

## Synoptic Meteorology I: Skew-T Diagrams and Thermodynamic Properties

### For Further Reading

Most information contained within these lecture notes is drawn from Chapters 1, 2, 4, and 6 of “The Use of the Skew T, Log P Diagram in Analysis and Forecasting” by the Air Force Weather Agency, a PDF copy of which is available from the course website. Chapter 5 of *Weather Analysis* by D. Djurić provides further details about the utility of skew-T/ln-p diagrams.

### A Refresher on Adiabatic Processes

An adiabatic process is one in which an air parcel *does not exchange energy with its surroundings* due to the temperature difference between the parcel and its surroundings. In other words, there is no diabatic heating being applied to the parcel from an external source. Adiabatic processes do not need to be isothermal, but the system’s internal energy (e.g., dry static energy  $s = c_p T + gz$ ) must be conserved following the motion. In other words, adiabatic processes are iso-energy processes.

For vertical motion in a subsaturated atmosphere, an air parcel’s volume is initially conserved. Let us consider how volume and, by extension, temperature changes upon ascent and descent:

If this parcel *ascends*, it moves from higher to lower pressure.

- This reduces the pressure exerted on the air parcel from its surroundings.
- The greater pressure inside the air parcel forces the parcel to increase in volume.
- This requires that work be done, which itself is associated with the conversion of kinetic to potential energy as the parcel ascends.
- Since temperature is a measure of a parcel’s kinetic energy, a reduction in kinetic energy leads to a reduction in temperature.
- The rate at which temperature decreases is equal to the dry adiabatic lapse rate.

If this parcel *descends*, it moves from lower to higher pressure. The sequence above holds, except in the opposite sense, for this air parcel. Its volume decreases and kinetic energy and temperature increase as the parcel descends.

What if the air parcel is saturated? The same basic tenets hold, except latent heat release associated with phase changes of water substance must be accounted for in the temperature change evaluation when an air parcel ascends. The rate at which a saturated ascending parcel’s temperature decreases with height is equal to the moist adiabatic or pseudoadiabatic lapse rate, the precise value of which is pressure- and temperature-dependent. The moist static energy ( $= c_p T + gz + L_v q$ ) rather than the dry static energy is conserved for such ascent. Note, however, that a descending air parcel becomes subsaturated almost immediately, so that saturation does not impact the physics described above.

### Thermodynamic Diagrams and the Skew-T Chart

Thermodynamic diagrams serve three general purposes. First, they provide a means to plot and analyze observations – namely temperature, dew point temperature (or mixing ratio), and wind – taken above the Earth’s surface at a given location. Second, thermodynamic diagrams enable us to graphically assess or compute stability-related fields that are functions of the observed variables. Finally, thermodynamic diagrams allow for atmospheric processes such as lifting an air parcel to be simulated, enabling an estimate of the effects of such processes on an air parcel’s properties.

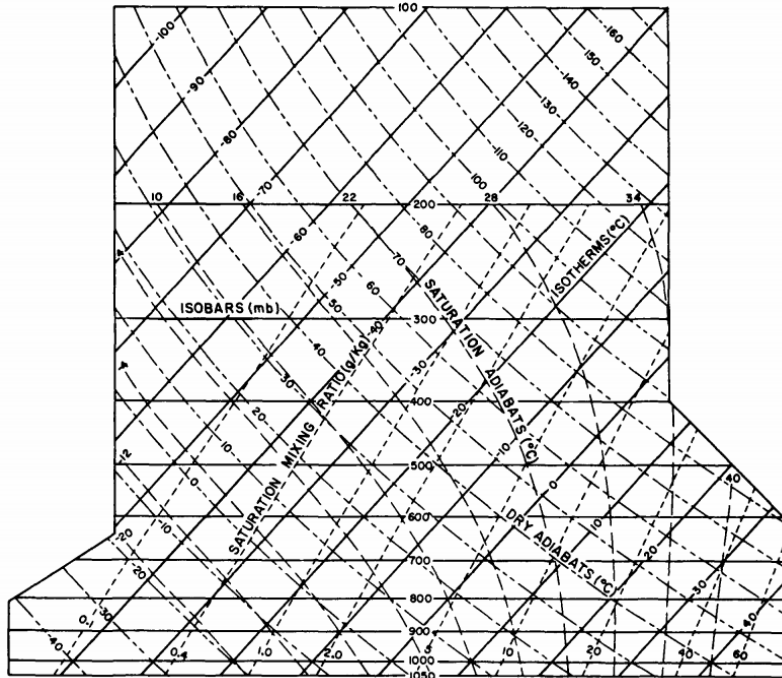
On any thermodynamic diagram, we require that there be at least five different kinds of isolines:

- **Isotherms.** Isotherms allow for observations of temperature and dew point temperature to be plotted. They allow for the assessment of derived fields such as potential temperature, equivalent potential temperature, and convective temperature.
- **Isobars.** Isobars allow for the vertical variability in temperature, dew point temperature, and wind speed and direction to be plotted. They serve as reference lines by which lifting levels and atmospheric layers may be identified.
- **Dry adiabats.** A parcel undergoing an adiabatic process follows a dry adiabat as it ascends or descends. A dry adiabat is a measure of the rate at which temperature changes ( $9.8^{\circ}\text{C}$  per 1 km) upon ascent or descent.
- **Mixing ratio lines.** Mixing ratio is a measure of the mass of water vapor per kilogram of dry air. For a subsaturated air parcel, mixing ratio is conserved (does not change) as an air parcel changes altitude. Mixing ratio lines provide a means of representing this property.
- **Saturated, moist, or pseudoadiabats.** The primary atmospheric exchange of heat between an air parcel and its surroundings (a *diabatic process*) is that associated with phase changes of water. For example, as an air parcel ascends, condensation or freezing may result, and the associated change of phase transfers heat from water to air. A saturated, moist, or pseudoadiabat is a measure of the rate at which temperature cools upon ascent after an air parcel has become saturated. Note that the precise value depends on moisture content, with higher moisture content associated with a smaller amount of cooling, and assumptions made regarding what happens to water in the air parcel immediately upon condensation, deposition, or freezing. In the lower- to midtroposphere, saturated adiabats reflect a cooling rate of  $\sim 6\text{-}7^{\circ}\text{C}$  per 1 km of ascent.

There exist many different types of thermodynamic diagrams, though a discussion of the benefits and drawbacks of each is beyond the scope of this class. Instead, we focus on one specific type of thermodynamic diagram: the *skew-T/ln-p*, or skew-*T* for short, and its benefits and applications.

The skew- $T$  diagram takes its name from the isotherm and isobar orientations on the diagram. The isotherms on a skew- $T$  diagram are skewed  $45^\circ$  from the vertical. The natural logarithm of pressure ( $\ln p$ ) is the vertical coordinate. Isobars are given by horizontal lines.

A sample skew- $T$  diagram is provided in Fig. 1. Temperature and dew point temperature observations are plotted on the chart upward from the bottom, with the former (latter) typically given by a solid red (green) line. Wind speed and direction on a skew- $T$  diagram are typically not plotted directly on the chart but, rather, to one side of the chart (typically the right side).



**Figure 1.** A sample skew- $T/\ln-p$  diagram. Isotherms are depicted by the solid skewed lines, isobars by the solid horizontal lines, mixing ratio lines by the short-dashed lines, dry adiabats by the alternating short-long dashed lines, and saturated adiabats by the long-dashed lines. Reproduced from “The Use of the Skew T, Log P Diagram in Analysis and Forecasting”, their Fig. 1d.

There are five desired qualities about thermodynamic diagrams that a skew- $T$  diagram meets:

- The important isopleths are straight, or nearly so, rather than curved. See Fig. 1 above.
- A large angle between isotherms and adiabats. This helps facilitate estimating atmospheric stability, as we will illustrate in our next lecture.
- There exists a direct proportionality between thermodynamic energy and the area between two lines on the chart, and this proportionality is approximately equal over the entire chart. We will also illustrate this in our next lecture.

- Vertical profiles of temperature, dew point temperature, and wind speed and direction can be plotted through the entirety of the troposphere and through the tropopause.
- The vertical coordinate of the diagram,  $\ln p$ , approximates the vertical coordinate of the atmosphere. Indeed,  $\ln p$  is approximately inversely proportional to  $-z$ .

## Definitions

Before we can describe how we can use a skew- $T$  diagram to draw atmospheric inferences, whether at a given location or over some larger region, we must first consider a bit of terminology that will help us along the way.

### Lapse Rate

Lapse rate refers to the change in some quantity with respect to height. Thus, the temperature lapse rate, or  $\Gamma$ , refers to the change of temperature with height,

$$\Gamma = -\frac{\partial T}{\partial z} \quad (1)$$

Temperature typically decreases with increasing height above sea level. Given the leading negative sign on the right-hand side of (1), this situation is characterized by  $\Gamma > 0$ . In the case where temperature increases with increasing height above sea level,  $\Gamma < 0$ .

The slopes of the dry and saturated adiabats on a skew- $T$  diagram are given by the dry adiabatic lapse rate ( $\Gamma_d$ ) and saturated adiabatic lapse rate ( $\Gamma_s$ ), respectively. The dry adiabatic lapse rate  $\Gamma_d$  is equal to  $g/c_p = 9.81 \text{ m s}^{-2}/1005.7 \text{ J kg}^{-1} \text{ K}^{-1} = 9.75 \text{ K km}^{-1}$ . Given that 1 K and 1°C changes in temperature are equivalent,  $\Gamma_d$  is often expressed as  $9.75^\circ\text{C km}^{-1}$  or, rounding to two significant figures,  $9.8^\circ\text{C km}^{-1}$ . The saturated adiabatic lapse rate  $\Gamma_s$  is smaller than the dry adiabatic lapse rate given the release of latent heat to the environment as water vapor condenses or deposits, or as liquid water freezes. The precise value of  $\Gamma_s$  varies as a function of water vapor content, pressure, and air temperature but is typically between  $6\text{-}7^\circ\text{C km}^{-1}$  in the lower-to-middle troposphere. Note how the slope of the saturated adiabats in Fig. 1 is steeper than that of the dry adiabats.

If one lifts an air parcel along an adiabat, the dry adiabatic lapse rate describes how that air parcel's temperature will change upon ascent *so long as it remains subsaturated*. It also describes how any air parcel's temperature will change upon descent. The saturated adiabatic lapse rate describes how that air parcel's temperature will change upon ascent after it has become saturated.

In most situations, the temperature (or environmental) lapse rate is smaller than or equal to the dry adiabatic lapse rate. When the environmental lapse rate is larger than the dry adiabatic lapse rate, the environmental lapse rate is said to be *superadiabatic* (i.e., larger than adiabatic). This is not

common; it typically occurs only at and immediately above a strongly heated surface, particularly during the warm-season (i.e., late spring through early fall).

### *Layers in the Atmosphere*

There exist multiple types of layers bounded by two isobaric surfaces that can be identified using a skew- $T$  diagram. For example, an *adiabatic layer* is one in which  $\Gamma \approx \Gamma_d$ . This is usually caused by turbulent vertical mixing and is most commonly found within the planetary boundary layer (i.e., near the surface). Turbulent vertical mixing homogenizes, or makes uniform, certain variables over the vertical layer in which the mixing occurs – namely potential temperature and mixing ratio. It is most commonly found near the surface since strong sensible heating (leading to dry convection and localized thermals) and vertical wind shear are the primary drivers of the requisite turbulence.

An *isothermal layer* is one in which  $\Gamma = 0$ , or one where the observed temperature is constant with increasing height above sea level. In such a case, the observed temperature trace is parallel to the isotherms. Similarly, an *inversion layer* is a layer in which the environmental lapse rate is negative ( $\Gamma < 0$ ), describing a situation where the observed temperature increases with increasing height.

### **Derived Thermodynamic Variables and the Skew- $T$ Diagram**

Plotted on a skew- $T$  diagram are four elements: temperature, dew point temperature, wind speed, and wind direction. We are particularly interested in the first two of these elements. However, we can readily compute many other thermodynamic variables given these two fields and a properly constructed skew- $T$  diagram. In this discussion, we wish to identify many of the most commonly used derived thermodynamic variables, describe how they can be obtained from a skew- $T$  diagram, and, in many instances, state why we are interested in that given field. Note that there are many more derived thermodynamic variables that may be computed using a skew- $T$  diagram; please refer to Chapter 4 of “The Use of the Skew T, Log P Diagram in Analysis and Forecasting” for details on these variables and their computation.

#### *Mixing Ratio and Saturation Mixing Ratio*

Mixing ratio ( $w$ ) is defined as the ratio of the mass of water vapor contained within a given sample of air to the mass of dry air contained within that sample of air. It is one of several measures by which moisture content in the air may be quantified. To find the mixing ratio at a desired isobaric level, read or interpolate the value of the mixing ratio line that intersects the observed dew point temperature curve at the isobaric level of interest.

Saturation mixing ratio ( $w_s$ ) is defined as the ratio of the mass of water vapor contained within a given sample of air *if it is saturated* to the mass of dry air contained within that sample of air. The units of mixing ratio and saturation mixing ratio are  $\text{g kg}^{-1}$ . The same procedure used to find mixing

ratio is used to find saturation mixing ratio, except substituting the observed temperature curve for the observed dew point temperature curve.

Consider Fig. 10 on page 4-2 of “The Use of the Skew T, Log P Diagram in Analysis and Forecasting.” In this example, at 800 hPa, the mixing ratio is  $\sim 3.4 \text{ g kg}^{-1}$  and the saturation mixing ratio is  $6 \text{ g kg}^{-1}$ .

### *Relative Humidity*

Relative humidity (RH) is defined as the ratio, in percent, of the mixing ratio to the saturation mixing ratio. In other words, it represents the fraction of water vapor present in the air compared to the water vapor that would be present if the air were saturated. To find the relative humidity, first find the mixing ratio and saturation mixing ratio as described above, then divide  $w$  by  $w_s$  and multiply the result by 100.

### *Vapor Pressure and Saturation Vapor Pressure*

Vapor pressure ( $e$ ) is defined as the portion of the total atmospheric pressure (at a given isobaric level) that is contributed by water vapor molecules. It, like mixing ratio, is one of several measures by which moisture content in the air may be quantified. To find vapor pressure, first identify the dew point temperature at the desired isobaric level. Next, follow an isotherm up (or down) from this observation until you reach 622 hPa. Read the value of the mixing ratio line that intersects this isotherm at 622 hPa. This value, in hPa, is your vapor pressure.

Saturation vapor pressure ( $e_s$ ) is defined as the portion of the total atmospheric pressure (at a given isobaric level) that would be contributed by water vapor molecules if the air sample were saturated. The procedure to find the saturation vapor pressure is identical to that for vapor pressure, except starting by identifying the temperature at the desired isobaric level. This process is illustrated in Fig. 12 on page 4-4 of “The Use of the Skew T, Log P Diagram in Analysis and Forecasting.”

Why do we read up to 622 hPa to determine vapor pressure and saturation vapor pressure? We can quantify this utilizing the relationship between mixing ratio and vapor pressure (or, equivalently, saturation mixing ratio and saturation vapor pressure). To wit, the saturation mixing ratio is related to the saturation vapor pressure by the following relationship:

$$w_s = 0.622 \frac{e_s}{p - e_s} \quad (2)$$

In (2),  $w_s$  has units of  $\text{g g}^{-1}$  rather than  $\text{g kg}^{-1}$ . By dividing the left-hand side by 1000, or equivalently multiplying the right-hand side by 1000, we can get  $w_s$  in units of  $\text{g kg}^{-1}$ . Further, typically  $p$  is much larger than  $e_s$  such that  $p - e_s \approx p$ . Substituting, (2) becomes:

$$w_s = 622 \frac{e_s}{p} \quad (3)$$

When we described how to obtain the saturation vapor pressure, we stated to read up (or down) an isotherm until reaching 622 hPa. This means that saturation vapor pressure *does not* change as pressure changes but that it *does* change as temperature changes (i.e., reading up or down a different isotherm). Under this condition, we can plug in any value for  $p$  – say, 622 hPa. If we do so, then we find that  $w_s = e_s$ , noting the different units ( $w_s$  in  $\text{g kg}^{-1}$ ,  $e_s$  in hPa), the conversions for which we have neglected to include with our 0.622 value.

### *Potential Temperature*

Potential temperature ( $\theta$ ) is the temperature that a sample of air would have if it were brought dry adiabatically (i.e., without heat exchange between an air parcel and its surroundings) to 1000 hPa. To determine potential temperature, identify the temperature at a desired isobaric level and proceed (typically, downward) along the dry adiabat that intersects the temperature curve at that isobaric level until reaching 1000 hPa. The isotherm at 1000 hPa defines the potential temperature. This process is illustrated in Fig. 13 on page 4-6 of “The Use of the Skew T, Log P Diagram in Analysis and Forecasting.” The units of potential temperature are K.

When diabatic heating is not occurring, potential temperature is conserved following the motion in both the horizontal and vertical directions. This attribute is incredibly beneficial for identifying synoptic-scale areas of ascent and descent, as we will demonstrate next semester. Note that so long as the lapse rate is less than the dry adiabatic lapse rate ( $\Gamma < \Gamma_d$ ), potential temperature increases with increasing altitude above sea level. This implies that potential temperature can be used as an alternative vertical coordinate, the applications of which we will explore next semester.

### *Wet-Bulb Temperature*

The wet-bulb temperature ( $T_w$ ) is the lowest temperature to which a sample of air (at constant pressure) can be cooled by evaporating water into it. If the sample of air is saturated, such that no more water can be evaporated into it, then  $T_w = T$ . Otherwise, since evaporation requires heat input from the surrounding environment (the environment cools),  $T_w < T$ . However, since evaporation increases water vapor content in the air sample,  $T_w > T_d$ . Thus,  $T_d < T_w < T$ . The units of wet-bulb temperature are  $^{\circ}\text{C}$  or K.

To obtain the wet-bulb temperature, identify the dew point temperature at the desired isobaric level. Draw a line upward along the mixing ratio line that intersects this dew point temperature reading. Next, identify the temperature at the desired isobaric level. Draw a line upward along the dry adiabat that intersects this temperature reading until you intersect the line drawn upward along the mixing ratio line. Follow the saturated adiabat that intersects this intersection point down until you reach the isobaric level at which you started. The value of the isotherm at this level gives you

the wet-bulb temperature. This process is illustrated in Fig. 14 on page 4-7 of “The Use of the Skew T, Log P Diagram in Analysis and Forecasting.”

### *Equivalent Temperature and Equivalent Potential Temperature*

Equivalent temperature ( $T_e$ ) is the temperature that a sample of air would have if all of its moisture were condensed out by a pseudoadiabatic process (one in which all liquid or solid water substance falls out of an air parcel immediately after it forms) and then is brought back dry adiabatically to its original pressure. Equivalent potential temperature ( $\theta_e$ ) is identical to  $T_e$ , except the sample of air is brought dry adiabatically to 1000 hPa rather than its original pressure. The units of both equivalent temperature and equivalent potential temperature are K.

To obtain  $T_e$ , identify the dew point temperature at the desired isobaric level. Draw a line upward along the mixing ratio line that intersects this dew point temperature reading. Next, identify the temperature at the desired isobaric level. Draw a line upward along the dry adiabat that intersects this temperature until you intersect the line drawn upward along the mixing ratio line. From here, follow the saturated adiabat that intersects this intersection point upward until you reach the isobaric level at which the dry and saturated adiabats become parallel to each other (i.e.,  $\Gamma_d \approx \Gamma_s$ ). Finally, follow the dry adiabat that intersects this point downward until you reach the isobaric level at which you started. The value of the isotherm at this level gives you the equivalent temperature. You obtain the equivalent potential temperature if you continue downward to 1000 hPa and read the value of the intersecting isotherm. This process is illustrated in Fig. 15 on page 4-8 of “The Use of the Skew T, Log P Diagram in Analysis and Forecasting.”

Note that  $T_e > T$  and, by extension,  $\theta_e > \theta$ . Why would we expect this to be the case? Both  $T_e$  and  $\theta_e$  involve condensing moisture out of an air parcel. Condensation, freezing, and deposition cause the water substance to lose heat to its surrounding environment. This release of latent heat to the environment causes the environment to warm, thus resulting in  $T_e > T$  and, by extension,  $\theta_e > \theta$ . Only in the case where there is no water vapor in the atmosphere ( $w = 0$ ) does  $T_e = T$  and  $\theta_e = \theta$ .

Equivalent potential temperature is approximately conserved following the motion, whether or not an air parcel is initially saturated, under pseudoadiabatic conditions. It provides a measure of both the potential warmth and moistness of a sample of air. While we will not make extensive use of equivalent potential temperature in this class it is beneficial to keep this dialogue in mind for other situations in which it might be beneficial.

### *Virtual Temperature*

Virtual temperature ( $T_v$ ) is the temperature at which dry air ( $w = 0$ ) would have the same density as a observed air sample ( $w > 0$ ). Virtual temperature is related to temperature  $T$  and mixing ratio  $w$  by the following equation:



$$T_v = T(1 + 0.6w) \quad (4)$$

Note that  $w$  here is expressed in  $\text{g g}^{-1}$  and not  $\text{g kg}^{-1}$ . The units of virtual temperature are K.

For the case where  $w = 0$  (no water vapor present in the atmosphere),  $T_v = T$ . Otherwise, where  $w > 0$ ,  $T_v > T$ . Why? Warm air and moist air are both less dense than cool air and dry air. Since our observed air sample contains some water vapor, it is less dense than dry air independent of the temperature. For the two to have equal density, the temperature of the dry air must be warmer such that it is less dense than the cool air. In other words, the effects of warm vs. cool and moist vs. dry cancel each other out. Thus, the virtual temperature is always greater than the temperature, albeit by a small amount (1-5 K) given typically observed values of  $w$  ( $w < 0.04 \text{ g g}^{-1}$ ). The temperature and virtual temperature are nearly identical at and above 500 hPa, where  $w$  is typically  $\sim 0 \text{ g g}^{-1}$ .

### *Precipitable Water*

Precipitable water (PW) is the total water vapor contained within a vertical column of air (over a unit area of  $1 \text{ m}^2$ ) bounded by two isobaric levels. The total precipitable water (TPW) is the special case where the two isobaric levels are the ground and the top of the atmosphere, or the tropopause. The units of precipitable water and total precipitable water are  $\text{kg m}^{-2}$  or, in equivalent notation, mm. Mathematically, total precipitable water can be expressed as:

$$TPW = \frac{1}{g} \int_{p_{sfc}}^{p_{trop}} w dp \quad (5)$$

In (5),  $w$  is the mixing ratio,  $g$  is the gravitational constant ( $9.81 \text{ m s}^{-2}$ ),  $p_{trop}$  is the pressure at the tropopause, and  $p_{sfc}$  is the pressure at the surface. High values of TPW are associated with greater moisture content and, in precipitating regions, greater liquid equivalent values of accumulated precipitation. However, because it represents only an instantaneous vertical snapshot (and thus does not consider processes such as advection that can influence local moisture amounts), the TPW does not provide a minimum or maximum bound on the amount of precipitation that can or does fall during a precipitation event.