surface overturning or other folding effects. In addition, 592

<sup>593</sup> algorithms are being developed which will be utilized to move entrained air
<sup>594</sup> volumes from a discrete two-phase representation into the continuum mul<sup>595</sup> tiphase representation, in order to continue computations without requiring
<sup>596</sup> the VOF algorithm to maintain the identity of larger entrained bubbles.

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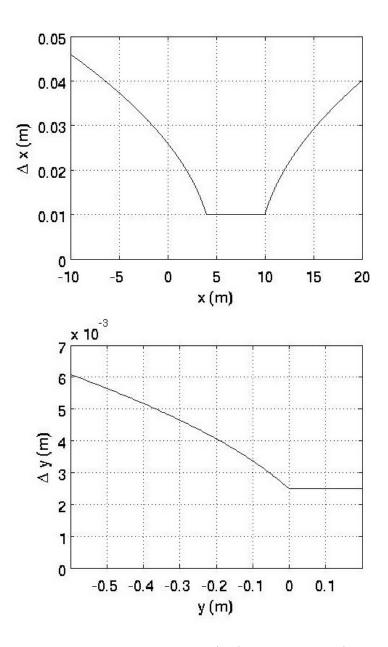


Figure 1: Grid spacing in x direction (top) and y direction (bottom).

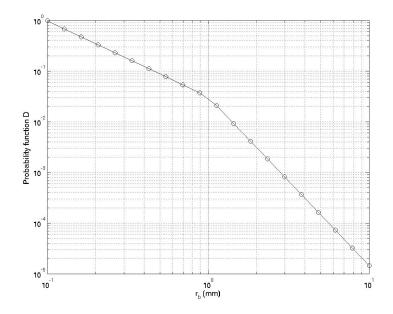


Figure 2: Bubble size probability density function D (circles represent values at 10 radius bins in the present application).

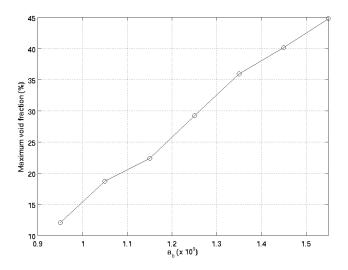


Figure 3: Maximum void fractions from test runs with different  $a_b$ .

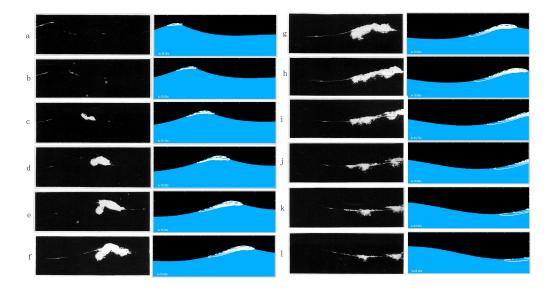


Figure 4: Photographs of the breaking wave and bubble cloud (first and third column) in Rapp and Melville (1990) versus predicted wave surface and void fraction contours above 0.1% in the case of  $f_c = 0.88$  Hz,  $ak_c = 0.352$ , and  $\Delta f/f_c = 0.73$ . Time is from 18.90 to 20.40 s with 0.10 s interval.

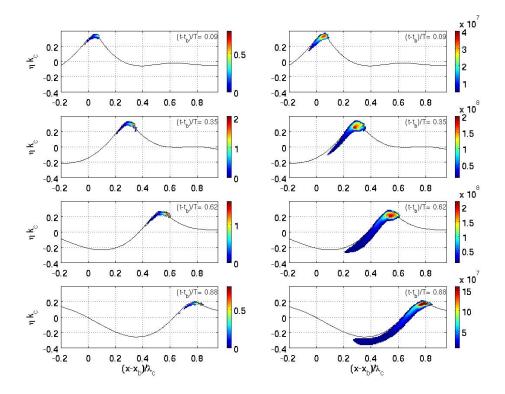


Figure 5: Left: shear production above the threshold  $P_{r0} = 0.02m^2/s^3$ , right: bubble number density above  $5 \times 10^6$  per  $m^3$ , at  $(t - t_b)/T = 0.09, 0.35, 0.62$  and 0.88. Case:  $f_c = 0.88$  Hz,  $ak_c = 0.38$ , and  $\Delta f/f_c = 0.73$ .

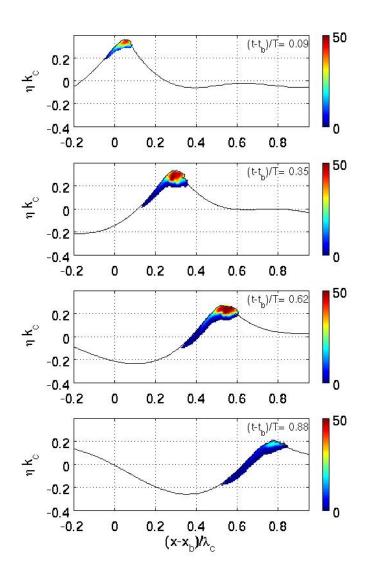


Figure 6: Void fraction larger than the threshold 0.3% at  $(t - t_b)/T = 0.09, 0.35, 0.62$  and 0.88 in the case of  $f_c = 0.88$  Hz,  $ak_c = 0.38$ , and  $\Delta f/f_c = 0.73$ .

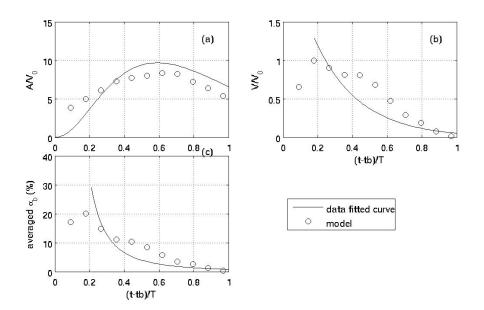


Figure 7: Moments calculated using the void fraction threshold of 0.3%, (a) Cross-sectional area A of bubble plume normalized by  $V_0$ ; (b) Air volume V normalized by  $V_0$ ; (c) Mean void fraction  $\bar{\alpha}_b$ . Case:  $f_c = 0.88$  Hz,  $ak_c = 0.38$ , and  $\Delta f/f_c = 0.73$ . Solid curves are functional fits to laboratory data from LM91. Model results shown as open circles.

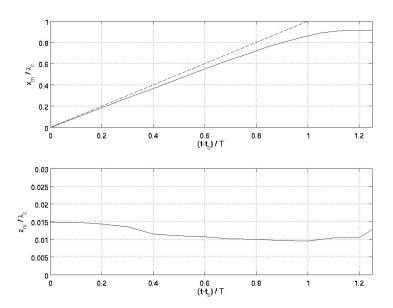


Figure 8: The horizontal centroid (top) and vertical centroid (bottom) normalized by wave length.

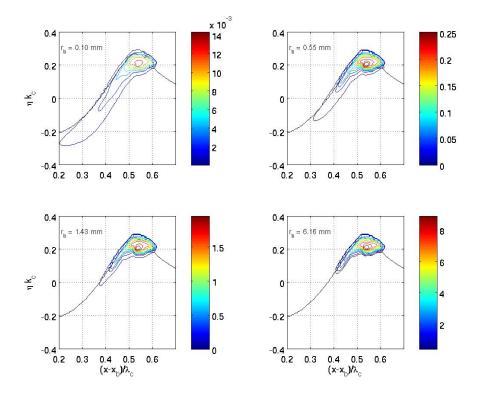


Figure 9: Void fraction (color contours %) contributed from group bins  $r_b = 0.10, 0.55, 1.43$ and 6.16 mm in the case of  $f_c = 0.88$  Hz,  $ak_c = 0.38$ , and  $\Delta f/f_c = 0.73$ .

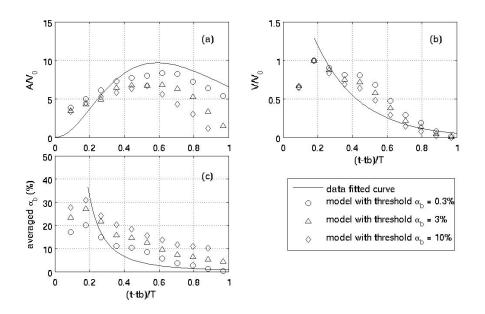


Figure 10: Moments calculated using the void fraction threshold of 0.3%, 3% and 10%, (a) Cross-sectional area A of bubble plume normalized by  $V_0$ ; (b) Air volume V normalized by  $V_0$ ; (c) Mean void fraction  $\bar{\alpha}_b$ . Case:  $f_c = 0.88$  Hz,  $ak_c = 0.38$ , and  $\Delta f/f_c = 0.73$ .

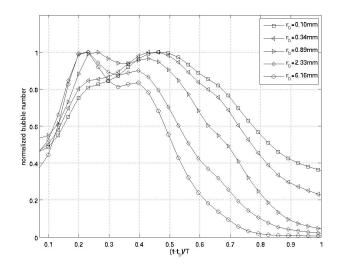


Figure 11: Evolutions of normalized bubble numbers in the water column between  $(x - x_b)/\lambda_c = 0$  and 2.5 for bubble group bins  $r_b = 0.10, 0.34, 0.89, 2.33$  and 6.16 mm.

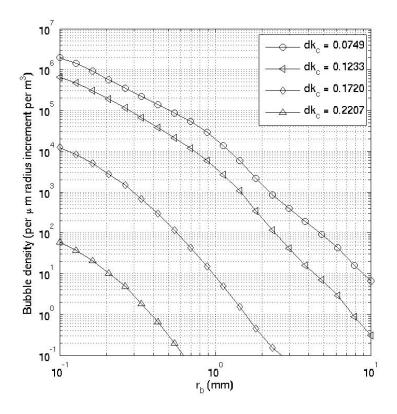


Figure 12: Bubble size spectrum at depth  $dk_c = 0.0749, 0.1233, 0.1720$ , and 0.2207 from the wave surface, at  $(x - x_b)/\lambda_c = 0.5530$  and  $(t - t_b)/T = 0.62$